

Process control TKP4140 – Midterm Exam

October 2016

Student number: SOLUTION (short)

- Write your **student number** on **every** page.
- Write your answers in the designated spaces.
- Do **not** separate the sheets.
- If you need extra space, use the extra pages in the end.

Problem 1

1. (7 points) The transfer function $g(s)$ is a first order transfer function with time delay and the output is $y(s) = g(s)u(s)$. The step response $y(t)$ is plotted in Figure 1, and:

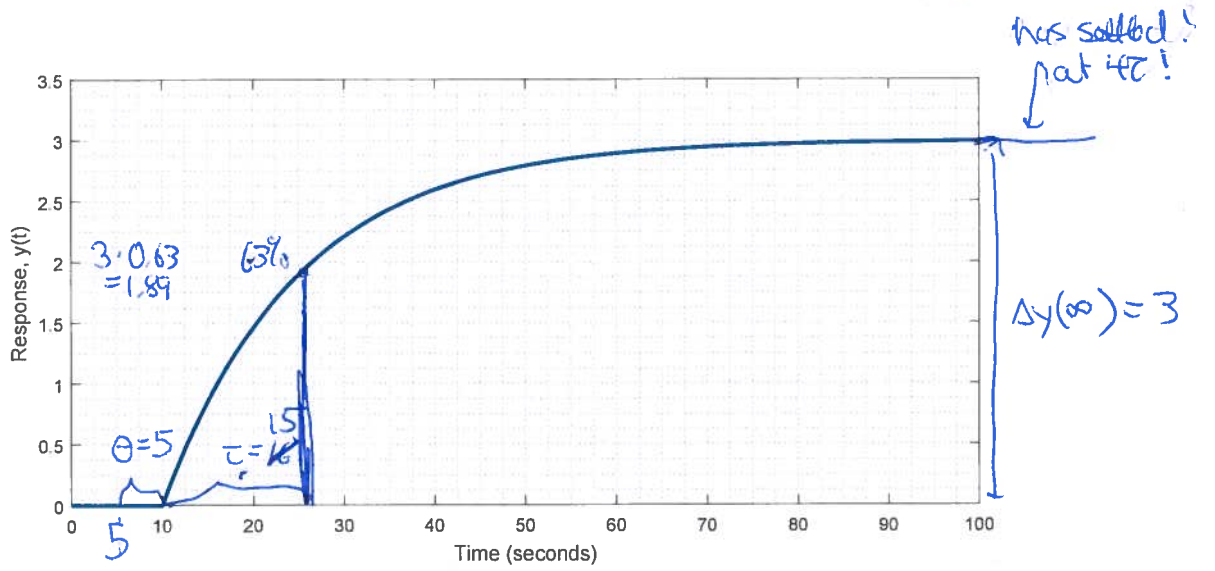
$$u(t) = \begin{cases} 0 & \text{for } t < 5 \\ 2 & \text{for } t \geq 5 \end{cases}$$

- 2 (a) Sketch $u(t)$ in Figure 1.
- 2 (b) Indicate the time delay (θ), time constant (τ), and steady state value (y_∞) in Figure 1.
- (c) Write down θ , τ , k , and $g(s)$.

3 $\theta = 5s$ $\tau = 15s$ $k = \frac{\Delta y_\infty}{\Delta u} = \frac{3}{2} = 1.5$

$$g(s) = \frac{1.5e^{-5s}}{15s+1}$$

(3)



(2)

