

POLYNOMIAL DESIGN OF CONTROLLERS FOR TWO – VARIABLE SYSTEMS – PRACTICAL IMPLEMENTATION

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Abstract: This paper presents the design and simulation of adaptive control for a two input – two output system together with the real – time control of a laboratory model using this designed method. The synthesis is based on a polynomial approach. Decoupling, where the compensator is placed ahead of the system, suppresses the interactions between control loops. The results of the simulation and the real-time experiments are also given. *Copyright © 2002 IFAC*

Keywords: Two input – two output system, controller synthesis, decoupling, compensators, adaptive control

1 INTRODUCTION

Many technological processes require that several variables relating to one system are controlled simultaneously. Each input may influence all system outputs. The design of a controller able to cope with such a system must be quite sophisticated. There are many different methods of controlling multivariable systems. Several of these use decentralized PID controllers (Luyben 1986), others apply single input-single-output (SISO) methods extended to cover multiple inputs (Chien *et al.* 1987). Here decoupling methods are used to transform the multivariable system into a series of independent SISO loops (Krishnawamy *et al.* 1991, Tade *et al.* 1986, Wittenmark *et al.* 1987, Skogestad and Postlethwaite 1996).

This paper is organized as follows: Section 2 presents the controlled model; Section 3 describes how feedback control without decoupling is designed; Section 4 describes two decoupling methods; Section 5 describes the system identification method; Section 6 gives the simulation results; Section 7 contains the experimental results; finally, Section 8 concludes the paper.

2 A DESCRIPTION OF A TWO INPUT – TWO OUTPUT SYSTEM

The internal structure of the the system is shown in Fig. 1

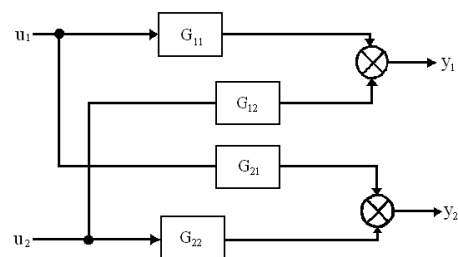


Fig. 1 A two input – two output system – the “P” structure

The transfer matrix of the system is

$$\mathbf{G} = \begin{bmatrix} G_{11} & G_{12} \\ G_{21} & G_{22} \end{bmatrix} \quad (1)$$

We can assume that the system is described by the matrix fraction

$$\mathbf{G}(z^{-1}) = \mathbf{A}^{-1}(z^{-1})\mathbf{B}(z^{-1}) = \mathbf{B}_1(z^{-1})\mathbf{A}_1^{-1}(z^{-1}) \quad (2)$$

Where polynomial matrices $A \in R_{22}[z^{-1}]$, $B \in R_{22}[z^{-1}]$ are the left indivisible decomposition of matrix $G(z^{-1})$ and matrices $A_1 \in R_{22}[z^{-1}]$, $B_1 \in R_{22}[z^{-1}]$ are the right indivisible decomposition.

The matrices of our discrete model are

$$A(z^{-1}) = \begin{bmatrix} 1 + a_1 z^{-1} + a_2 z^{-2} & a_3 z^{-1} + a_4 z^{-2} \\ a_5 z^{-1} + a_6 z^{-2} & 1 + a_7 z^{-1} + a_8 z^{-2} \end{bmatrix} \quad (3)$$

$$B(z^{-1}) = \begin{bmatrix} b_1 z^{-1} + b_2 z^{-2} & b_3 z^{-1} + b_4 z^{-2} \\ b_5 z^{-1} + b_6 z^{-2} & b_7 z^{-1} + b_8 z^{-2} \end{bmatrix}$$

and the differential equations of the model are

$$y_1(k) = -a_1 y_1(k-1) - a_2 y_1(k-2) - a_3 y_2(k-1) - a_4 y_2(k-2) + b_1 u_1(k-1) + b_2 u_1(k-2) + b_3 u_2(k-1) + b_4 u_2(k-2) \quad (4)$$

$$y_2(k) = -a_5 y_1(k-1) - a_6 y_1(k-2) - a_7 y_2(k-1) - a_8 y_2(k-2) + b_5 u_1(k-1) + b_6 u_1(k-2) + b_7 u_2(k-1) + b_8 u_2(k-2)$$

