# BOUNDARY CONTROL OF CHEMICAL TUBULAR REACTORS<sup>2</sup>

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Abstract: In this paper a globally stabilizing boundary feedback control law for an arbitrarily fine discretization of a nonlinear PDE model of a chemical tubular reactor is presented. The objective is to stabilize an unstable steady–state of the system using boundary control of temperature and concentration on the inlet side of the reactor. We discretize the original nonlinear PDE model in space using finite difference approximation and get a high order system of coupled nonlinear ODEs. Then, using backstepping design for parabolic PDEs we transform the original coupled system into two uncoupled target systems that are asymptotically stable in  $l^2$ –norm with appropriate homogeneous boundary conditions. In the real system the designed control laws would be implemented through small variations of the prescribed inlet temperature and prescribed inlet concentration.

Keywords: Global Stabilization, Backstepping, Feedback Boundary Control

# 1. INTRODUCTION

A feedback boundary control law that globally stabilizes an unstable steady state is designed for a chemical tubular reactor. The control is applied through variations of the inlet temperature and inlet concentration.

Due to the numerous industrial applications for chemical tubular reactors, the problem of monitoring and controlling them effectively is of great safety and economical importance. It has been shown numerically, analytically, and experimentally (see (Hlaváček and Hofmann, 1970a) and references therein) that in some cases the parabolic PDEs governing the temperature and concentration inside the tubular reactor can have more than one steady state solution. The multiple steady states can be either stable or unstable. The standard approach, once it was realized that there could be more than one steady state, was to find a priori estimates of the conditions under which there would be uniqueness or multiplicity. The obtained estimates would then be used to design the equipment such that undesired phenomena would be eliminated and the equipment operated rationally.

An alternative way to suppress the undesired behavior in chemical tubular reactors is through active control. Although the majority of the work on the control of chemical reactors was done for lumped parameter nonisothermal reactors (see (Ray, 1981) and references therein), significant research efforts have focused on the analysis of the properties of PDE models for chemical tubular reactors (see (Varma and Aris, 1977) for a survey), and more recently the analysis of existence and uniqueness of the state trajectories (Laabissi et al., 2001). A large research activity has been also dedicated to the control designs based on PDE models of tubular reactors. Using the fact that for parabolic PDE systems the eigenspectrum of the spatial differential operator can be partitioned into a finite-dimensional slow one and an infinitedimensional stable fast one, Christofides (2001) used a combination of Galerkin's method with a procedure for the construction of approximate inertial manifolds for the derivation of ODE systems of dimension equal to the number of slow modes. The ODE systems obtained in this fashion yield solutions which are close, up to a desired accuracy, to the ones of the PDE system, for almost all times. These ODE systems were then used as the basis for the synthesis of output feedback controllers for nonisothermal tubular reactors

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that guarantee stability and enforce the output of the closed–loop system to follow, up to a desired accuracy, a prescribed response for almost all times. The distributed control was applied using the jacket temperature as the manipulated input. In a more recent paper by Orlov and Dochain (2001) a sliding mode control developed for minimum phase semilinear infinite–dimensional systems was applied to stabilization of chemical reactors (both plug flow and tubular). In that paper authors use distributed control to stabilize the system around prespecified temperature and concentration steady state profiles corresponding to a desired coolant temperature.

In this paper we use the most general model of a chemical tubular reactor. The only assumptions made in the model are that the average velocity of reactant flow is constant, the dispersive fluxes of mass and heat follow Fick's and Fourier's laws with constant mass and energy dispersion coefficients, the heat transfered from any element of the jacket surrounding the reactor is proportional to the difference of the local temperature and the constant jacket temperature, and that the reaction rate at any point inside the domain is a non-linear function of the temperature and concentration at that same point.

Our objective is to stabilize an unstable steady state using boundary control of temperature and concentration on the inlet side of the tubular reactor. To achieve that we first discretize the original PDE model in space using finite difference method which gives a high order system of coupled nonlinear ODEs for temperature and concentration. Then, using backstepping design (Krstić et al., 1995), we obtain a discretized coordinate transformation that transforms the original coupled system into a target system consisting of two uncoupled systems that are asymptotically stable in  $l^2$ -norm with the same type of boundary conditions as the original system. The fact that the discretized coordinate transformation is invertible, for an arbitrary (finite) grid choice, implies global asymptotic stability of the discretized version of the original system. The coordinate transformation is then used to obtain nonlinear feedback boundary control laws for temperature and concentration in the original set of coordinates.

