

Ch. 13

Frequency analysis

Force linear system with input $x(t) = A \sin \omega t$.
Here is the output $y(t)$:

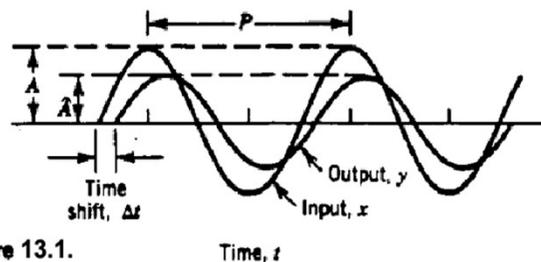


Figure 13.1.

Time, t

Attenuation and time shift between input and output sine waves ($K = 1$). The phase angle ϕ of the output signal is given by $\phi = -\Delta t/P \times 360^\circ$, where Δt is the time (period) shift and P is the period of oscillation.

4.2.3 Sinusoidal Response

As a final example of the response of first-order processes, consider a sinusoidal input $u_{sin}(t) = A \sin \omega t$ with transform given by Eq. (4-15):

$u(t) = A \sin(\omega t)$

As $t \rightarrow \infty$:
 $y(t) = AR * A * \sin(\omega t + \phi)$

$$u(s) = A \frac{\omega}{s^2 + \omega^2} \tag{4-23}$$

$$y(s) = \frac{KA\omega}{(\tau s + 1)(s^2 + \omega^2)} \tag{4-24}$$

$$= \frac{KA}{\omega^2 \tau^2 + 1} \left(\frac{\omega \tau^2}{\tau s + 1} - \frac{s \omega \tau}{s^2 + \omega^2} + \frac{\omega}{s^2 + \omega^2} \right)$$

Inversion gives

$$y(t) = \frac{KA}{\omega^2 \tau^2 + 1} (\omega \tau e^{-t/\tau} - \omega \tau \cos \omega t + \sin \omega t) \tag{4-25}$$

or, by using trigonometric identities,

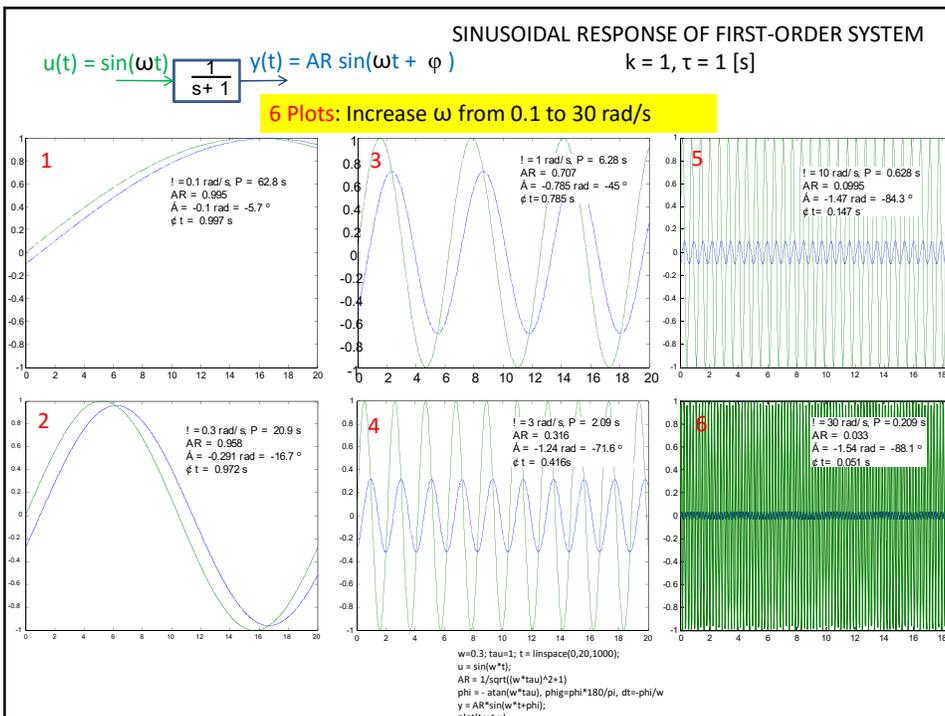
$$y(t) = \frac{KA\omega \tau}{\omega^2 \tau^2 + 1} e^{-t/\tau} + \frac{KA}{\sqrt{\omega^2 \tau^2 + 1}} \sin(\omega t + \phi) \tag{4-26}$$