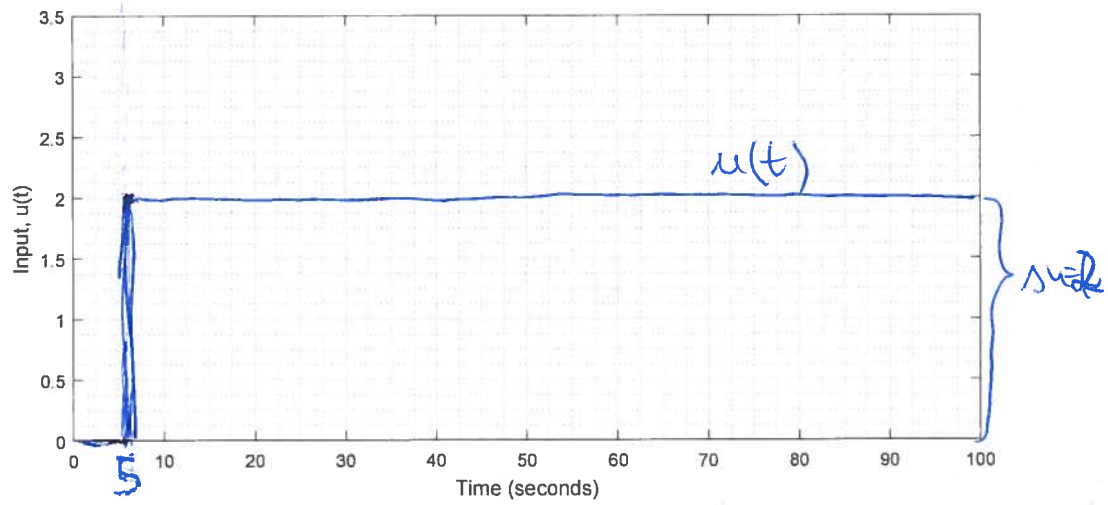


Figure 1: First order transfer function.

Problem 2

2. (18 points) Consider a heated tank with perfectly controlled level as the one in Figure 2. Assume: constant density (ρ), constant heat capacity (C_p), constant volume (V), perfect mixing.

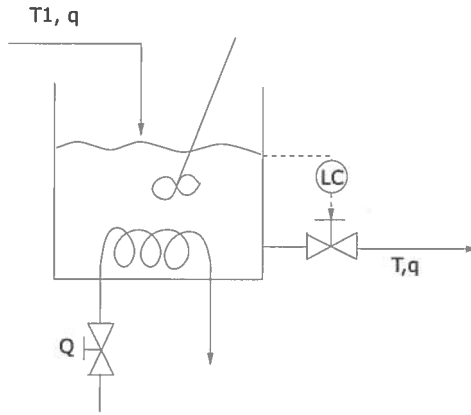


Figure 2: Heated tank

Table 1: Parameters for the heated tank.

V	$=$	10 000	ℓ
q	$=$	1.5	m^3/s
ρ	$=$	1000	kg/m^3
$C_v \approx C_p$	$=$	4200	J/kgK

- (a) Formulate the dynamic energy balance of the system and write it in the form $\frac{dT}{dt} = \dots$

Energy balance: $\frac{dU}{dt} = \underbrace{H_{in} - H_{out}}_{\text{flows}} + Q + \underbrace{U_s}_{\downarrow 0}$

6

$$\rho V C_p \frac{dT}{dt} = \rho q C_p (T_1 - T) + Q$$

$$\Rightarrow \boxed{\frac{dT}{dt} = \frac{q}{V} (T_1 - T) + \frac{Q}{\rho V C_p}}$$

- (b) Linearize the model and introduce deviation variables. $\Delta T = T - T^c$ etc.

3

Linearize: $\frac{d\Delta T}{dt} = \frac{T_1 - T^c}{V} \Delta q + \frac{q^c}{V} (\Delta T_1 - \Delta T) + \frac{1}{\rho V C_p} \Delta Q$

Laplace assuming $\Delta q = 0$

$$s \Delta T(s) = \frac{q^c}{V} \Delta T_1(s) - \frac{q^c}{V} \Delta T(s) + \frac{1}{\rho V C_p} \Delta Q(s)$$

3

$$\Rightarrow \Delta T(s) = \frac{q^c V}{s + q^c/V} \Delta T_1(s) + \frac{1/\rho V C_p}{s + q^c/V} \Delta Q(s)$$

$$= \underbrace{\frac{1}{\tau_1 s + 1}}_{g_1(s)} \Delta T_1(s) + \underbrace{\frac{1}{\rho q C_p} \frac{1}{s + 1}}_{g_2(s)} \Delta Q(s) \quad \text{where } \tau_1 = \frac{\rho V}{q^c}$$

(c) Take the Laplace transform and derive the transfer functions $g_1(s)$ and $g_2(s)$:

$$T(s) = g_1 T_1(s) + g_2(s) Q(s)$$

$$g_1(s) = \frac{1}{\tau s + 1}$$

$$g_2(s) = \frac{k_2}{\tau s + 1}$$

$$\tau = \frac{V}{\dot{q}}$$

$$k_2 = \frac{1}{\rho q c_p}$$

(d) Considering the parameters in Table 1, what is the value of the steady state gain k_1 and the time constant τ_1 of $g_1(s)$? (write the units)