3 DESIGNING FEEDBACK CONTROL

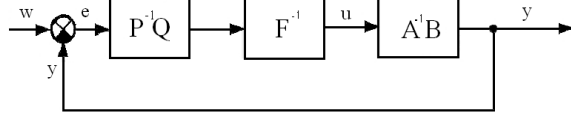


Fig. 2 Block diagram of the closed loop system

In the same way as the controlled system, the transfer matrix of the controller takes the form of matrix fraction

$$G_R(z^{-1}) = P^{-1}(z^{-1})Q(z^{-1}) = Q_1(z^{-1})P_1^{-1}(z^{-1}) \quad (5)$$

The matrix of an integrator for permanent zero control error is

$$F(z^{-1}) = \begin{bmatrix} 1 - z^{-1} & 0 \\ 0 & 1 - z^{-1} \end{bmatrix} \quad (6)$$

The control law apparent in the block diagram (operator z^{-1} will be omitted from some operations for the sake of simplification) has the form

$$U = F^{-1}Q_1 P_1^{-1}E \quad (7)$$

We can derive the following equation for the system output

$$Y = A^{-1}BF^{-1}P^{-1}QE = A^{-1}BF^{-1}P^{-1}Q(W - Y) \quad (8)$$

which can be modified to give

$$Y = P_1(AFP_1 + BQ_1)^{-1}BQ_1 P_1^{-1}W \quad (9)$$

The closed loop system is stable when the following diophantine equation is satisfied

$$AF P_1 + BQ_1 = M \quad (10)$$

where $M(z^{-1}) \in R_{22}[z^{-1}]$ is a stable diagonal polynomial matrix.

$$M(z^{-1}) = \begin{bmatrix} 1 + m_1 z^{-1} + m_2 z^{-2} + & 0 \\ + m_3 z^{-3} + m_4 z^{-4} & \\ 0 & 1 + m_5 z^{-1} + m_6 z^{-2} + \\ & + m_7 z^{-3} + m_8 z^{-4} \end{bmatrix} \quad (11)$$

The roots of this polynomial matrix are the ruling factor in the behaviour of the closed loop system. They must be inside the unit circle if the system is to be stable.

The product AFP_1 is dimensionally correct if the number of inputs is equal to the number of outputs.

The degree of the controller matrices polynomials depends on the internal properness of the closed loop. The structure of matrices P_1 and Q_1 was chosen so that the number of unknown controller parameters equals the number of algebraic equations resulting from the solution of the diophantine equation using the uncertain coefficients method.

$$P_1(z^{-1}) = \begin{bmatrix} 1 + p_1 z^{-1} & p_2 z^{-1} \\ p_3 z^{-1} & 1 + p_4 z^{-1} \end{bmatrix} \quad (12)$$

$$Q_1(z^{-1}) = \begin{bmatrix} q_1 + q_2 z^{-1} + q_3 z^{-2} & q_4 + q_5 z^{-1} + q_6 z^{-2} \\ q_7 + q_8 z^{-1} + q_9 z^{-2} & q_{10} + q_{11} z^{-1} + q_{12} z^{-2} \end{bmatrix}$$

The solution to the diophantine equation results in a set of sixteen algebraic equations with unknown controller parameters. We obtain the controller parameters by solving these equations.

4 DESIGNING DECOUPLING CONTROL USING COMPENSATORS

There are several ways to control multivariable systems with internal interactions. Some make use of decentralized PID controllers, whilst others are composed of a string of single input – single output methods.

One possibility is the serial insertion of a compensator ahead of the system (Krishnawamy *et al.* 1991, Peng 1990, Tade *et al.* 1986, Wittenmark *et al.* 1987). The aim here is to suppress of undesirable interactions between the input and output variables so that each input affects only one controlled variable.

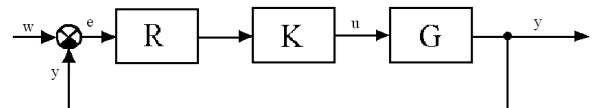


Fig. 3 A Closed loop system with compensator

The resulting transfer function H is then given by

$$H = KG \quad (13)$$

The decoupling conditions are fulfilled when matrix H is diagonal.