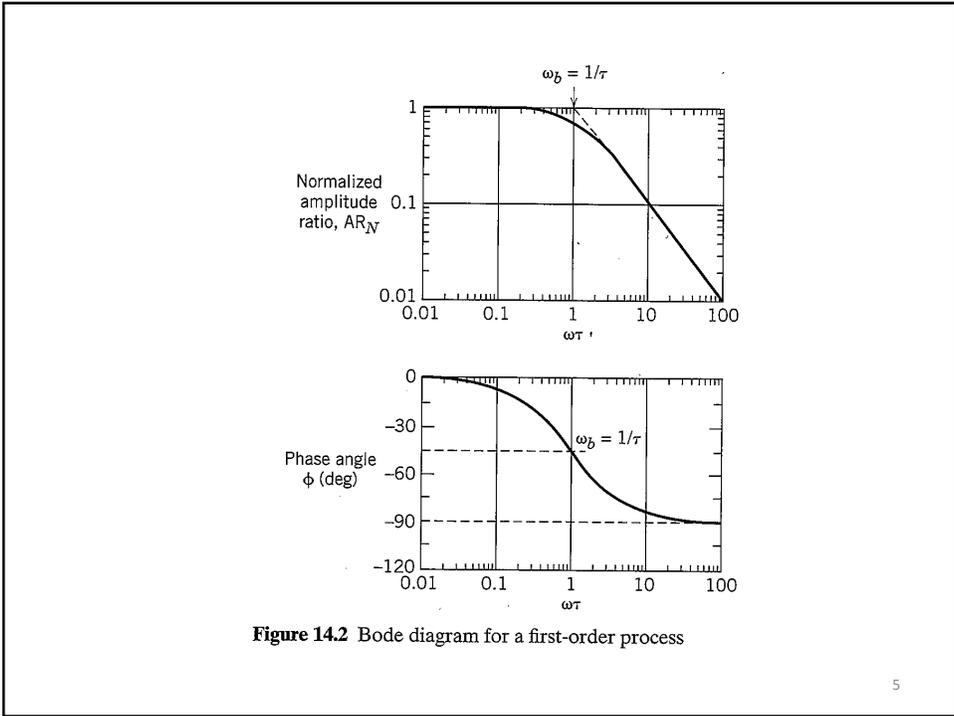
where

$$\phi = -\tan^{-1}(\omega \tau) \tag{4-27}$$

General (VERY SIMPLE).
Set $s=j\omega$ in $G(s)$. Then
 $AR = |G(j\omega)|$
 $\phi = \angle G(j\omega)$

Notice that in both (4-25) and (4-26) the exponential term goes to zero as $t \rightarrow \infty$, leaving a pure sinusoidal response. This property is exploited in Chapter 13 for frequency response analysis.





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Mathematics. Complex numbers, $j^2=-1$

The diagram shows a complex plane with a horizontal axis labeled 'Re(G)' and a vertical axis labeled 'Im(G)'. A vector originates from the origin and points into the first quadrant. The horizontal component is labeled 'R' and the vertical component is labeled 'I'. The vector is labeled $G(j\omega)=R+jI$. The angle between the vector and the positive real axis is labeled $A=\angle G$.

$$s = j\omega \quad G(j\omega) = R + jI$$

$$|G| = AR = \sqrt{R^2 + I^2}$$

$$\phi = \angle G = \arctan \frac{I}{R}$$

} Polar form

Polar form:
 $G = R + jI = |G|(\cos \angle G + j \sin \angle G) = |G|e^{j\angle G}$
 Note: $e^{j\pi} = -1$

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Polar form

Multiply complex numbers:

Multiply magnitudes and add phases

$$G = G_1 \cdot G_2 \cdot G_3$$

$$|G| = |G_1| \cdot |G_2| \cdot |G_3|$$

$$\angle G = \angle G_1 + \angle G_2 + \angle G_3$$

Similar – for – ratio :

$$G = \frac{G_1}{G_2}$$

$$|G| = |G_1| / |G_2|$$

$$\angle G = \angle G_1 - \angle G_2$$

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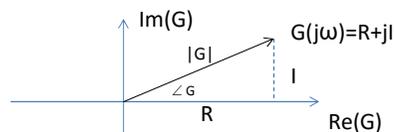
Simple method to find sinusoidal response of system $G(s)$

1. Input signal to linear system: $u = u_0 \sin(\omega t)$
2. Steady-state (“persistent”, $t \rightarrow \infty$) output signal: $y = y_0 \sin(\omega t + \phi)$
3. What is AR = y_0/u_0 and ϕ ?

Solution (extremely simple!)

1. Find system transfer function, $G(s)$
2. Let $s=j\omega$ (imaginary number, $j^2=-1$) and evaluate $G(j\omega) = R + jI$ (complex number)
3. Then (“believe it or not!”)

$$\boxed{\begin{array}{l} \text{AR} = |G(j\omega)| \\ \phi = \angle G(j\omega) \end{array}} \quad \begin{array}{l} \text{(magnitude of the complex number)} \\ \text{(phase of the complex number)} \end{array}$$



Proof: $y(s) = G(s)u(s)$ where $u(s) = \frac{u_0\omega}{s^2 + \omega^2} = \frac{u_0\omega}{(s-j\omega)(s+j\omega)}$, etc...
(poles of $G(s)$ “die out” as $t \rightarrow \infty$)

Example 13.1:

1. $G(s) = \frac{1}{\tau s + 1}$

2. $G(j\omega) = \frac{1}{1 + \tau j\omega} \cdot \frac{1 - \tau j\omega}{1 - \tau j\omega} \quad (j^2 = -1)$

$$G(j\omega) = \frac{1}{1 + \omega^2 \tau^2} - \frac{\omega \tau}{1 + \omega^2 \tau^2} j$$

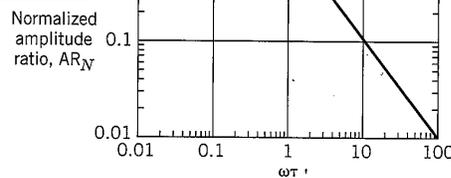
$\underbrace{\hspace{1.5cm}}_{\mathbf{R}} \quad \underbrace{\hspace{1.5cm}}_{\mathbf{I}}$

3. $|G| = AR = \sqrt{R^2 + I^2} = \frac{1}{\sqrt{1 + \omega^2 \tau^2}}$ } Gain and phase shift of sinusoidal response!
 $\phi = \angle G = \arctan \frac{I}{R} = -\arctan(\omega \tau)$ }