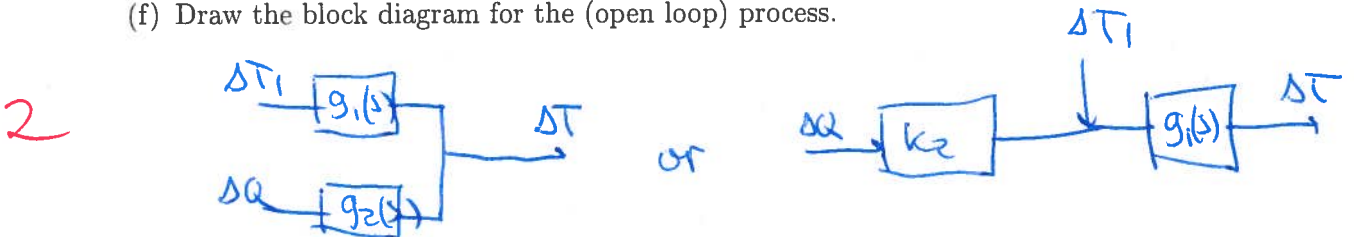
2 $k_1 = 1 \text{ [K/K]}$

$$\tau_1 = \frac{V}{\dot{q}} = \frac{100000}{1500} \text{ s} = \underline{\underline{6.7 \text{ s}}}$$

(e) Considering the parameters in Table 1, what is the value of the steady-state gain k_2 of $g_2(s)$? (write the units)

2 $k_2 = \frac{1}{\rho q c_p} = \frac{1}{1000 \frac{\text{kg}}{\text{m}^3} \cdot 1.5 \frac{\text{m}^3}{\text{s}} \cdot 4200 \frac{\text{J}}{\text{kg} \cdot \text{K}}} = \underline{\underline{0.158 \cdot 10^{-6} \frac{\text{K}}{\text{J/s}}}}$
 $= \underline{\underline{1.59 \cdot 10^{-4} \text{ K/kW}}}$

(f) Draw the block diagram for the (open loop) process.



Problem 3

3. (30 points) Consider the following block diagram:

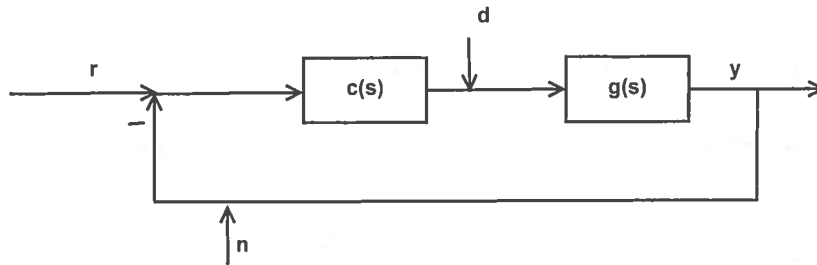


Figure 3: Closed loop block diagram

General rule
"direct"
T.F. = $\frac{\text{"direct"}}{1 + \text{loop}}$

$$y = T(s) r + M(s) d + N(s) n$$

(a) Write the transfer functions $T(s)$, $M(s)$ and $N(s)$. Use symbols only ($c(s)$, $g(s)$).

$$2 \quad T(s) = \frac{gc}{1+gc}$$

$$2 \quad M(s) = \frac{g}{1+gc}$$

$$2 \quad N(s) = \frac{-gc}{1+gc}$$

(b1) Find $T(s)$ and $M(s)$ when: $g(s) = \frac{3}{4s+1}$ and $c(s) = 2$
Write the denominator in the form: $\tau s + 1$

$$2 \quad T(s) = \frac{gc}{1+gc} = \frac{6}{4s+1+6} = \frac{6/7}{\frac{4}{7}s+1} = \frac{0.857}{0.575s+1}$$

$$2 \quad M(s) = \frac{g}{1+gc} = \frac{3}{4s+1+6} = \frac{3/7}{\frac{4}{7}s+1} = \frac{0.428}{0.575s+1}$$

(b2) Find the analytical expression for $y(t)$ when $r(t)$ is a unit step, $d(t) = 0$, and $n(t) = 0$

$$y(t) = 0.857(1 - e^{-t/0.57})$$

Proof:

You should remember that for $g(s) = \frac{k}{cs+1}$ the step response is: $y(t) = k(1 - e^{-t/\tau}) \Delta(t)$

$$= 0.857(1 - e^{-t/0.57}) \underbrace{\Delta(t)}_{=1}$$

(c1) Find $T(s)$ when: $g(s) = \frac{3}{4s+1}$ and $c(s) = 0.5(1 + \frac{1}{s}) = 0.5 \frac{s+1}{s}$
 → Please note that this is not a well-tuned controller.

$$T(s) = \frac{gc}{1+gc} = \frac{1.5(s+1)}{(4s+1) \cdot s + 1.5(s+1)} = \frac{s+1}{\frac{4}{1.5}s^2 + \frac{2.5}{1.5}s + 1}$$

Standard form

$$T = \frac{s+1}{\underbrace{2.67}s^2 + \underbrace{1.67}s + 1} \quad (*)$$

(c2) Calculate the damping factor and the time constant of $T(s)$.

Hint: the denominator in the second order transfer function is $\tau^2 s^2 + 2\tau\zeta s + 1$

$$\zeta = 0.510$$

$$\tau = \sqrt{2.67} = 1.63$$

Proof: ↑

$$2\tau\zeta = 1.67$$

$$\zeta = \frac{1.67}{2 \cdot 1.63} = 0.510$$

(c3) Compute $y(0)$, $y'(0)$, and $y(\infty)$ for $T(s)$ when a unit step input $r(s) = 1/s$ is applied.

$y(0) = 0$

$y'(0) = 0.37$

$y(\infty) = 1$

$$T(s) = \frac{s+1}{2.67s^2 + 1.67s + 1}$$

Step response for unit r .

- 4
- 1 $y(\infty) = T(0) = 1$ (as expected, no steady-state offset with integral action in the controller)
 - 2 $y'(0) = \lim_{s \rightarrow \infty} sT(s) = \lim_{s \rightarrow \infty} \frac{s^2}{2.67s^2} = 0.37$
 - 1 $y(0) = \lim_{s \rightarrow \infty} T(s) = \frac{1}{\infty} = 0$

(c4) Sketch the response $y(t)$, when $r(t)$ is a unit step, $d = 0$ and $n = 0$.

Hints:

- Consider the answers you gave in (c2) and (c3).
- Note that the period of oscillations is approximately $2\pi\tau$
- The first peak is at $t \approx 4.7s$

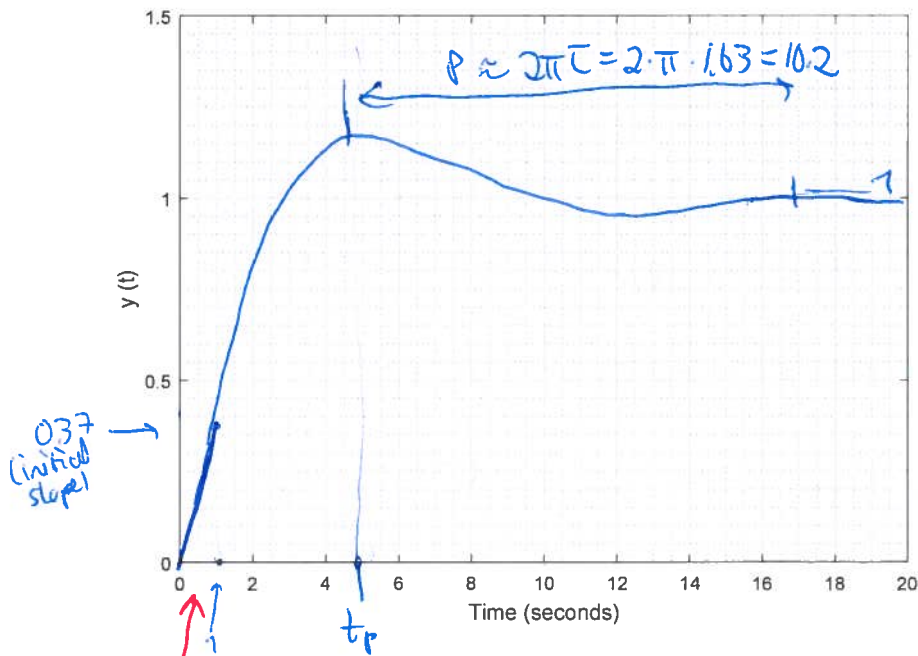


Figure 4: Step response

Stand out with initial slope 0.37/1

Problem 4

4. (15 points) Given the transfer function:

$$g(s) = \frac{(-5s + 1)e^{-3s}}{(2s + 1)^3(7s + 1)} \quad (1)$$

(a) Write the first order plus time delay approximation $g_1(s)$ using the half-rule.

