

Multivariable control

- I. Single-loop control (decentralized)
- II. Decoupling (similar to feedforward)
- III. Model predictive control (MPC)

I. Multivariable control using single loops

- Interactions
- Choice of pairings (RGA)

Example (Exercise 12): Shower (mix hot and cold water)

- $u = (q_h \ q_c)$
- $y = (T \ q)$

What pairing?

Problem 3: Water mixer

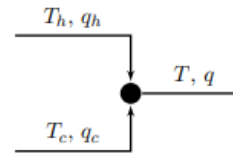


Figure 1: Mixer system

Consider the process of mixing hot and cold water, as shown in Figure 1. The process has inputs $u_1 = \Delta q_h$ [ℓ/s], $u_2 = \Delta q_c$ [ℓ/s], and outputs $y_1 = \Delta T$ [$^{\circ}C$], $y_2 = \Delta q$ [ℓ/s].

The control objective is to have a mixing temperature $T = 40^{\circ}C$ and a total flow leaving the mixer of $q = 1 \ell/s$. At the nominal operating point we have $T_c = 30^{\circ}C$ and $T_h = 60^{\circ}C$.

1. Formulate the energy and mass balances. The dynamics of this process are very fast; so, a steady-state model is sufficient to get T and q .
2. Linearize the model and show that the linear model can be written $y = Gu$, where:

$$G = \begin{bmatrix} k_1 & k_2 \\ 1 & 1 \end{bmatrix}$$

$$\text{with: } k_1 = (T_h^* - T^*)/q^* \quad k_2 = (T_c^* - T^*)/q^* \quad u = [u_1 \quad u_2]^T$$

The symbol * denotes the steady state value.

3. What are the steady state values for q_c and q_h ?
4. Find the gain matrix G at the nominal operating point.
5. Based on G , which stream (q_h or q_c) would you use to control the temperature (T)? Explain briefly.

In Exercise 13, you will find out if your intuition was right.

Multivariable process

Distillation column

“Increasing reflux L from 1.0 to 1.1 changes y_D from 0.95 to 0.97, and x_B from 0.02 to 0.03”

“Increasing boilup V from 1.5 to 1.6 changes y_D from 0.95 to 0.94, and x_B from 0.02 to 0.01”

Steady-State Gain Matrix

$$\begin{pmatrix} \Delta Y_D \\ \Delta x_B \end{pmatrix} = \mathbf{G}(0) \begin{pmatrix} \Delta L \\ \Delta V \end{pmatrix}$$

$$\mathbf{G}(0) = \begin{bmatrix} g_{11} & g_{12}(0) \\ g_{21} & g_{22}(0) \end{bmatrix} = \begin{bmatrix} \frac{0.97 - 0.95}{1.1 - 1.0} & \frac{0.94 - 0.95}{1.6 - 1.5} \\ \frac{0.03 - 0.02}{1.1 - 1.0} & \frac{0.01 - 0.02}{1.6 - 1.5} \end{bmatrix} = \begin{bmatrix} 0.2 & -0.1 \\ 0.1 & -0.1 \end{bmatrix}$$

Effect of input 1 (ΔL) on output 2 (Δx_B)

Can also include dynamics :

$$\mathbf{G}(s) = \begin{bmatrix} \frac{0.2}{1 + 50s} & -\frac{0.1}{1 + 50s} \\ \frac{0.1}{1 + 40s} & -\frac{0.1}{1 + 40s} \end{bmatrix} \begin{matrix} \rightarrow \Delta y_D \\ \rightarrow \Delta x_B \end{matrix}$$

(Time constant 50 min for y_D)

(time constant 40 min for x_B)

Analysis of Multivariable processes

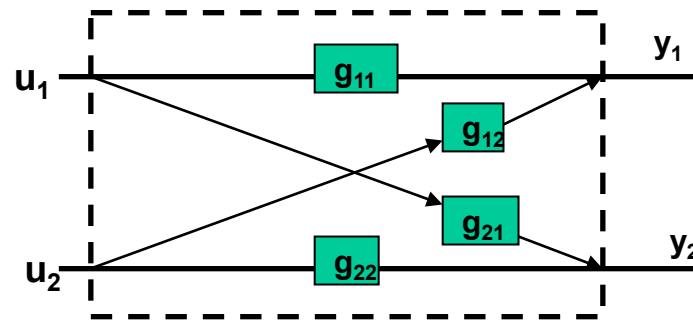
Process Model 2x2

"Open- loop"