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SIMPLER: Use $G=G1/G2$, where $G1=1$, $G2=\tau s+1$.
 set $s=j\omega$. Get $|G|=1/|G2|=1/\sqrt{(w*\tau)^2+1}$, $\text{angle}(G)=0-\text{angle}(G2) = -\text{arctg}(w*\tau)$

$$AR = |G(j\omega)| = \frac{1}{\sqrt{(\omega\tau)^2 + 1}}$$



$$\phi = \angle G(j\omega) = -\arctan(\omega\tau)$$

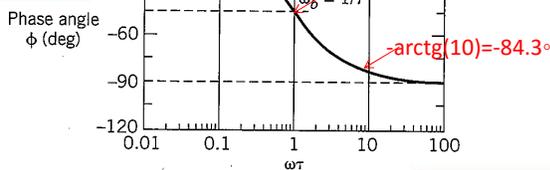


Figure 14.2 Bode diagram for a first-order process $G(s) = \frac{1}{\tau s + 1}$

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Example 2

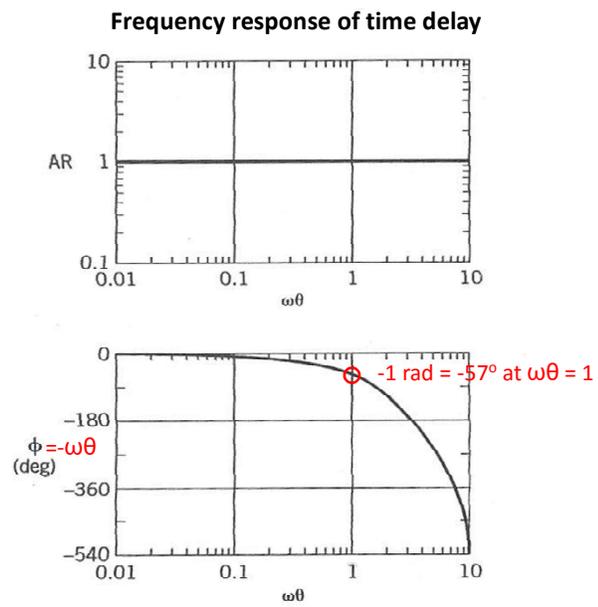
$$g(s) = \frac{k(Ts+1)}{(\tau_1s+1)(\tau_2s+1)} = \frac{g_1 g_2}{g_3 g_4}$$

$$g_1 = k$$

$$g_2 = Ts + 1$$

$$g_3 = \tau_1s + 1$$

$$g_4 = \tau_2s + 1$$

Figure 14.4 Bode diagram for a time delay, $e^{-\theta s}$.

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1. DERIVATIVE

$$g_1(s) = s$$

Frequency response: $g(j\omega) = j\omega = 0 + j\omega$

$$|g_1(j\omega)| = \omega$$

$$\angle g_1(j\omega) = 90^\circ = \pi/2 \text{ rad (purely complex at all } \omega)$$

Check:

$$u(t) = u_0 \sin(\omega t)$$

$$y(t) = u'(t) = u_0 \omega \cos(\omega t) = \omega u_0 \sin(\omega t + \pi/2) \quad \text{OK!}$$

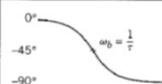
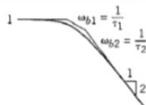
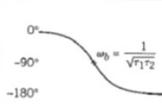
2. INTEGRATOR

$$g_2(s) = \frac{1}{s} = \frac{1}{g_1}$$

$$|g_2(j\omega)| = \frac{1}{|g_1|} = \frac{1}{\omega}$$

$$\angle g_2(j\omega) = 0^\circ - \angle g_1 = -90^\circ = -\pi/2 \text{ rad}$$

Table 13.2 Frequency Response Characteristics of Important Process Transfer Functions

Transfer Function	$G(s)$	AR = $ G(j\omega) $	Plot of log AR vs. log ω	$\phi = \angle G(j\omega)$	Plot of ϕ vs. log ω
1. First-order	$\frac{K}{\tau s + 1}$	$\frac{K}{\sqrt{(\omega\tau)^2 + 1}}$		$-\tan^{-1}(\omega\tau)$	
2. Integrator	$\frac{K}{s}$	$\frac{K}{\omega}$		-90°	
3. Derivative	Ks	$K\omega$		$+90^\circ$	
4. Overdamped second-order	$\frac{K}{(\tau_1 s + 1)(\tau_2 s + 1)}$	$\frac{K}{\sqrt{(\omega\tau_1)^2 + 1} \sqrt{(\omega\tau_2)^2 + 1}}$		$-\tan^{-1}(\omega\tau_1) - \tan^{-1}(\omega\tau_2)$	
5. Critically damped second-order	$\frac{K}{(\tau s + 1)^2}$	$\frac{K}{(\omega\tau)^2 + 1}$		$-2 \tan^{-1}(\omega\tau)$	

6. Underdamped second-order	$\frac{K}{\tau^2 s^2 + 2\zeta\tau s + 1}$	$\frac{K}{\sqrt{(1-(\omega\tau)^2)^2 + (2\zeta\omega\tau)^2}}$		$-\tan^{-1}\left[\frac{2\zeta\omega\tau}{1-(\omega\tau)^2}\right]$	
7. Left-half plane (positive) zero	$K(\tau_a s + 1)$	$K\sqrt{(\omega\tau_a)^2 + 1}$		$+\tan^{-1}(\omega\tau_a)$	
8. Right-half plane (negative) zero	$-\tau_a s + 1$	$K\sqrt{(\omega\tau_a)^2 + 1}$		$-\tan^{-1}(\omega\tau_a)$	
9. Lead-lag unit ($\tau_a < \tau_l$)	$K\frac{\tau_a s + 1}{\tau_l s + 1}$	$K\frac{\sqrt{(\omega\tau_a)^2 + 1}}{\sqrt{(\omega\tau_l)^2 + 1}}$		$+\tan^{-1}(\omega\tau_a) - \tan^{-1}(\omega\tau_l)$	
10. Lead-lag unit ($\tau_a > \tau_l$)	$K\frac{\tau_a s + 1}{\tau_l s + 1}$	$K\frac{\sqrt{(\omega\tau_a)^2 + 1}}{\sqrt{(\omega\tau_l)^2 + 1}}$		$+\tan^{-1}(\omega\tau_a) - \tan^{-1}(\omega\tau_l)$	
11. Time delay	$Ke^{-\theta s}$	K	1	$-\omega\theta$	

ASYMPTOTES

Frequency response of term $(Ts+1)$: set $s=j\omega$.