Several well – known compensators are given in (Krishnawamy *et al.* 1991, Peng 1990, Tade *et al.* 1986, Wittenmark *et al.* 1987). Control algorithms were derived for the model above with two compensators. These will be referred to as C_1 and C_2 .

Compensator C_1 is the inversion of the controlled system. Matrix H is, therefore, a unit matrix.

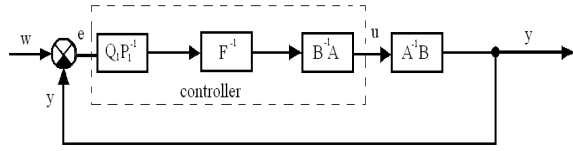


Fig. 4 The closed loop system with compensator C_1

This block diagram leads to an equation for the system output which takes the form

$$Y = P_1(FP_1 + Q_1)^{-1} Q_1 P_1^{-1} W \quad (14)$$

The following equation must be satisfied if the closed loop system is to be stable

$$FP_1 + Q_1 = M \quad (15)$$

The structure of the polynomial matrices of the controller were chosen to suit physical demands.

$$P_1(z^{-1}) = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \quad (16)$$

$$Q_1(z^{-1}) = \begin{bmatrix} q_1 z^{-1} & 0 \\ 0 & q_2 z^{-1} \end{bmatrix}$$

Consequently, matrix M was chosen to be

$$M(z^{-1}) = \begin{bmatrix} 1 + m_1 z^{-1} & 0 \\ 0 & 1 + m_2 z^{-1} \end{bmatrix} \quad (17)$$

The controller parameters are the result from the equation (15). The control law can be described by matrix equation

$$FU = B^{-1} A Q_1 P_1^{-1} E \quad (18)$$

This controller is unsuitable for non – minimum phase systems.

Compensator C_2 is adjugated matrix B . When C_2 was included in the design of the closed loop the model was simplified by considering matrix A as diagonal. The multiplication of matrix B and adjugated matrix B results in diagonal matrix H . The determinants of matrix B represent the diagonal elements. When matrix A is nondiagonal, its inverted form must be placed ahead of the system in order to obtain diagonal matrix H , otherwise it may increase the order of the controller and sophistication of the closed loop system. Although designed for a diagonal matrix, compensator C_2 also improves the control process for non – diagonal matrix A in the controlled system. This is demonstrated in the simulation results.

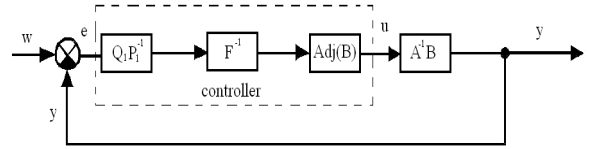


Fig. 5 The closed loop system with compensator C_2

The equation for the system output as shown in this block diagram takes the form

$$Y = P_1(AFP_1 + B_v Q_1) B_v Q_1 P_1^{-1} W \quad (19)$$

where

$$B_v = B \text{adj}(B) = \begin{bmatrix} \det(B) & 0 \\ 0 & \det(B) \end{bmatrix} \quad (20)$$

To achieve stability in the closed loop system the following diophantine equation must be fulfilled

$$AFP_1 + B_v Q_1 = M \quad (21)$$

The controller polynomial matrices are chosen as shown below

$$P_1(z^{-1}) = \begin{bmatrix} 1 + p_1 z^{-1} & p_2 z^{-1} \\ p_3 z^{-1} & 1 + p_4 z^{-1} \end{bmatrix} \quad (22)$$

$$Q_1(z^{-1}) = \begin{bmatrix} q_1 + q_2 z^{-1} + q_3 z^{-2} & q_4 + q_5 z^{-1} + q_6 z^{-2} \\ q_7 + q_8 z^{-1} + q_9 z^{-2} & q_{10} + q_{11} z^{-1} + q_{12} z^{-2} \end{bmatrix}$$

and matrix M is

$$M(z^{-1}) = \begin{bmatrix} 1 + m_1 z^{-1} + m_2 z^{-2} + m_3 z^{-3} + m_4 z^{-4} + m_5 z^{-5} & 0 \\ 0 & 1 + m_6 z^{-1} + m_7 z^{-2} + m_8 z^{-3} + m_9 z^{-4} + m_{10} z^{-5} \end{bmatrix} \quad (23)$$

Solving the diophantine equation defines a set of algebraic equations which we subsequently use to obtain the unknown controller parameters.