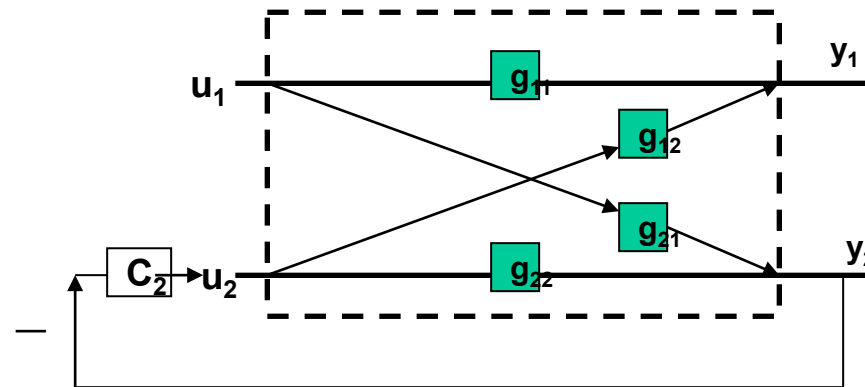
$$y_1(s) = g_{11}(s)u_1(s) + g_{12}(s)u_2(s) \quad (1)$$

$$y_2(s) = g_{21}(s)u_1(s) + g_{22}(s)u_2(s) \quad (2)$$

INTERACTIONS: Caused by nonzero offdiagonal elements (g_{12} and/or g_{21})



RGA: Consider effect of u_1 on y_1



1) “Open-loop” ($C_2 = 0$):

$$y_1 = g_{11}(s) \cdot u_1$$

2) “Closed-loop” (close loop 2, $C_2 \neq 0$):

$$y_1 = \left(g_{11}(s) - \frac{g_{12}g_{21} \cdot C_2}{1 + g_{22} \cdot C_2} \right) u_1$$

Derivation.

Close loop 2: $u_2 = -c_2(y_2 - y_{2s})$

Here: $y_2 = g_{21}u_1 + g_{22}u_2$ and assume $y_{2s} = 0$:

$$\Rightarrow u_2 = -c_2(g_{21}u_1 + g_{22}u_2) \Rightarrow u_2 = \frac{-c_2g_{21}}{1 + g_{22}c_2} u_1$$

Effect of u_1 on y_1 with loop 2 closed is then:

$$y_1 = g_{11}u_1 + g_{12}u_2 = \underbrace{g_{11} \left(1 - \frac{g_{12}g_{21}c_2}{1 + g_{22}c_2} \right)}_{\hat{g}_{11}} u_1$$

Change caused by “interactions”

Limiting Case $C_2 \rightarrow \infty$ (perfect control of y_2) *

$$y_1 = \left(g_{11}(s) - \frac{g_{12} g_{21}}{g_{22}} \right) u_1 = g_{11}(1/\lambda_{11}) \cdot u_1$$

How much has “gain” from u_1 to y_1 changed by closing loop 2 with perfect control?

$$\text{Relative Gain} = \frac{(y_1/u_1)_{OL}}{(y_1/u_1)_{CL}} = \frac{g_{11}}{g_{11} - \frac{g_{12} g_{21}}{g_{22}}} = \frac{1}{1 - \frac{g_{12} g_{21}}{g_{11} g_{22}}} \stackrel{\text{def}}{=} \lambda_{11}^{\text{RGA}}$$

The relative Gain Array (RGA) is the matrix formed by considering all the relative gains

$$\text{RGA} = \Lambda = \begin{bmatrix} \lambda_{11} & \lambda_{12} \\ \lambda_{21} & \lambda_{22} \end{bmatrix} = \begin{bmatrix} \frac{(y_1/u_1)_{OL}}{(y_1/u_1)_{CL}} & \frac{(y_1/u_2)_{OL}}{(y_1/u_2)_{CL}} \\ \frac{(y_2/u_1)_{OL}}{(y_2/u_1)_{CL}} & \frac{(y_2/u_2)_{OL}}{(y_2/u_2)_{CL}} \end{bmatrix}$$