#### 2. MATHEMATICAL MODEL

In this section we derive a mathematical model for the chemical tubular reactor. A model that assumes no radial velocity and concentration gradients in the reactor, the temperature gradient described by use of a proper value of the effective radial conductivity, absence of temperature and concentration gradients within and outside of a catalyst particle, the properties of the reaction mixture characterized by average values, the mechanism of axial mixing described by a single parameter in Fick's or Fourier's law, and the kinetics of the first order is considered. The mass and energy balance equations for the tubular chemical reactor are in that case given by (Varma and Aris, 1977)

$$T_t(z,t) = \frac{\lambda_{ea}}{\rho C_p} T_{zz}(z,t) - V T_z(z,t) - \frac{\Delta H}{\rho C_p} k_0$$
$$\times C(z,t) e^{-\frac{E}{RT}} - \frac{4h}{d\rho C_p} \left(T(z,t) - T_w\right) , \quad (1)$$
$$C_t(z,t) = D_{ma} C_{zz}(z,t) - V C_z(z,t)$$
$$-k_0 C(z,t) e^{-\frac{E}{RT}} , \quad (2)$$

with boundary conditions

$$\frac{\lambda_{ea}}{\rho C_p} T_z(0,t) = V \times \left(T(0,t) - T_{in}\right),\tag{3}$$

$$D_{ma}C_{z}(0,t) = V(C(0,t) - C_{in}), \qquad (4)$$

$$\frac{\kappa_{ea}}{\rho C_p} T_z(L,t) = 0, \tag{5}$$

$$D_{ma}C_z(L,t) = 0. (6)$$

In the above system T > 0 denotes temperature, C > 0 stands for concentration, V is the linear velocity,  $\lambda_{ea}$  is the axial energy dispersion coefficient,  $D_{ma}$  is the axial mass dispersion coefficient,  $\Delta H$  is the reaction heat,  $\rho$  is the fluid density,  $C_p$  is the specific heat,  $k_0$  is the kinetic constant, E is the activation energy, R is the gas constant, h is the wall heat transfer coefficient, d is the reactor diameter,  $T_w > 0$  is the coolant temperature,  $T_{in} > 0$  is the inlet temperature,  $C_{in} > 0$  is the inlet reactant concentration, and subscripts denote partial derivatives with respect to the corresponding variables.

Introducing nondimensional spatial variable, time, and nondimensional temperature, coolant temperature, and concentration respectively as  $z' = \frac{z}{L}$ ,  $t' = \frac{V}{L}t$ ,  $\theta(z',t') = \frac{E}{RT_{in}} \frac{T(z',t') - T_{in}}{T_{in}}$ ,  $\theta_c(z',t') = \frac{E}{RT_{in}} \frac{T_w(z',t') - T_{in}}{T_{in}}$ ,  $y(z',t') = \frac{C_{in} - C(z',t')}{C_{in}}$ , and omitting superscripts ' for convenience, we obtain a nondimensionalized system

$$\theta_{t}(z,t) = \frac{1}{Pe_{\theta}} \theta_{zz}(z,t) - \theta_{z}(z,t) + BD_{a} (1 - y(z,t)) e^{\frac{\theta}{1 + \varepsilon \theta}} - \delta(\theta(z,t) - \theta_{c})$$
(7)  
$$y_{t}(z,t) = \frac{1}{Pe_{y}} y_{zz}(z,t) - y_{z}(z,t) + D_{a} (1 - y(z,t)) e^{\frac{\theta}{1 + \varepsilon \theta}},$$
(8)