Asymptotes:

$(j\omega T + 1) \sim 1$ for $\omega T \ll 1$ (slope $n=0$, phase=0)

$(j\omega T + 1) \sim j\omega T$ for $\omega T \gg 1$ (slope $n=1$, phase=90°)

Gain slope n : $|G| \sim \omega^n$

Rule for asymptotic Bode-plot, $L = k(Ts+1)/(\tau s+1)$:

1. Start with low-frequency asymptote ($s \rightarrow 0$)

(a) If constant ($L(0)=k$):

Gain= k (slope=0)

Phase=0°

(b) If integrator ($L=k'/s$):

Gain slope= -1 (on log-log plot). Need one fixed point, for example, gain=1 at $\omega=k'$

Phase: -90°.

2. Break frequencies (order from large T to small T):

	Change in gain slope	Change in phase
$\omega=1/T$ (zero)	+1	+90° (-90° if T negative)
$\omega=1/\tau$ (pole)	-1	-90° (+90° if τ negative)

3. Time delay, $e^{-\theta s}$. Gain: no effect, Phase contribution: $-\omega\theta$ [rad] (-1 rad = -57° at $\omega=1/\theta$)

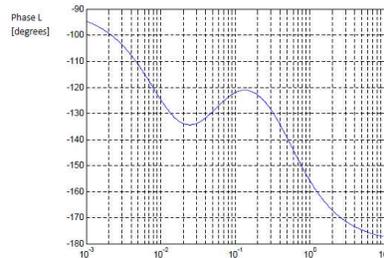
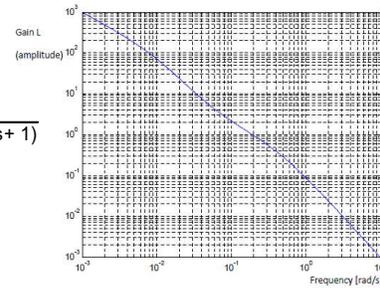
Example HAND OUT (in class)

TASK 1: Bode-plot of $L(s) = (20s+1)/[s(100s+1)(2s+1)]$. Write on the asymptotes

EXAMPLE

$$L(s) = \frac{20s+1}{s(100s+1)(2s+1)}$$

$L(s)=G(s)C(s)$:
 Loop transfer function for
 SIMC PI-control with $\tau_c=4$ for
 $G(s) = 1/(100s+1)(2s+1)$



```

s=tf('s')
L=(20*s+1)/(s*(100*s+1)*(2*s+1))
figure(1), bode(L), 'g', 'm', 'b'
w=logspace(-3, 2, 1000)
[mag, phase] = bode(L, w)
figure(1), loglog(w, mag), 'g', 'm', 'b'
figure(1), loglog(w, phase), 'g', 'm', 'b'
    
```

SOLUTION

$$L(s) = \frac{20s+1}{s(100s+1)(2s+1)}$$

$L(s)$: SIMC PI-control with $\tau_c=4$ for $g(s) = 1/(100s+1)(2s+1)$

Low-frequency asymptote ($s = j\omega \rightarrow 0$)
 is integrator: $L = \frac{1}{j\omega} = -\frac{1}{\omega}j$
 Gain = $\frac{1}{\omega}$ (slope -1 on log-log),
 Phase = -90°

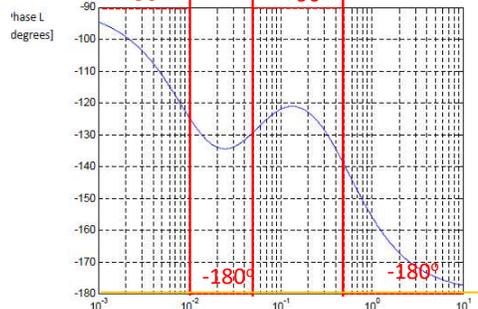
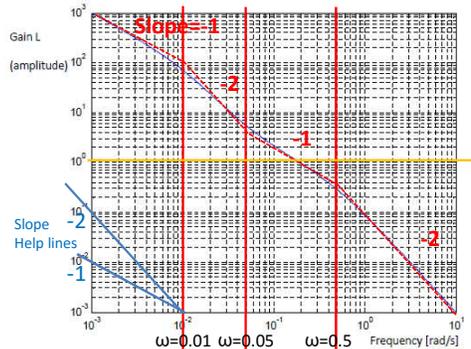
Asymptotes: Start at low frequency, $\omega \rightarrow 0$:
 $|L(j\omega)| = 1/\omega$. So: $|L| = 10^3$ at $\omega = 10^{-3}$