The control law is given by the block diagram

$$FU = \text{adj}(B) Q_1 P_1^{-1} E \quad (24)$$

5 IDENTIFICATION

The algorithms designed here were incorporated into an adaptive control system with recursive identification. The recursive least squares method proved effective for self-tuning controllers (Kulhavý 1987) and was used as the basis for our algorithm. For our two-variable example we considered the disintegration of identification into two independent parts.

The parameter vectors are completed as shown below:

$$\begin{aligned}\boldsymbol{\theta}_1^T(k) &= [a_1, a_2, a_3, a_4, b_1, b_2, b_3, b_4] \\ \boldsymbol{\theta}_2^T(k) &= [a_5, a_6, a_7, a_8, b_5, b_6, b_7, b_8]\end{aligned}\quad (25)$$

The data vector is

$$\begin{aligned}\boldsymbol{\phi}^T(k-1) &= [-y_1(k-1), -y_1(k-2), -y_2(k-1), \\ &-y_2(k-2), u_1(k-1), u_1(k-2), u_2(k-1), u_2(k-2)]\end{aligned}\quad (26)$$

The parameter estimates are actualized using the recursive least squares method plus directional forgetting.

6 SIMULATION

Matlab + Simulink for Windows (The MathWork, Inc.) were used to create a program and diagrams to simulate and verify the algorithms. Verification by simulation was carried out on a range of systems with varying dynamics. The control of the model below is given here as our example.

$$\begin{aligned}\mathbf{A}(z^{-1}) &= \begin{bmatrix} 1+0,3z^{-1}+0,1z^{-2} & 0,1z^{-1}+0,2z^{-2} \\ 0,1z^{-1}+0,3z^{-2} & 1+0,3z^{-1}+0,1z^{-2} \end{bmatrix} \\ \mathbf{B}(z^{-1}) &= \begin{bmatrix} 0,1z^{-1}+0,4z^{-2} & 0,9z^{-1}+0,4z^{-2} \\ 0,6z^{-1}+0,2z^{-2} & 0,3z^{-1}+0,4z^{-2} \end{bmatrix}\end{aligned}\quad (27)$$

Fig. 6 shows the system's step response

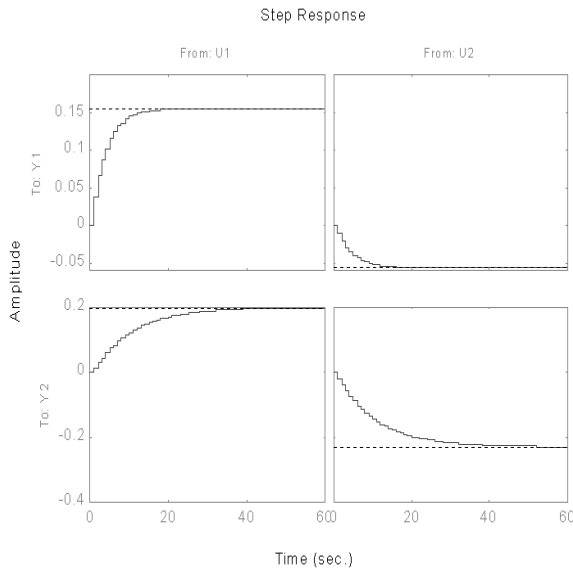


Fig. 6 The step response of the system

The right side control matrices are denoted as follows: without compensator - \mathbf{M}_1 , with compensator \mathbf{C}_1 - \mathbf{M}_2 , and with compensator \mathbf{C}_2 - \mathbf{M}_3 .