with boundary conditions  $\theta_z(0,t) = Pe_{\theta}\theta(0,t)$ ,  $y_z(0,t) = Pe_yy(0,t)$ ,  $\theta_z(1,t) = 0$ ,  $y_z(1,t) = 0$ , where  $Pe_{\theta} = \frac{PC_pLV}{\lambda_{ea}}$ ,  $Pe_y = \frac{LV}{D_{ma}}$ ,  $\delta = \frac{4h}{d} \frac{L}{\rho C_p V}$ ,  $D_a = \frac{k_0 L}{V} e^{-\frac{E}{RT_{in}}}$ ,  $B = \frac{(-\Delta H)C_{in}}{\rho C_p T_{in}} \frac{E}{RT_{in}}$ , and  $\varepsilon = \frac{RT_{in}}{E}$  respectively stand for the Peclet number for heat transfer, Peclet number for mass transfer, dimensionless heat transfer parameter, Damköhler number, dimensionless adiabatic temperature rise, and dimensionless activation energy. For  $\theta_c$  we may assume, without loss of generality,  $\theta_c = 0$  (Hlaváček and Hofmann, 1970*a*). Depending on the values of the nondimensional Peclet numbers, Damköhler number, the dimensionless activation energy, the system (7)–(8) can have multiple equilibria that

can be either stable or unstable. For a particular case with  $Pe_{\theta} = Pe_y = 6$ , B = 10,  $\varepsilon = 0.05$ ,  $D_a = 0.05$ , and  $\delta = 0$  (adiabatic case) the above system has three equilibrium profiles (Hlaváček and Hofmann, 1970*b*). The temperature steady state profiles  $\overline{\Theta}(z)$  for that case are shown in Figure 1. It can be shown for the case of



Fig. 1. Steady state temperature profiles for the adiabatic chemical tubular reactor with  $Pe_{\theta} = Pe_y=6$ , Da=0.05,  $\varepsilon=0.05$ , and B=10.

an adiabatic tubular reactor with equal Peclet numbers (Varma and Aris, 1977) that if the system parameters are such that multiple steady states exist, then they are alternatively asymptotically stable and unstable, in the pattern s–u–s...s–u–s (s = asymptotically stable, u = unstable). The middle of the three steady state profiles in Figure 1 is therefore plotted with a dashed line to indicate that the steady state is unstable, while the outer two profiles plotted with solid lines are asymptotically stable.

Although not obvious from the equations (7)–(8), it is physically justifiable to apply feedback boundary control at z = 0 only. Since our backstepping control assumes that the control is applied at the 1–end, we introduce new spatial, temperature and concentration variables respectively as x = 1 - z,  $u(x,t) = \theta(1 - z,t) - \overline{\theta}(1-z)$  and  $v(x,t) = y(1-z,t) - \overline{y}(1-z)$ , where  $\overline{\theta}$  and  $\overline{y}$  are the steady state solutions of the system (7)–(8) with the corresponding boundary conditions. We then have  $\overline{u}(x) = \overline{\theta}(1-z)$  and  $\overline{v}(x) = \overline{y}(1-z)$ , and the system (7)–(8) becomes

$$u_{t}(x,t) = \frac{1}{Pe_{\theta}} u_{xx}(x,t) + u_{x}(x,t) + BD_{a} g(u,v,\overline{u},\overline{v}) -\delta u(x,t) \qquad (9)$$
$$v_{t}(x,t) = \frac{1}{Pe_{y}} v_{xx}(x,t) + v_{x}(x,t) + D_{a}g(u,v,\overline{u},\overline{v}) , \qquad (10)$$

where g is defined as

$$g(u,v,\overline{u},\overline{v}) = (1-v-\overline{v}) e^{\frac{u+\overline{u}}{1+\varepsilon(u+\overline{u})}} - (1-\overline{v}) e^{\frac{\overline{u}}{1+\varepsilon\overline{u}}}, \quad (11)$$

with boundary conditions

$$u_x(0,t) = 0$$
, (12)

$$v_x(0,t) = 0$$
, (13)

$$u_x(1,t) = -Pe_{\theta}u(1,t) + \Delta u_x(1,t) , \qquad (14)$$

$$v_x(1,t) = -Pe_y v(1,t) + \Delta v_x(1,t) .$$
 (15)

Note that the equilibrium of the system is now shifted to (u, v) = (0, 0). For the case when  $\overline{\theta}$  and  $\overline{y}$  (equivalently  $\overline{u}$  and  $\overline{v}$ ) are those corresponding to the unstable steady state, the equilibrium at the origin of the system (9)–(15) is open loop ( $\Delta u_x(1,t) = \Delta v_x(1,t) = 0$ ) unstable. The objective is to stabilize u(x,t) and v(x,t) for that case by using  $\Delta u_x(1,t)$  and  $\Delta v_x(1,t)$  for control. In real application control would be implemented on the inlet side of the reactor through small variations of  $T_{in}$  and  $C_{in}$ . From a physical point of view this implies that the total temperature (concentration) control at the inlet side will consist of a prescribed component  $T_{in}$  $(C_{in})$  modulated by a control signal  $\Delta T_{in}$  ( $\Delta C_{in}$ ).