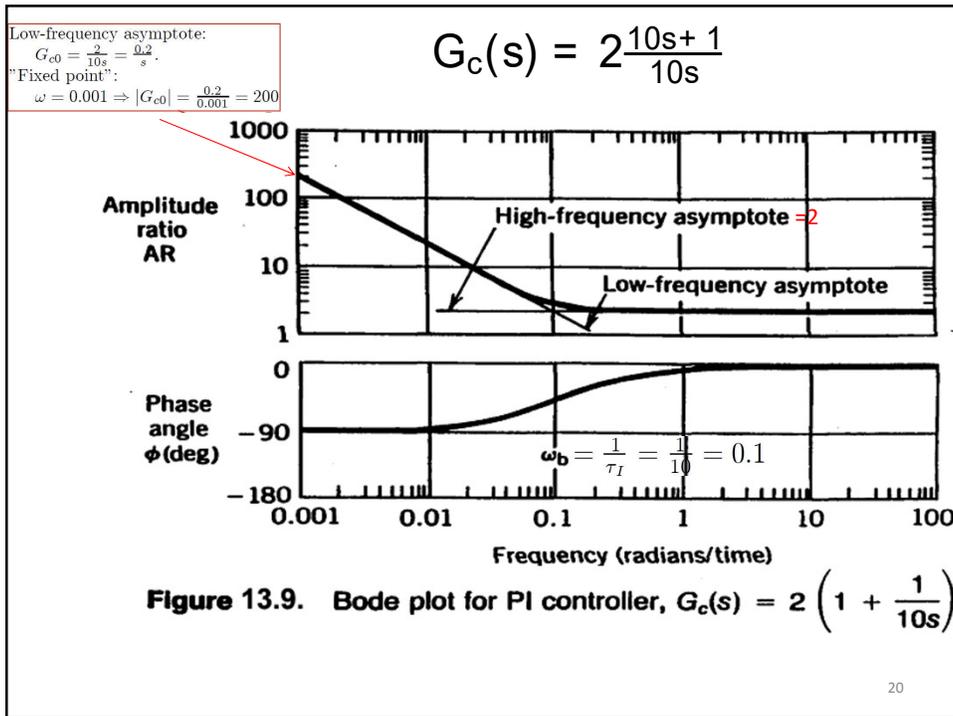
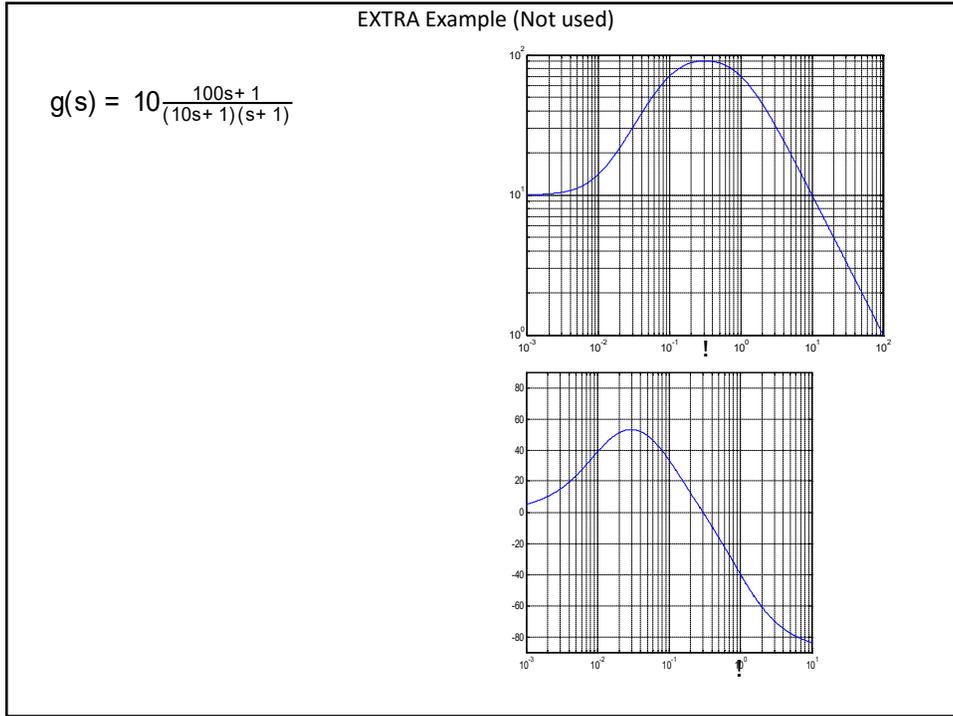
Break frequencies:
 $\omega = 1/100=0.01$ (pole), $1/20=0.05$ (zero), $1/2=0.5$ (pole)

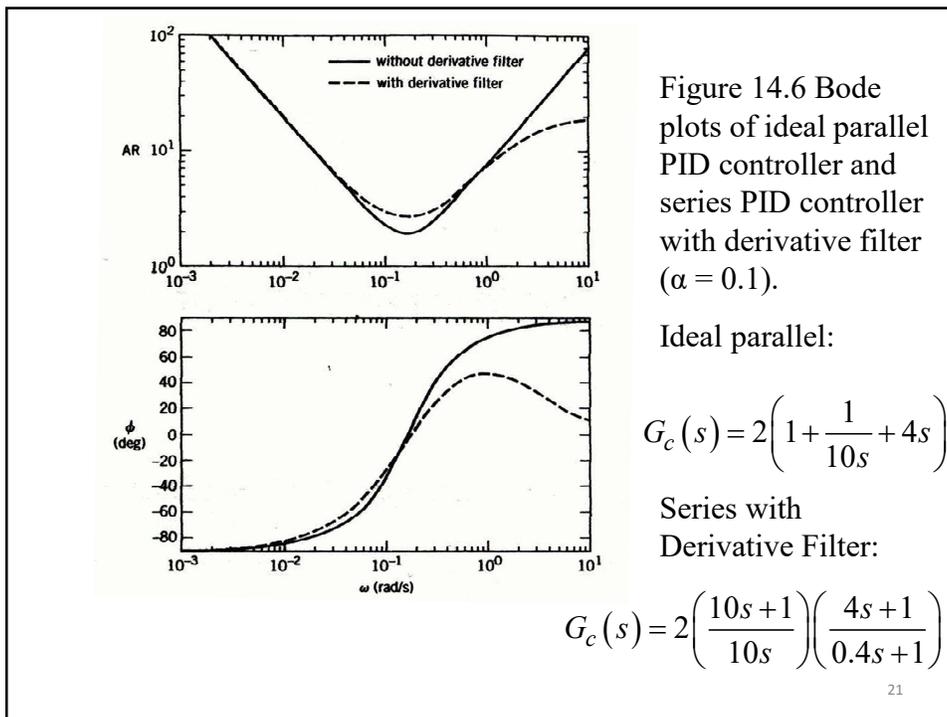
First break frequency (at 0.01) is a pole:
 Slope changes by -1 to -2 (log-log)
 \Rightarrow gain drops by factor 100 when ω increases by factor 10
 Phase drops by -90° to -180°
 Asymptote = $\frac{1}{100(j\omega)^2} = -\frac{1}{100\omega^2}$