$$\begin{aligned}\mathbf{M}_1(z^{-1}) &= \begin{bmatrix} (1-0,1z^{-1})^4 & 0 \\ 0 & (1-0,1z^{-1})^4 \end{bmatrix} \\ \mathbf{M}_2(z^{-1}) &= \begin{bmatrix} 1-0,1z^{-1} & 0 \\ 0 & 1-0,1z^{-1} \end{bmatrix}\end{aligned}\quad (28)$$

$$\mathbf{M}_3(z^{-1}) = \begin{bmatrix} (1-0,1z^{-1})^5 & 0 \\ 0 & (1-0,1z^{-1})^5 \end{bmatrix}$$

The same initial conditions for system identification were used for all the types of adaptive control we tested. The initial parameter estimates were chosen to be

$$\boldsymbol{\theta}_1^T(0) = [0,1,0,2,0,3,0,4,0,1,0,2,0,3,0,4] \quad (29)$$

$$\boldsymbol{\theta}_2^T(0) = [0,5,0,6,0,7,0,8,0,5,0,6,0,7,0,8]$$

The results of simulation are shown in Figs 7 – 12.

We can draw several conclusions from the simulation results of our experiments on linear static systems. The basic requirement to ensure permanent zero control error was satisfied in all cases. The criteria on which we judge the quality of the control process are the overshoot on the controlled values and the speed with which zero control error is achieved. According to these criteria the controller incorporating compensator \mathbf{C}_1 performed the best. However, this controller appears to be unsuited to adaptive control due to the size of the overshoot and the large spread of process and controller outputs. The controller which uses compensator \mathbf{C}_2 seems to work best in adaptive control. With regards to decoupling, it is clear that controllers with compensators greatly reduce interaction.

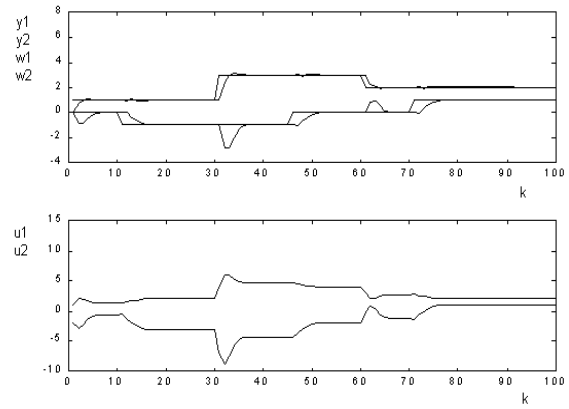


Fig. 7 Deterministic control without compensator

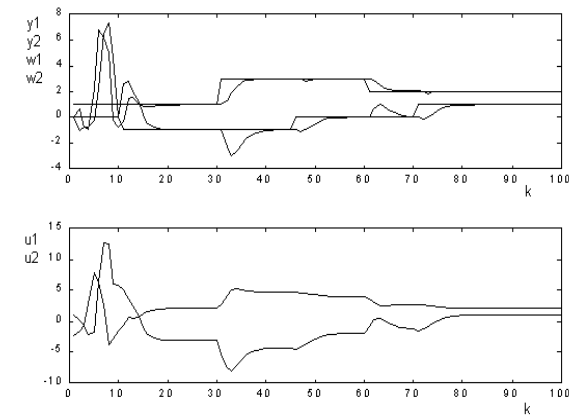


Fig. 8 Adaptive control without a compensator

7 VERIFICATION – CONTROLLING A LABORATORY MODEL

Suggested controllers has been verified for the control of the laboratory through – flow air heater (Fig. 13). This laboratory equipment is two input – two output system. Manipulated variables are the heat source (electric resistance heating 2) and the air flow source (ventilator 1). Controlled variables are the air temperature, measured by resistance thermometer 4 and air flow, measured by flow speed indicator (position 7). The air flow can be changed with the throttle valve (position 5). The task was to apply the methods we designed for the adaptive control of a model representing a non-linear system with variable parameters which is, therefore, impossible to control deterministically.