## 3. CONTROL LAW

To discretize the problem, let us start by denoting  $h = \frac{1}{N}$ , where *N* is an integer. Then, with  $u_i$ ,  $v_i$ ,  $\overline{u}_i$ , and  $\overline{v}_i$  respectively defined as  $u_i(t) = u(ih,t)$ ,  $v_i(t) = v(ih,t)$ ,  $\overline{u}_i = \overline{u}(ih)$ , and  $\overline{v}_i = \overline{v}(ih)$ , i = 0, ..., N, we represent the nondimensional system (9) and (10) as

$$\begin{split} \dot{u}_{i} &= \frac{1}{Pe_{\theta}} \frac{u_{i+1} - 2u_{i} + u_{i-1}}{h^{2}} + \frac{u_{i+1} - u_{i}}{h} - \delta u_{i} \\ &+ BD_{a} g\left(u_{i}, v_{i}, \overline{u}_{i}, \overline{v}_{i}\right) \\ \dot{v}_{i} &= \frac{1}{Pe_{y}} \frac{v_{i+1} - 2v_{i} + v_{i-1}}{h^{2}} + \frac{v_{i+1} - v_{i}}{h} \\ &+ D_{a} g\left(u_{i}, v_{i}, \overline{u}_{i}, \overline{v}_{i}\right) \end{split}$$
(16)

with boundary conditions at x = 0 expressed as  $\frac{u_1 - u_0}{h} = 0$  and  $\frac{v_1 - v_0}{h} = 0$ . We now suggest a backstepping controller which transforms the original system into the discretization of the system

$$w_t(x,t) = \frac{1}{Pe_{\theta}} w_{xx}(x,t) + w_x(x,t) - C_1 w(x,t) , \quad (18)$$

$$s_t(x,t) = \frac{1}{Pe_y} s_{xx}(x,t) + s_x(x,t) - C_2 s(x,t) , \quad (19)$$

where  $C_1 > 0$  and  $C_2 > 0$ , with homogeneous Neumann boundary conditions at x = 0 given as  $w_x(0,t) =$  $s_x(0,t) = 0$ , and boundary conditions of a third kind at x = 1 given as  $w_x(1,t) = -Pe_{\theta}w(1,t)$  and  $s_x(1,t) =$  $-Pe_{v}s(1,t)$ , which is asymptotically stable in  $L^{2}$ norm. We should stress that the choice of the target system is one of the key issues. When transforming the original system we should try to keep its parabolic character, i.e., keep the second spatial derivative in the transformed coordinates. Even when applied for linear parabolic PDEs, the control laws obtained using standard backstepping would have gains that grow unbounded as  $n \to \infty$ . The problem with standard backstepping is that it would not only attempt to stabilize the equation, but also place all of its poles, and thus as  $n \rightarrow \infty$ , change its parabolic character. The coordinate transformation is sought in the form

$$w_i = u_i - \alpha_{i-1} \left( u_1, \dots, u_{i-1}, v_1, \dots, v_{i-1} \right)$$
(20)

$$s_i = v_i - \beta_{i-1} \left( u_1, \dots, u_{i-1}, v_1, \dots, v_{i-1} \right)$$
(21)

where  $w_i(t) = w(ih, t)$  and  $s_i(t) = s(ih, t)$ . The discretized form of equations (18) and (19) is

$$\dot{w_i} = \frac{1}{Pe_{\theta}} \frac{w_{i+1} - 2w_i + w_{i-1}}{h^2} + \frac{w_{i+1} - w_i}{h} - C_1 w_i \quad (22)$$

$$\dot{s}_{i} = \frac{1}{Pe_{y}} \frac{s_{i+1} - 2s_{i} + s_{i-1}}{h^{2}} + \frac{s_{i+1} - s_{i}}{h} - C_{2}s_{i} \qquad (23)$$

with  $\frac{w_1 - w_0}{h} = \frac{s_1 - s_0}{h} = 0$ ,  $\frac{w_N - w_{N-1}}{h} = -Pe_{\theta}w_N$ , and  $\frac{s_N - s_{N-1}}{h} = -Pe_y s_N$ .