Example from before

$$\mathbf{G} = \begin{bmatrix} 0.2 & -0.1 \\ 0.1 & -0.1 \end{bmatrix}, \quad \lambda_{11} = \frac{1}{1 - \underbrace{\frac{0.1 \ 0.1}{0.2 \ 0.1}}_{0.5}} = 2$$

$$\mathbf{RGA} = \begin{bmatrix} 2 & -1 \\ -1 & 2 \end{bmatrix}$$

Only acceptable pairings :

$$u_1 \leftrightarrow y_1$$

$$u_2 \leftrightarrow y_2$$

Not recommended :

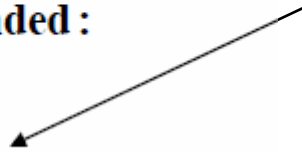
$$u_1 \leftrightarrow y_2$$

$$u_2 \leftrightarrow y_1$$

With integral action :

Negative RGA \Rightarrow individual

loop unstable OR overall system unstable
when individual loops saturates



Use of RGA:

(1) Interactions

- RGA-element (λ) > 1: Smaller gain by closing other loops (“fighting loops” gives slower control)
- RGA-element (λ) < 1: Larger gain by closing other loops (can be dangerous)
- RGA-element (λ) negative: Gain reversal by closing other loops (Oops!)

Rule 1. Avoid pairing on negative steady-state relative gain – otherwise you get instability if one of the loops become inactive (e.g. because of saturation)

Rule 2. Choose pairings corresponding to RGA-elements close to 1

Traditional: Consider Steady-state

Better (improved Rule 2): Consider frequency corresponding to closed-loop time constant

Property of RGA:

- ✚ Columns and rows always sum to 1
- ✚ RGA independent of scaling (units) for u and y.

RGA for general case:

$$[RGA]_{ij} = (g_{ij})_{OL} / (g_{ij})_{CL} = [G]_{ij} [G^{-1}]_{ji}$$

= element-by-element multiplication of G and G^{-1T} .

Matlab: `RGA = G.*pinv(G) .'`

Example

$$G = [5 \ 10 \ 1; 20 \ -10 \ 0; 18 \ 0 \ 2]$$

G =

5 10 1

20 -10 0

18 0 2

>> rga=G.*pinv(G)'

rga =

0.3125 1.2500 -0.5625

1.2500 -0.2500 0

-0.5625 0 1.5625

Conclusion: of the 6 possible pairings only one has positive RGA's

Example 3x3 process: 6 possible pairing options

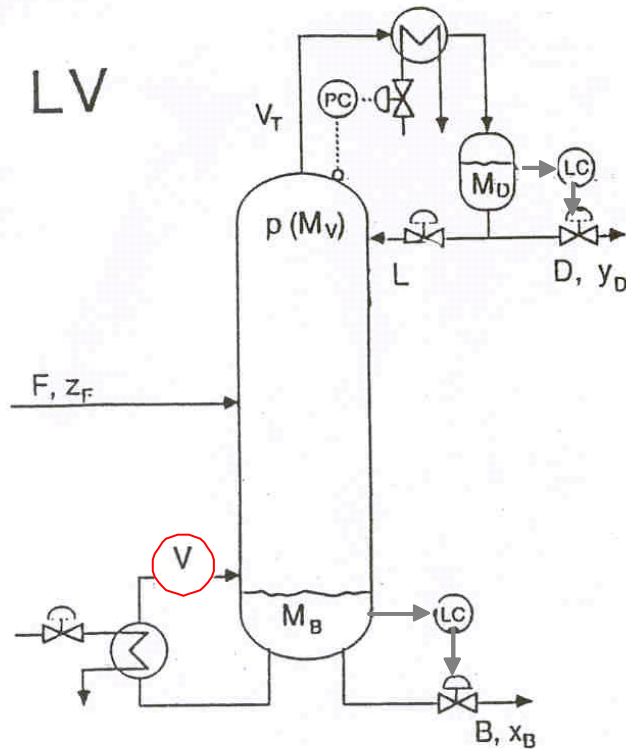
$$G = \begin{pmatrix} 16.8 & 30.5 & 4.30 \\ -16.7 & 31.0 & -1.41 \\ 1.27 & 54.1 & 5.40 \end{pmatrix}, \quad RGA(G) = \begin{pmatrix} 1.50 & 0.99 & -1.48 \\ -0.41 & 0.97 & 0.45 \\ -0.08 & -0.95 & 2.03 \end{pmatrix}$$

Only diagonal pairings give positive steady-state RGA's!