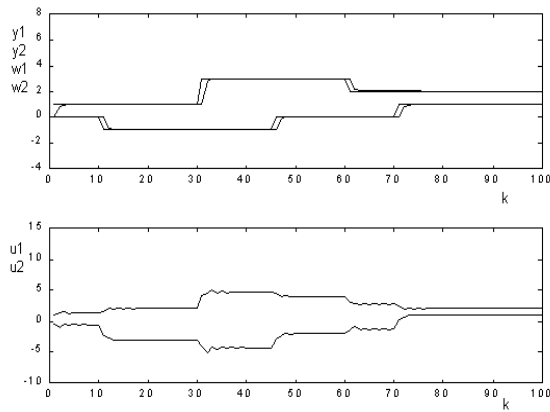


Fig. 9 Deterministic control with compensator C_1

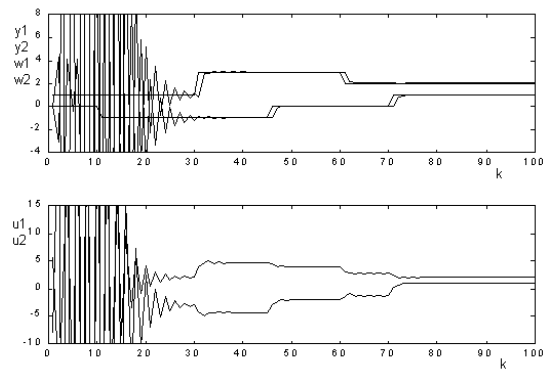


Fig. 10 Adaptive control with compensator C_1

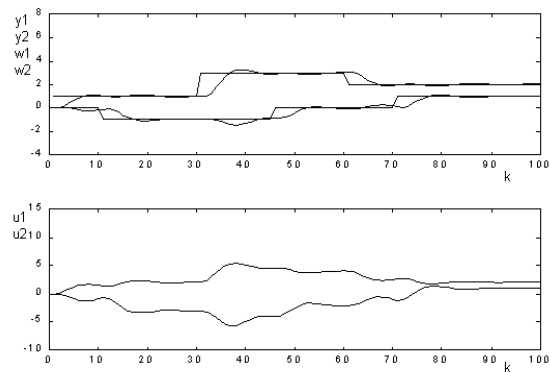


Fig. 11 Deterministic control with compensator C_2

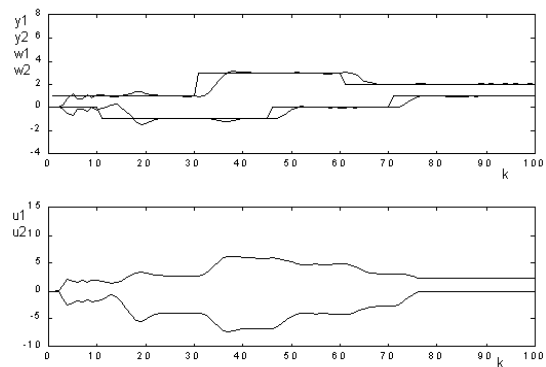


Fig. 12 Adaptive control with compensator C_2

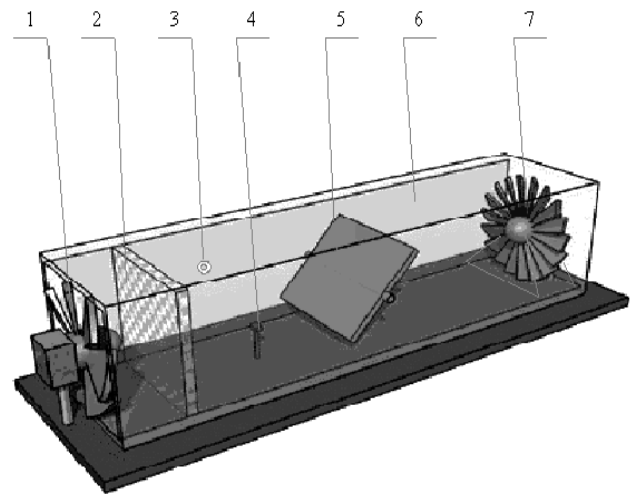


Fig. 13 Laboratory through – flow air heater

1 – ventilator, 2 – electric resistance heating , 3 – pressure sensor, 4 – resistance thermometer, 5 – throttle valve, 6 – cover of tunnel, 7 – flow indicator