By combining the above expressions, namely subtracting (22) from (16), expressing the obtained equation in terms of  $u_k - w_k$ , k = i - 1, i, i + 1, and applying (20) (analogously (23) from (17) for the concentration subsystem, and then using (21)) we obtain

$$\begin{split} &\alpha_{i} = \frac{1}{1 + Pe_{\theta}h} \Biggl\{ (2 + Pe_{\theta}h + C_{1}Pe_{\theta}h^{2}) \alpha_{i-1} - \alpha_{i-2} \\ &- (C_{1} - \delta)Pe_{\theta}h^{2}u_{i} - Pe_{\theta}h^{2}BD_{a} g(u_{i}, v_{i}, \overline{u}_{i}, \overline{v}_{i}) \\ &+ Pe_{\theta}h^{2} \frac{\partial \alpha_{i-1}}{\partial u_{1}} \Biggl[ \frac{1}{Pe_{\theta}} \frac{u_{2} - u_{1}}{h^{2}} + \frac{u_{2} - u_{1}}{h} - \delta u_{1} \\ &+ BD_{a} g(u_{1}, v_{1}, \overline{u}_{1}, \overline{v}_{1}) \Biggr] \\ &+ Pe_{\theta}h^{2} \sum_{k=2}^{i-1} \frac{\partial \alpha_{i-1}}{\partial u_{k}} \Biggl[ \frac{1}{Pe_{\theta}} \frac{u_{k+1} - 2u_{k} + u_{k-1}}{h^{2}} \\ &+ \frac{u_{k+1} - u_{k}}{h} - \delta u_{k} + BD_{a} g(u_{k}, v_{k}, \overline{u}_{k}, \overline{v}_{k}) \Biggr] \\ &+ Pe_{\theta}h^{2} \frac{\partial \alpha_{i-1}}{\partial v_{1}} \Biggl[ \frac{1}{Pe_{y}} \frac{v_{2} - v_{1}}{h^{2}} + \frac{v_{2} - v_{1}}{h} \\ &+ D_{a} g(u_{1}, v_{1}, \overline{u}_{1}, \overline{v}_{1}) \Biggr] \\ &+ Pe_{\theta}h^{2} \sum_{k=2}^{i-1} \frac{\partial \alpha_{i-1}}{\partial v_{k}} \Biggl[ \frac{1}{Pe_{y}} \frac{v_{k+1} - 2v_{k} + v_{k-1}}{h^{2}} \\ &+ \frac{v_{k+1} - v_{k}}{h} + D_{a} g(u_{k}, v_{k}, \overline{u}_{k}, \overline{v}_{k}) \Biggr] \Biggr\},$$
(24)  
$$\beta_{i} = \frac{1}{1 + Pe_{y}h} \Biggl\{ (2 + Pe_{y}h + C_{2}Pe_{y}h^{2})\beta_{i-1} - \beta_{i-2} \\ &- C_{2}Pe_{y}h^{2}v_{i} - Pe_{y}h^{2}D_{a} g(u_{i}, v_{i}, \overline{u}_{i}, \overline{v}_{i}) \\ &+ Pe_{y}h^{2} \frac{\partial \beta_{i-1}}{\partial u_{1}} \Biggl[ \frac{1}{Pe_{\theta}} \frac{u_{2} - u_{1}}{h^{2}} + \frac{u_{2} - u_{1}}{h} - \delta u_{1} \\ &+ BD_{a} g(u_{1}, v_{1}, \overline{u}_{1}, \overline{v}_{1}) \Biggr] \\ &+ Pe_{y}h^{2} \frac{\partial \beta_{i-1}}{\partial u_{k}} \Biggl[ \frac{1}{Pe_{\theta}} \frac{u_{k+1} - 2u_{k} + u_{k-1}}{h^{2}} \\ &+ \frac{u_{k+1} - u_{k}}{h} - \delta u_{k} + BD_{a} g(u_{k}, v_{k}, \overline{u}_{k}, \overline{v}_{k}) \Biggr] \Biggr\}$$