Distillation

$$y = \begin{pmatrix} y_D \\ x_B \end{pmatrix}, \quad u = \begin{pmatrix} L \\ V \end{pmatrix}$$

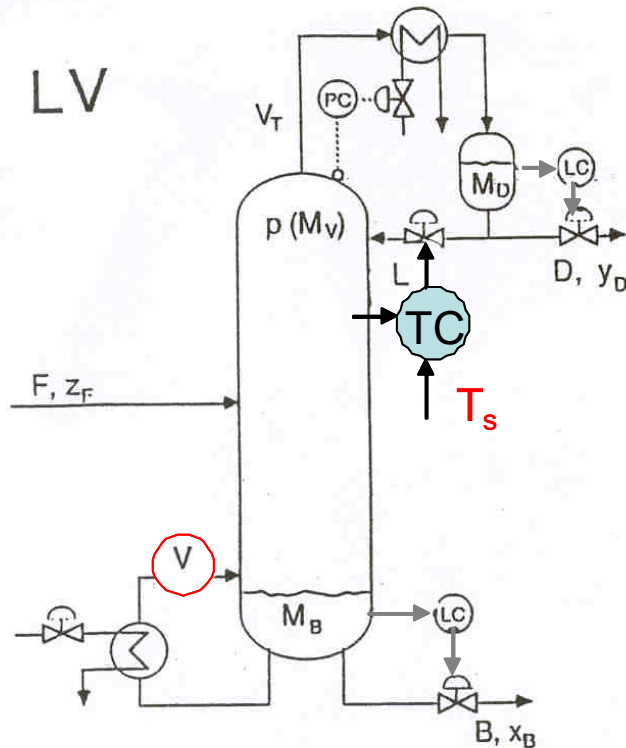
$$G(0) = \begin{pmatrix} 87.8 & -86.4 \\ 108.2 & -109.6 \end{pmatrix}, \quad RGA(0) = \begin{pmatrix} 35 & -34 \\ -34 & 35 \end{pmatrix}$$



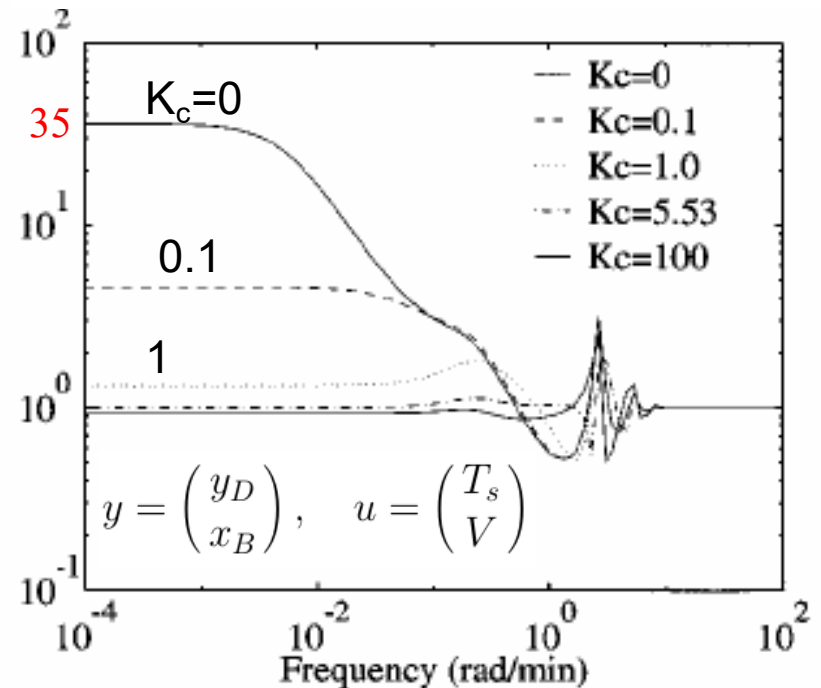
Distillation

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Can break interactions with cascade:
Frequency-dependent RGA with TC