Next break frequency (at 0.05) is a zero:
 Slope changes by +1 to -1 (log-log)
 Phase increases by $+90^\circ$ to -90°
 Asymptote = $\frac{20}{100j\omega} = -\frac{1}{5\omega}j$

Final break frequency (at 0.5) is a pole:
 Slope changes by -1 to -2 (log-log)
 Phase drops by -90° to -180°
 Asymptote = $\frac{1}{10(j\omega)^2} = -\frac{1}{10\omega^2}$



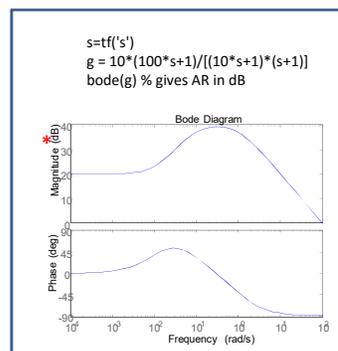




Electrical engineers (and Matlab) use decibel for gain

- $|G| \text{ [dB]} = 20 \log_{10} |G|$

$ G $	$ G \text{ [dB]}$
0.1	-20 dB
1	0 dB
2	6 dB
10	20 dB
100	40 dB
1000	60 dB



*To change magnitude from dB to abs: Right click + properties + units

Other way: $|G| = 10^{|G|(\text{dB})/20}$

GM=2 is same as GM = 6dB

CLOSED-LOOP STABILITY

- $L = g_c g_m$ = loop transfer function with negative feedback
- Bode's stability condition: $|L(\omega_{180})| < 1$
 - Limitations
 - Open-loop stable ($L(s)$ stable)
 - Phase of L crosses -180° only once

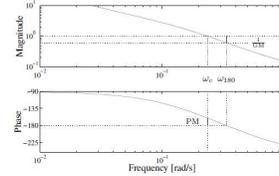
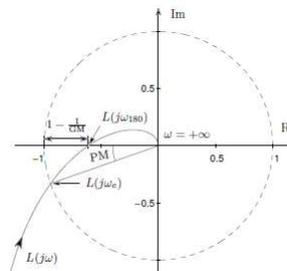


Figure 2.12: Typical Bode plot of $L(j\omega)$ with PM and GM indicated

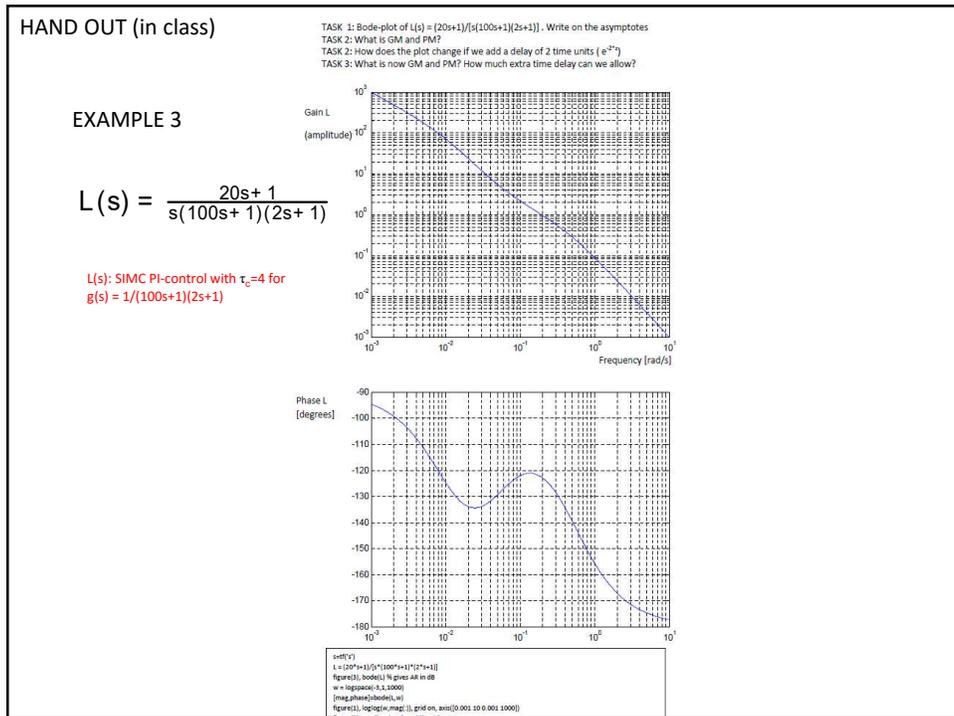
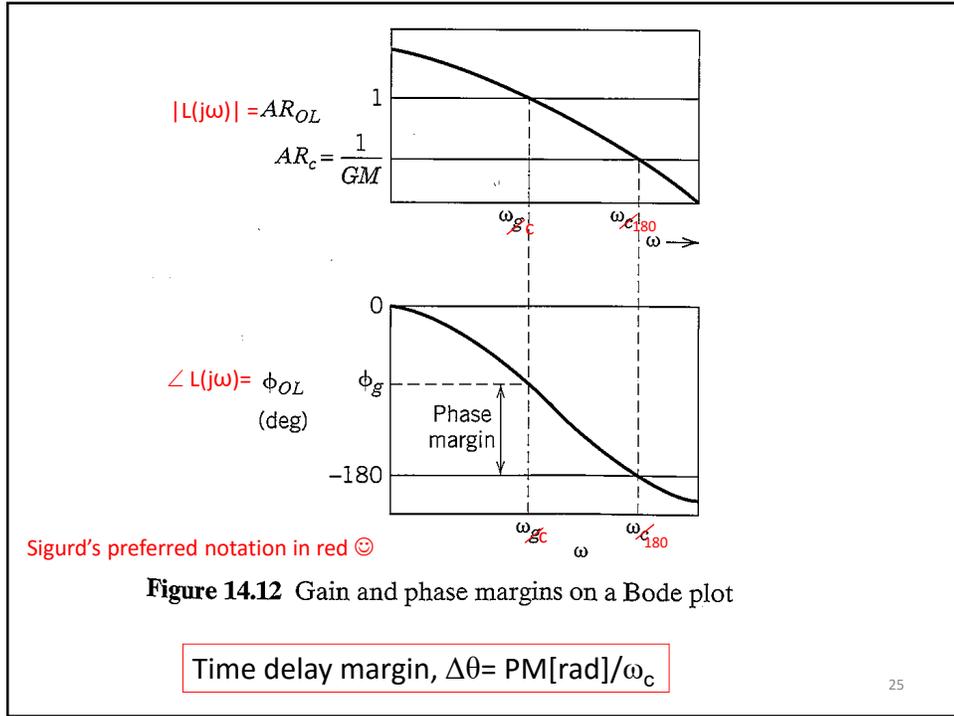
- More general: Nyquist stability condition:

Locus of $L(j\omega)$ should encircle the (-1) -point P times in the anti-clockwise direction (where P = no. of unstable poles in L).



Stable plant ($P=0$): Closed-loop stable if L has no encirclements of -1 (=Bode's stability condition)

- Example 1. P-control of delay process. For what K_c is system stable?
- Example 2. I-control of delay process. For what K_I is system stable? & compare with SIMC for delay process



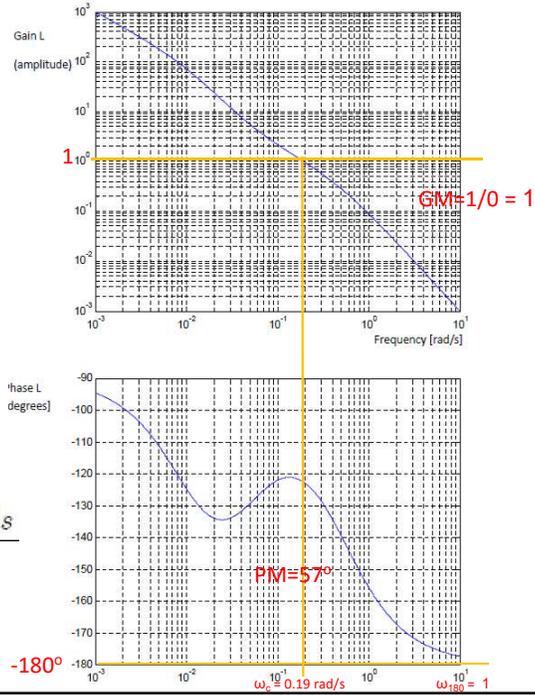
SOLUTION

$$L(s) = \frac{20s+1}{s(100s+1)(2s+1)}$$

L(s): SIMC PI-control with $\tau_c=4$ for $g(s) = 1/(100s+1)(2s+1)$

Time delay margin

$$\Delta\theta = \frac{PM[rad]}{\omega_c[rad/s]} = \frac{1}{0.19} = 5.2s$$

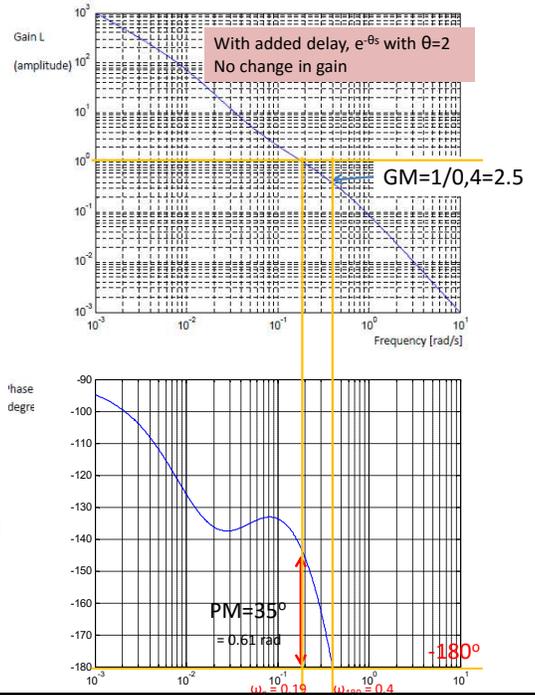


SOLUTION: ADD 2 UNITS OF DELAY

$$L = \frac{20s+1}{s(100s+1)(2s+1)} e^{-2s}$$

New time delay margin

$$\Delta\theta = \frac{PM[rad]}{\omega_c[rad/s]} = \frac{0.61}{0.19} = 3.2s$$

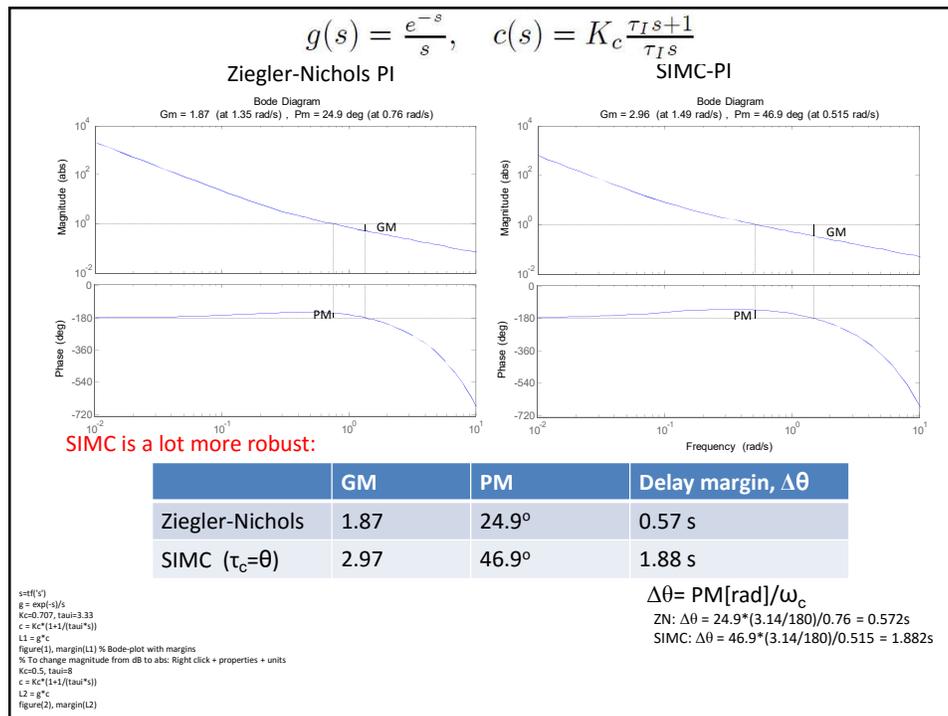


Example. PI-control of integrating process with delay

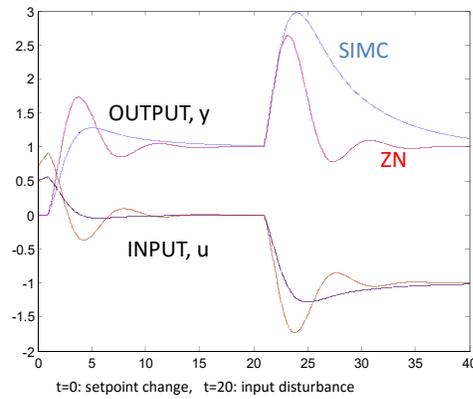
- $g(s) = k'e^{-\theta s}/s$
 - Derive: $P_u = 4\theta$ and $K_u = (\pi/2)/(k'\theta)$
- PI-controller, $c(s) = K_c (1+1/(\tau_I s))$

	K_c	τ_I
Ziegler-Nichols	$0.45K_u = 0.707/(k'\theta)$	$P_u/1.2=3.33\theta$
SIMC ($\tau_c=0$)	$0.5/(k'\theta)$	8θ

Task: Compare Bode-plot ($L=gc$), robustness and simulations (use $k'=1, \theta=1$).