h

$$+D_{a} g(u_{1}, v_{1}, \overline{u}_{1}, \overline{v}_{1}) \bigg]$$

$$+Pe_{y}h^{2} \sum_{k=2}^{i-1} \frac{\partial \beta_{i-1}}{\partial v_{k}} \bigg[ \frac{1}{Pe_{y}} \frac{v_{k+1} - 2v_{k} + v_{k-1}}{h^{2}}$$

$$+ \frac{v_{k+1} - v_{k}}{h} + D_{a} g(u_{k}, v_{k}, \overline{u}_{k}, \overline{v}_{k}) \bigg] \bigg\}, \qquad (25)$$

starting with  $\alpha_0 = \beta_0 = 0$ . The controls are defined as

$$\Delta u_x(1,t) = Pe_{\theta}\alpha_{N-1} + \frac{\alpha_{N-1} - \alpha_{N-2}}{h} , \qquad (26)$$

$$\Delta v_x(1,t) = P e_y \beta_{N-1} + \frac{\beta_{N-1} - \beta_{N-2}}{h} .$$
 (27)

By inspection of the recursive control design algorithm one can verify that the coordinate transformation is invertible (which implies global asymptotic stability of the discretized system) and that the control law is smooth.

### 4. ASYMPTOTIC STABILITY OF THE DISCRETIZED SYSTEM IN MODIFIED **COORDINATES**

In this section we prove global asymptotic stability for (18) and (19) with  $w_x(0,t) = s_x(0,t) = 0$ ,  $w_x(1,t) = 0$  $-Pe_{\theta}w(1,t)$  and  $s_x(1,t) = -Pe_ys(1,t)$  in  $L^2$ -norm. To prove the stability for the w-system we start with a Lyapunov function  $V_1 = \frac{1}{2} \int_0^1 w(x,t)^2 dx$  and find its derivative with respect to time, along the trajectories of the system (18), to be

$$\dot{V}_{1} = \int_{0}^{1} ww_{t} dx = \int_{0}^{1} \left[ \frac{1}{Pe_{\theta}} w_{xx} + w_{x} - C_{1}w \right] wdx$$

$$= \frac{1}{Pe_{\theta}} \left[ ww_{x} \Big|_{0}^{1} - \int_{0}^{1} w_{x}^{2} dx \right] + \frac{w^{2}}{2} \Big|_{0}^{1} - C_{1} \int_{0}^{1} w^{2} dx$$

$$= -\frac{1}{2} \left( w^{2}(1) + w^{2}(0) \right) - \frac{1}{Pe_{\theta}} \int_{0}^{1} w_{x}^{2} dx - C_{1} \int_{0}^{1} w^{2} dx$$

$$\leq -2C_{1}V_{1}, \qquad (28)$$

which implies that the system (18) with  $w_x(0,t) = 0$ and  $w_x(1,t) = -Pe_{\theta}w(1,t)$  is asymptotically stable in and  $w_x(1,t) = -Pe_{\theta}w(1,t)$  is asymptotically stable in  $L^2$ -norm. The proof that (22) is asymptotically stable in  $l^2$ -norm with  $\frac{w_1-w_0}{h} = 0$  and  $\frac{w_N-w_{N-1}}{h} = -Pe_{\theta}w_N$  would be completely analogous. We would start with a Lyapunov function  $V_{1d} = \frac{1}{2} \sum_{i=0}^{N(t)} w_i^2$ , follow the ex-actly same procedure, and obtain  $V_{1d} \le -2C_1V_{1d}$ . The proof for the asymptotic stability of the s-system is completely analogous (the system equation and boundary conditions are of exactly the same form as for the *w*-system) and is therefore omitted.