Sometimes useful: Iterative RGA

- For large processes, lots of pairing alternatives
- RGA evaluated iteratively is helpful for quick screening

$$\text{RGA}(G) = \Lambda(G) = G \times (G^{-1})^T$$

$$\Lambda^2(G) = \Lambda(\Lambda(G))$$

$$\Lambda^\infty = \lim_{k \rightarrow \infty} \Lambda^k(G)$$

- Converges to “Permuted Identity” matrix (correct pairings) for generalized diagonally dominant processes.
- Can converge to incorrect pairings, when no alternatives are dominant.
- Usually converges in 5-6 iterations

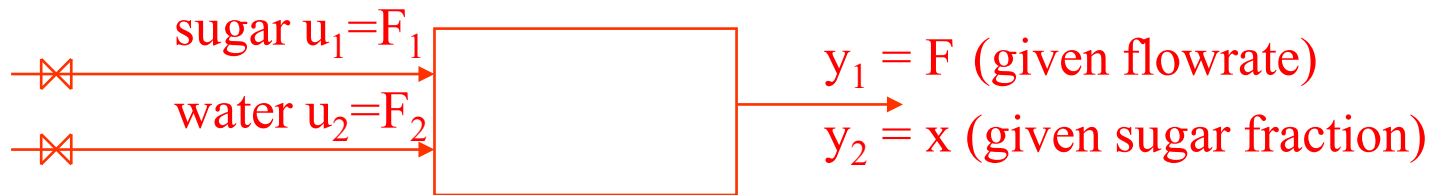
Example of Iterative RGA

$$G = \begin{bmatrix} 1 & 2 \\ -1 & 1 \end{bmatrix}$$

$$\Lambda = \begin{bmatrix} 0.33 & 0.67 \\ 0.67 & 0.33 \end{bmatrix} \quad \Lambda^2 = \begin{bmatrix} -0.33 & 1.33 \\ 1.33 & -0.33 \end{bmatrix}$$
$$\Lambda^3 = \begin{bmatrix} -0.07 & 1.07 \\ 1.07 & -0.07 \end{bmatrix} \quad \Lambda^4 = \begin{bmatrix} 0.00 & 1.00 \\ 1.00 & 0.00 \end{bmatrix}$$

Correct pairing

Exercise. Blending process



- Mass balances (no dynamics)
 - Total: $F_1 + F_2 = F$
 - Sugar: $F_1 = x F$
- (a) Linearize balances and introduce: $u_1=dF_1$, $u_2=dF_2$, $y_1=dF_1$, $y_2=dx$,
- (b) Obtain gain matrix G ($y = G u$)
- (c) Nominal values are $x=0.2$ [kg/kg] and $F=2$ [kg/s]. Find G
- (d) Compute RGA and suggest pairings
- (e) Does the pairing choice agree with “common sense”?

Solution.

(a) The balances “mass in = mass out” for total mass and sugar mass are

$$F_1 + F_2 = F; \quad F_1 = xF$$

Note that the mixing process itself has no dynamics. Linearization yields

$$dF_1 + dF_2 = dF : \quad dF_1 = x^* dF + F^* dx$$

With $u_1 = dF_1$, $u_2 = dF_2$, $y_1 = dF$ and $y_2 = dx$ we then get the model

$$\begin{aligned} y_1 &= u_1 + u_2 \\ y_2 &= \frac{1-x^*}{F^*} u_1 - \frac{x^*}{F^*} u_2 \end{aligned}$$

where $x^* = 0.2$ is the nominal steady-state sugar fraction and $F^* = 2$ kg/s is the nominal amount.

(b,c) The transfer matrix then becomes

$$G(s) = \begin{pmatrix} 1 & 1 \\ \frac{1-x^*}{F^*} & -\frac{x^*}{F^*} \end{pmatrix} = \begin{pmatrix} 1 & 1 \\ 0.4 & -0.1 \end{pmatrix}$$

(d) The corresponding RGA matrix is (at all frequencies)

$$\Lambda = \begin{pmatrix} x^* & 1-x^* \\ 1-x^* & x^* \end{pmatrix} = \begin{pmatrix} 0.2 & 0.8 \\ 0.8 & 0.2 \end{pmatrix}$$

For decentralized control, it then follows from pairing rule 1 (“prefer pairing on RGA elements close to 1”) that we should pair on the off-diagonal elements; that is, use u_1 to control y_2 and use u_2 to control y_1 .