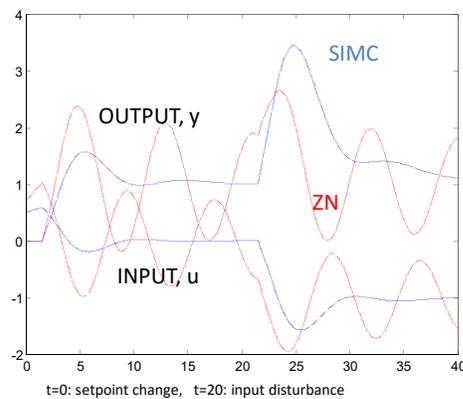
Closed-loop response: PI-control of $g(s) = \frac{e^{-s}}{s}$



Conclusion: Ziegler-Nichols (ZN) responds faster to the input disturbance, but is much less robust.

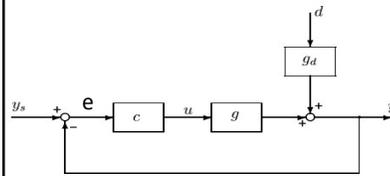
- ZN goes unstable if we increase delay from 1s to 1.57s.
- SIMC goes unstable if we increase delay from 1s to 2.88s.

INCREASE DELAY: $g(s) = \frac{e^{-1.5s}}{s}$



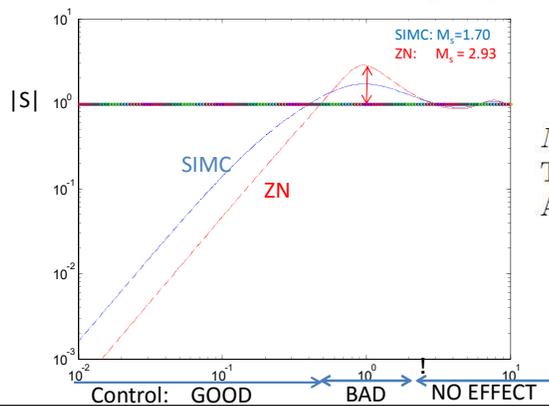
ZN is almost unstable when the delay is increased from 1s to 1.5s.
 SIMC does not change very much

Closed-loop frequency response



No control ($c = 0$): $e_{OL} = y_s - y = y_s - g_d d$
 With control: $e = y_s - y = S y_s - S g_d d = S e_{OL}$
 $S = \frac{1}{1+L}$ - sensitivity function = effect of control
 $L = gc$ - loop transfer function

Low ω where $|S| < 1$: Control reduces error
 Intermediate ω where $|S| > 1$: Control increases error
 High ω where $S = 1$ ($L \rightarrow 0$): Control has no effect



$M_s = \text{peak of } |S|$
 Typical requirement: $M_s < 2$
 At stability limit: $M_s \rightarrow \infty$

```
w = logspace(-2,1,1000);
[mag1,phase]=bode(1/(1+L1),w);
[mag2,phase]=bode(1/(1+L2),w);
figure(1), loglog(w,mag1(),red,'w',mag2(),blue,'w1','-')
axis([0.01,10,0,0.001,10])
```

Example Ziegler Nichols

Task: Find ZN-settings for integrating+ delay process

- First need to find P_u and K_u