#### 5. SIMULATION STUDY

In this section we present simulation results for a model of an adiabatic tubular reactor ( $\delta = 0$ ) from (Hlaváček and Hofmann, 1970b) with  $Pe_{\theta} = Pe_y = 6$ , Da=0.05,  $\varepsilon=0.05$ , and B=10. For this particular choice of parameters the system has three equilibria. The equilibrium profiles for temperature are shown in Figure 1. The equilibrium profiles for concentration have identical shape as the temperature ones, but with the amplitude scaled down with 1/B factor (see (Hlaváček and Hofmann, 1970*a*) for details), and are therefore not shown separately. As shown in (Hlaváček and Hofmann, 1970*b*), the middle profile is unstable while the outer two are stable. Our objective is to stabilize the unstable steady state using backstepping controller designed in Section 3.

As shown in Section 3, control laws for temperature (26) and concentration (27) are given in terms of  $\alpha_{N-1}$ ,  $\alpha_{N-2}$ , and  $\beta_{N-1}$ ,  $\beta_{N-2}$ , respectively, which can be easily obtained from the recursive expressions (24) and (25) by using symbolic tools available. Once the final expressions for temperature and concentration control are obtained, for some particular choice of *N*, one would have to use full state feedback to stabilize the system, i.e. the complete knowledge of temperature and concentration fields is necessary. Instead, we show that controllers of relatively low order (designed on a much coarser grid) can successfully stabilize the system for a variety of different simulation settings.

The idea of using controllers designed using only a small number of steps of backstepping, or equivalently using only a small number of state measurements, to stabilize the system for a certain range of the open–loop instability is based on the fact that in most real life systems only a finite number of open–loop eigenvalues is unstable. Indeed, the simulation studies for the heat convection loop (Bošković and Krstić, 2000*a*) and solid rocket propellant burning instability (Bošković and Krstić, 2000*b*) suggest that low order backstepping controllers are capable of detecting the occurrence of instability from a limited number of measurements, and therefore capable of successfully stabilizing the system for a variety of different simulation settings.

All simulations presented in this paper are run using BTCS finite difference method for N = 200 and the time step equal to 0.001 s. We start with a controller designed using only one step of backstepping, i.e. for  $N_c = 2$ , where the subscript "c" stands for controller. From now on we will use  $N_c$  to refer to a coarse grid discretization used in controller design, i.e. N and h appearing in expressions (24) and (25)will be replaced with  $N_c$  and  $h_c = \frac{1}{N_c}$  respectively, and N to refer to a fine grid used to simulate the behavior of the system described by equations (16) and (17). The initial distribution used for this simulation is  $u(x,0) = 0.02 \left( \overline{u}^{upp}(x) - \overline{u}^{mid}(x) \right)$  and v(x,0) =0.01  $(\overline{v}^{upp}(x) - \overline{v}^{mid}(x))$ , where superscripts *upp* and mid refer to the upper (stable) and the middle (unstable) steady state respectively. The motivation for using this type of initial profile, defined as a fraction of the difference between the two steady states, is motivated by a remark from (Varma and Aris, 1977). Applied to our system it says that for every perturbation that is in between the upper and the middle steady state profiles, the system goes to the upper one. Analogously, if we start between the middle and the lower one we will end up in the lower one. The system is initially perturbed for 2% and 1% of the difference between the upper and the middle temperature and concentration profiles, respectively. For the first 40 s we let the system evolve on its own and do not apply control. As the theory predicts, both temperature (see Figure 2) and concentration (not shown here) go to the upper stable equilibrium. After 40 s the system has settled into the new steady state, and that is when we apply control. As it can be seen from Figure 2, we successfully drive the temperature to the unstable temperature steady state in a couple of nondimensional seconds. The concentration response is qualitatively the same and is not shown here. The controller used in this particular setting was designed with  $C_1 = 3.5$ and  $C_2 = 5$  and uses only one temperature and one concentration measurement at x = 0.5.

The situation is slightly different if we start in between the middle and the lower steady state. As it can be seen from Figure 1, the distance between those two profiles is significantly bigger than the difference between the upper and the middle one, which will result in a more pronounced effect of the nonlinear terms. We can now stabilize the system only up to  $u(x,0) = 0.2 (\overline{u}^{low}(x) - \overline{u}^{mid}(x))$  and  $v(x,0) = 0.2 (\overline{v}^{low}(x) - \overline{v}^{mid}(x))$ , where the superscript low refers to the lower (stable) steady state, using controller designed for  $N_c = 2$ . For example, it took controller 8 s to stabilize the system if both initial distributions were 15% of the difference between the two equilibrium profiles. A further increase in the size of the initial distributions resulted in much higher values for control gains  $C_1$  and  $C_2$  necessary to stabilize the system. The system also underwent a long period of an oscillatory behavior.