(e) This corresponds to using the largest stream (water, u_2) to control the amount ($y_1 = F$), which is reasonable from a physical point of view. Also note that the RGA-elements are always between 0 and 1 for this process, and the RGA-elements are all 0.5, corresponding to “switching” the pairings, when $x^* = 0.5$, which is when the two feed streams are equal.

Decentralized control tuning

- Independent design
 - Use when small interactions (RGA close to I)
- Sequential design (similar to cascade)
 - Start with fast loop
 - NOTE: If close on negative RGA, system will go unstable if fast (inner) loop saturates
 - Sequential vs. independent design
 - + Generally better performance, but
 - - outer loop gets slow, and
 - - loops depend on each other

Summary

Single-loop control = Decentralized control

Use for: Noninteracting process

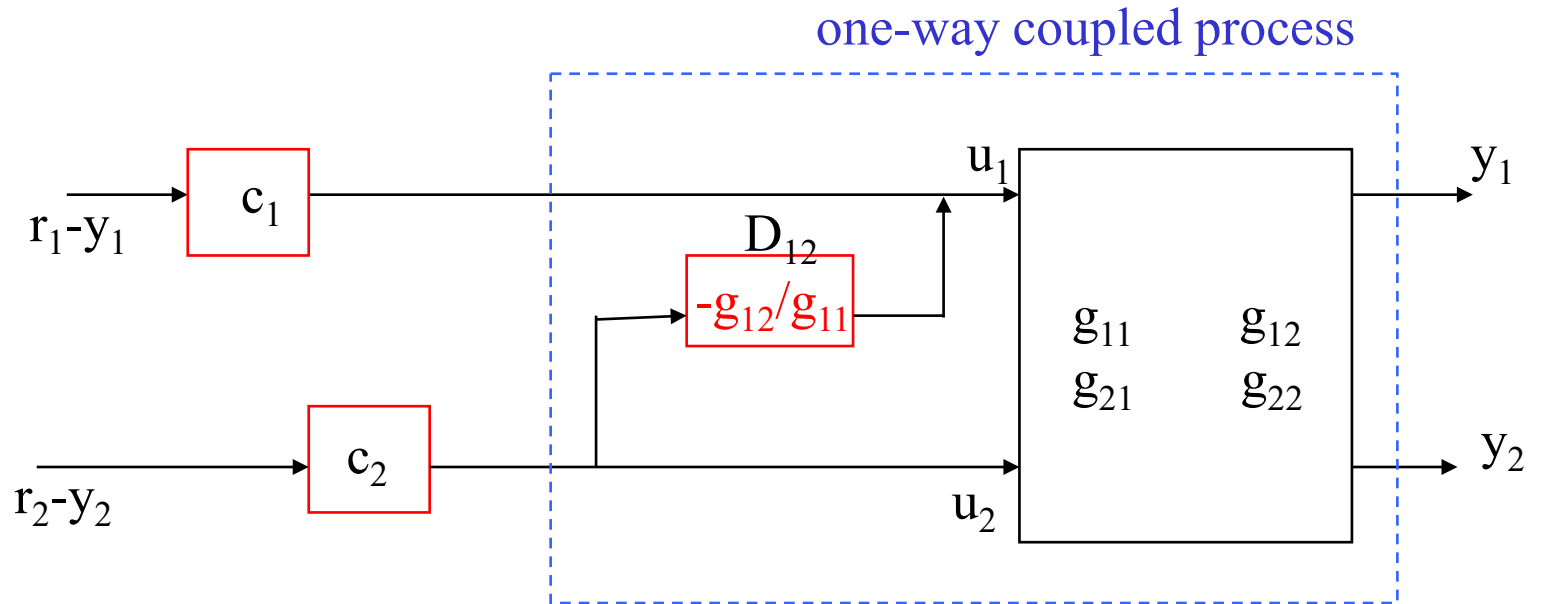
- + Tuning may be done on-line
- + No or minimal model requirements
- + Easy to fix and change
- Need to determine pairing
- Performance loss compared to multivariable control