4

$$g_1(s) = \frac{e^{-13s}}{8s + 1}$$

$\theta = \frac{\tau_{20}}{2} + \tau_{30} + \tau_{40} + T + \theta_0$
 $\tau_1 = \tau_1 + \frac{\tau_{20}}{2}$

$\theta = \frac{2}{2} + 2 \cdot 2 + 5 + 3 = 13$ $\tau_1 = 7 + \frac{2}{2} = 8$

(b) Give the SIMC PI settings, using $\tau_c = \theta$, where θ is the effective delay.

2

$$K_c = \frac{1}{k} \frac{\tau_1}{\tau_c + \theta} = \frac{8}{20} = 0.31$$

$\tau_I = \min(\tau_1, \frac{4(\tau_c + \theta)}{10}) = 8$

$\tau_c = \theta = 13$

(c) Write the second order plus time delay approximation $g_2(s)$ using the half-rule.

4

$$g_2(s) = \frac{e^{-11s}}{(7s + 1)(3s + 1)}$$

$\theta = \frac{2}{2} + 2 + 5 + 3 = 11$ $\tau_1 = 7$ $\tau_2 = 2 + \frac{2}{2} = 3$

(d) Give the SIMC PID settings (cascade PID), using $\tau_c = \theta$, where θ is the effective delay.

2

$$K_c = \frac{1}{k} \frac{\tau_1}{\tau_c + \theta} = 0.32$$

$\tau_I = \tau_c = 7$ $\tau_D = 3$

(e) Would you recommend a PI or a PID controller? Explain briefly.

3 PI, since τ_2 is much less than θ so there is little benefit with PID. Also note that K_c is almost the same, so little benefit.

Problem 5

5. (5 points) Given the transfer function:

$$g(s) = \frac{20e^{-3s}}{s(2s+1)(s+1)} \quad (2)$$

(a) Write the first order plus time delay approximation $g_1(s)$ using the half-rule.

$g_1(s) = \frac{20e^{-5s}}{s}$

This will actually be an integrating + time delay process!
 $\theta = \theta_0 + \frac{\tau_2}{2} + \tau_3 = 5$
 $\frac{1}{3}$ $\frac{2}{2}$ 1

Comment 2nd order model would be
 $g_2 = \frac{20e^{-3.5s}}{s(2.5s+1)}$

(b) Give the SIMC PI settings. Use $\tau_c = \theta$, where θ is the effective delay.

$K_c = \frac{1}{K} \frac{1}{\tau_c + \theta} = \frac{1}{200}$
 \uparrow $5+5$
 20

$\tau_I = 4(\tau_c + \theta) = 4 \cdot \left(\frac{5}{10} + \frac{5}{10}\right) = \underline{\underline{40}}$

Problem 6

6. (5 points) Indicate whether the following statements are true or false.

	True	False
The PID SIMC rule gives tunings for PID in ideal form.		X
Increasing the integral term τ_I in a PID controller increases the effect of integral action.		X
After tuning a PI controller using the SIMC rules with $\tau_c = \theta$, you realized that the closed loop response is faster than what you would like it to be. In order to slow down the response, you should decrease τ_c .		X
A closed loop with P-control always has a steady-state offset of $\frac{1}{1+K_c K}$, where K_c is the controller gain and K is the steady state process gain.	X	
Windup is caused by the integral part of the PI controller.	X	