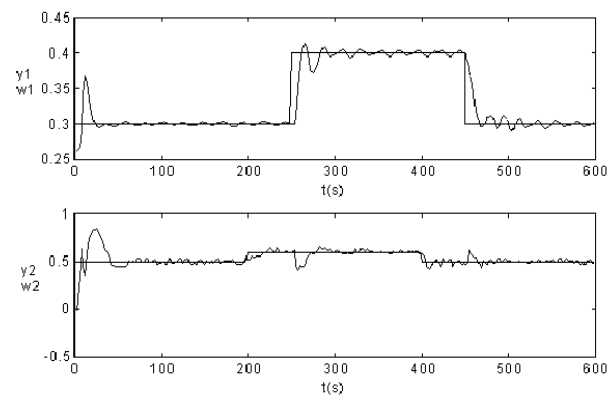


Fig. 14 Adaptive control of the real model using compensator C_2

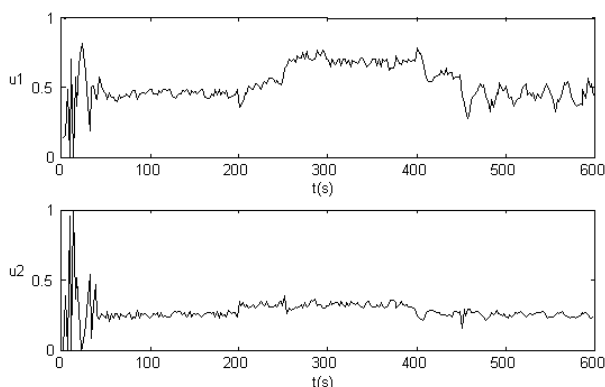


Fig. 15 The adaptive control of a real model using compensator C_2 — controller output

Adaptive control using recursive identification both with and without the use of compensators was performed. As indicated in the simulation, compensator C_1 was shown to be unsuitable and control broke down. The other two methods gave satisfactory results. The time responses of the control for both cases are shown in Fig. 14, Fig. 15, Fig. 16 and Fig. 17. The figures demonstrate that control with a compensator reduces interaction. The controlled variable y_1 is the air temperature and the controlled variable y_2 is the air flow. The manipulated variable u_1 is the heat source and the manipulated variable u_2 is the air flow source.

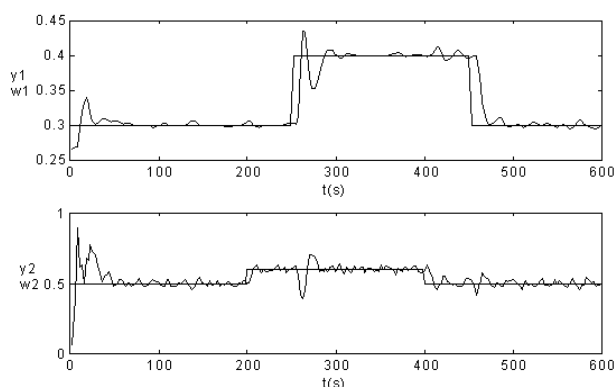


Fig. 16 Adaptive control of the real model without a compensator

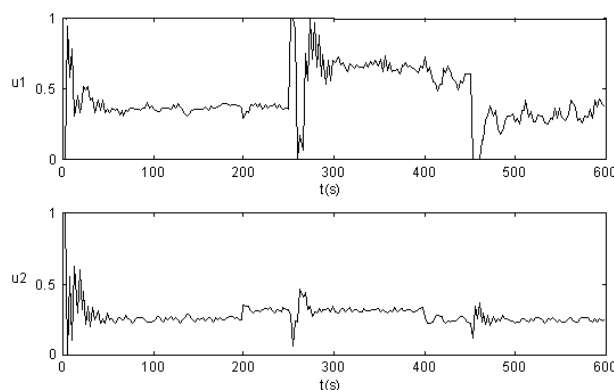


Fig. 17 Adaptive control of the real model without a compensator – controller output

8 CONCLUSIONS

The adaptive control of a two-variable system based on polynomial theory was designed. Decoupling problems were solved by the use of compensators. The designs were simulated and used to control a laboratory model. The simulation results proved that these methods are suitable for the control of linear systems. The control tests on the laboratory model gave satisfactory results despite the fact that the non-linear dynamics were described by a linear model.

ACKNOWLEDGMENTS

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