Multivariable control

1. Single-loop control (decentralized)
- 2. Decoupling (similar to feedforward)**
3. Model predictive control (MPC)

II. Decoupling

a) One-way Decoupling (improved control of y_1)



DERIVATION

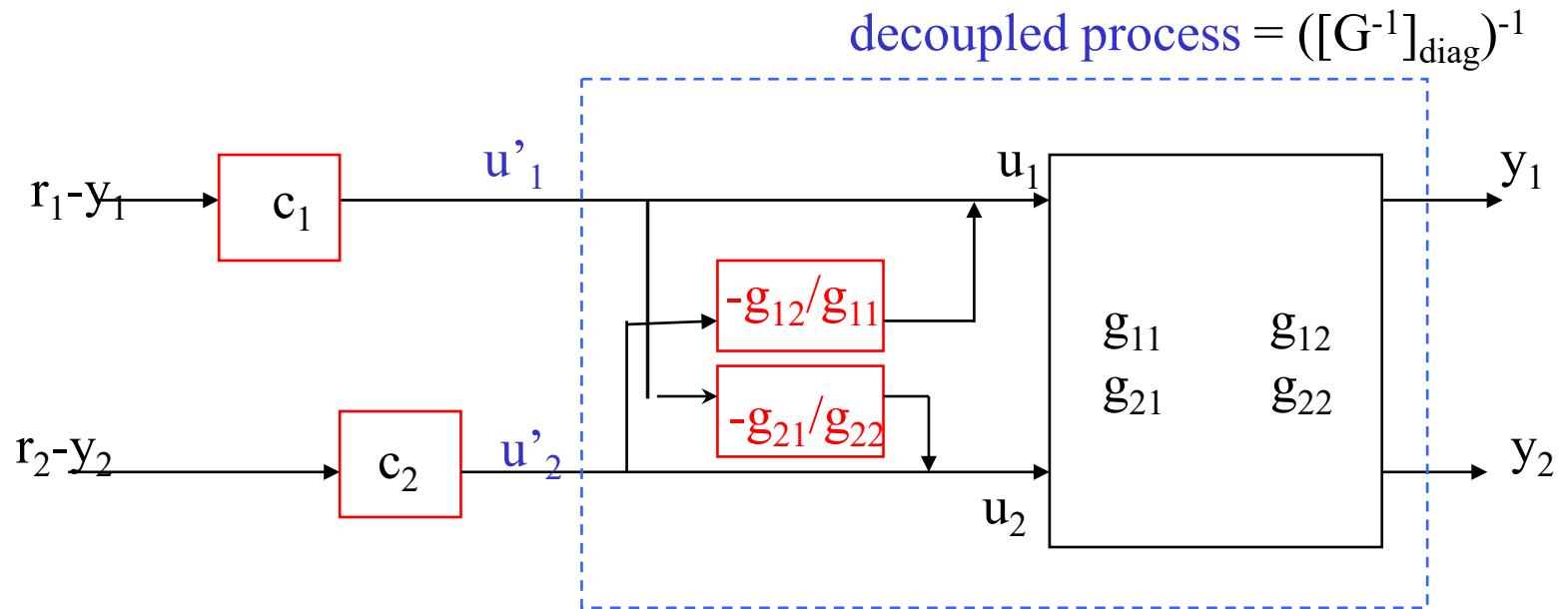
Process: $y_1 = g_{11} u_1 + g_{12} u_2 \quad (1)$

$y_2 = g_{21} u_1 + g_{22} u_2 \quad (2)$

Consider u_2 as disturbance for control of y_1 .