Problem 7

7. (20 points) Figure 5 depicts the responses of the following transfer functions to a step input:

$$g_1 = \frac{1}{7s+1} - \frac{1.1}{3s+1} = \frac{(-4.75s - 0.1)}{(7s+1)(3s+1)} = \frac{0.1(47s+1)}{(7s+1)(3s+1)} \quad (3)$$

$$g_2 = \frac{1}{3s+1} - \frac{1.1}{7s+1} = \frac{7s+1 - 1.1(3s+1)}{(7s+1)(3s+1)} = \frac{3.7s+0.1}{(7s+1)(3s+1)} \quad (4)$$

$$g_3 = \frac{(10s+1)}{(7s+1)(1.5s+1)^2} \quad (5)$$

$$g_4 = \frac{1}{s^2 + 0.4s + 1} \quad (6)$$

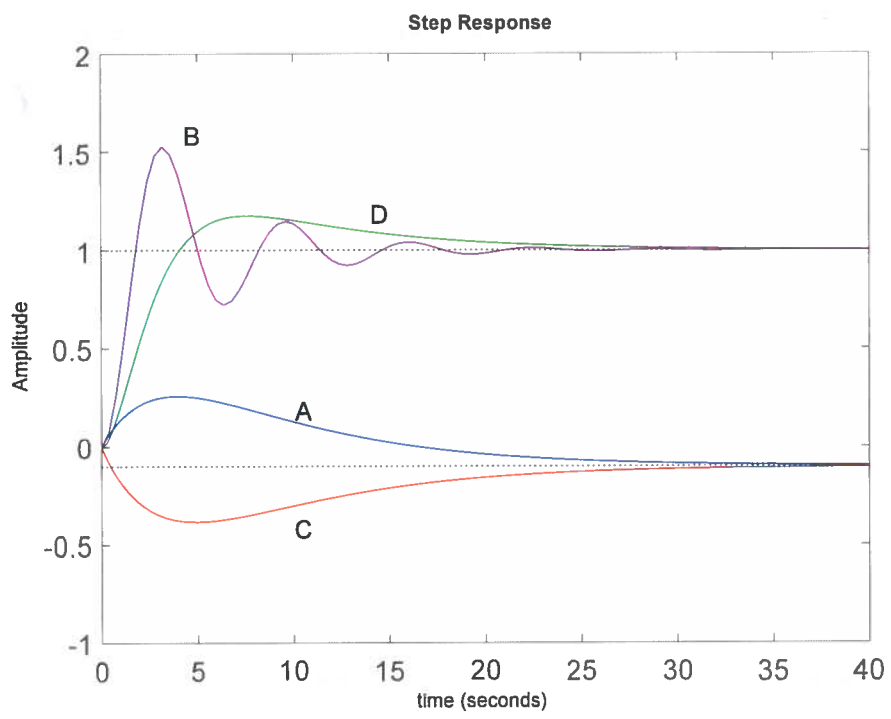


Figure 5: Step response of $g_i(s)$

Fill in the missing values in Table 2:

- Poles and zeros of $g_i(s)$
- Steady state gain (SS gain), initial gain and initial slope when a unit step input $u(s) = 1/s$ is applied at $t = 0$.
- As conclusion, identify the step responses in Figure 5 (A,B,C, or D).

Hints:

Initial slope of response to unit step input: $\lim_{t \rightarrow 0} y'(t) = \lim_{s \rightarrow \infty} s g(s)$

$j = \sqrt{-1}$

Table 2: Problem 7; SS: steady state; TF: transfer function

TF	Poles	Zeros	SS gain	Initial gain	Initial slope	Conclusion
g_1	$p_1 = -1/7$ $p_2 = -1/3$	$z_1 = -1/47$	-0.1	0	$-\frac{47 \cdot 0.1}{21}$ $= -0.224$	C
g_2	$p_1 = s_1 = -1/7$ $p_2 = s_2 = -1/3$	$z_1 = 1/37$ (RHP) (invers response)	-0.1	0	$\frac{-0.1 \cdot (-37)}{21}$ $= 0.176$	A ← invers response
g_3	$p_1 = -1/9$ $p_2 = -1/1.5$ $p_3 = -1/5.5$	$z_1 = -1/10$	1	0	0	D
g_4	Complex poles $p_{1,2} = -0.2 \pm j0.98$	None	1	0	0	B ↑ oscillations

$$s = \frac{-B \pm \sqrt{B^2 - 4AC}}{2A} = \frac{-0.4 \pm \sqrt{0.4^2 - 4 \cdot 1 \cdot 1}}{2}$$

$$= -0.2 \pm j \frac{1.96}{2} = \underline{\underline{-0.2 \pm 0.98j}}$$

initial gain = $g(\infty)$

initial slope = $y'(0^+) = \lim_{s \rightarrow \infty} s g(s)$

Extra space if needed

Please clearly indicate which problem the solution belongs to.