We now proceed to deriving control laws for  $N_c = 3$ by introducing  $h_c = \frac{1}{N_c}$ . Starting with  $\alpha_0 = \beta_0 \equiv 0$  and using (24) and (25) we find expressions for  $\alpha_1$ ,  $\alpha_2$ ,  $\beta_1$ , and  $\beta_2$ , and use those to find control laws (26) and (27). The control signals depend on  $u_i(t) = u(ih_c, t)$ , and  $v_i(t) = v(ih_c, t)$  for i = 1, 2 only, which means that we use only two temperature measurements  $u_1$ at  $x = \frac{1}{3}$  and  $u_2$  at  $x = \frac{2}{3}$ , and corresponding two concentration measurements  $v_1$  at  $x = \frac{1}{3}$  and  $v_2$  at  $x = \frac{2}{3}$  to compute control laws.

As expected, by refining the grid in controller design from  $N_c = 2$  to  $N_c = 3$  we were able to extend the range of initial perturbations for which we can effectively stabilize the system. We are now able to stabilize the system when both initial distributions were 15% of the difference between the two equilibrium profiles in 2 s, as opposed to 8 s with the controller designed for  $N_c = 2$ . In fact, we can now stabilize the system even when the initial distribution is closer to the lower profile than it is to the middle one. The closed loop temperature response with initial perturbations in temperature and concentration  $u(x,0) = 0.5 (\overline{u}^{low}(x) - \overline{u}^{mid}(x))$  and  $v(x,0) = 0.55 (\overline{v}^{low}(x) - \overline{v}^{mid}(x))$ , and control gains equal to  $C_1 = 5$  and  $C_2 = 2.5$  is shown in Figure 3. As it can be seen, the controller successfully brings the system to the unstable equilibrium after a short period of oscillatory behavior. The concentration response is qualitatively the same and is not shown here. Note that the controller designed for  $N_c = 3$  was capable of stabilizing the system with even higher values of the initial perturbations, but it required much higher gains and system underwent a longer period of oscillatory behavior. We were not able though to recover the system from the lower steady state as we were able to in the case of the upper steady state.

In general, to accommodate the higher levels of initial disturbance one would have to increase the order of controller by applying recursive expressions (24) and (25) for higher  $N_c$ . Designing controllers for higher  $N_c$  would help the controller extract more information about the disturbance and stabilize the system more effectively using smaller control gains. A similar type of pattern was also encountered in the case of the thermal convection loop (Bošković and Krstić, 2000*a*). The simulation results in (Bošković and Krstić, 2000*a*) suggested that to accommodate the flows with higher Rayleigh number one had to increase the order of controller.



Fig. 2. Closed–loop temperature response with controller designed for  $N_c = 2$ ,  $C_1 = 3.5$ , and  $C_2 = 5$ . (First row: u(x,t); Second row: The control effort  $\Delta u_x(1,t)$ .)

### 6. CONCLUSIONS

A nonlinear feedback controller based on Lyapunov backstepping design that achieves global asymptotic stabilization of an unstable steady state for chemical tubular reactors has been derived. The result holds for any finite discretization in space of the original PDE model.

One key question presents a challenge for future research. It would be of interest to extend this result from the case of an arbitrary finite discretization of the model in space to the continuous model itself. This would, among other things, involve the proof that the proposed coordinate transformation remains bounded



Fig. 3. Closed–loop temperature response with controller designed for  $N_c = 3$ ,  $C_1 = 5$ , and  $C_2 = 2.5$ . (First row: u(x,t); Second row: The control effort  $\Delta u_x(1,t)$ .)

in the limit when the spatial grid becomes infinitely fine, i.e. when *N* tends to infinity. We intend to use the proof of boundedness for the linearized version of the tubular reactor model from Bošković *et al.* (2001) and try to extend the result to the original nonlinear model.

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