Think «feedforward»: Adjust u_1 to make $y_1=0$. (1) gives $u_1 = - (g_{12}/g_{11}) u_2$

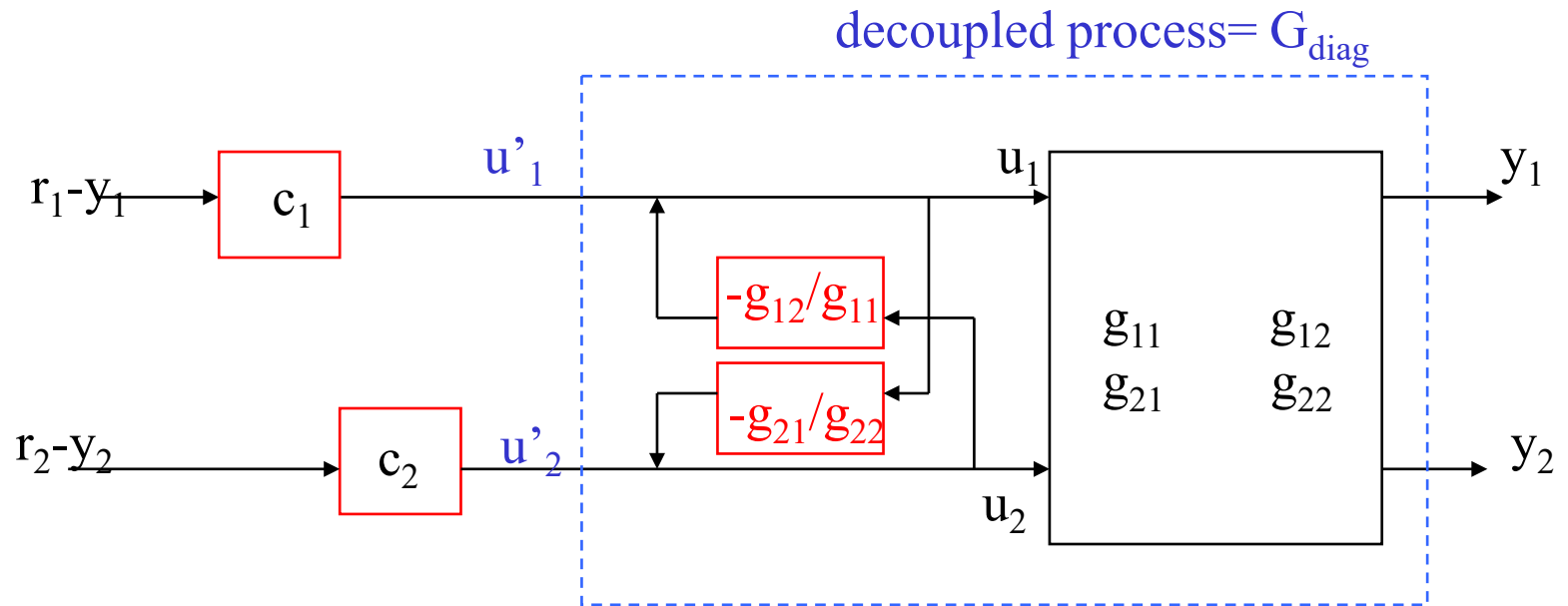
b) Two-way Decoupling: Standard implementation (Seborg)



... but note that diagonal elements of decoupled process are different from G
Problem for tuning!

Process: $y_1 = g_{11} u_1 + g_{12} u_2$
 Decoupled process: $y_1 = (g_{11} - g_{12} * g_{21} / g_{22}) u_1' + 0 * u_2'$
 Similar for y_2 .

Two-way Decoupling: «Inverted» implementation (Shinskey)



Advantages: (1) Decoupled process has same diagonal elements as G . Easy tuning!
 (2) Handles input saturation! (if u_1 and u_2 are actual inputs)

Proof (1): $y_1 = g_{11} u_1 + g_{12} u_2$, where $u_1 = u_1' - (g_{12}/g_{11})u_2$.
 Gives : $y_1 = g_{11} u_1' + 0 * u_2'$
 Similar: $y_2 = 0 * u_1' + g_{22} u_2'$

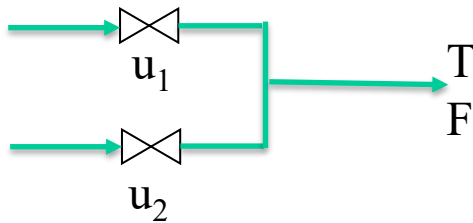
Pairing and decoupling

- To get ideal decoupling, diagonal elements should have smaller effective delay than the off-diagonal elements
- Thus, we should pair on elements with small effective delay (“pair close rule”)
- Pairing on negative steady state RGA elements is not necessarily a problem if we use decoupling
 - Because negative RGA-elements are caused by interactions, which is what we are cancelling with decoupling

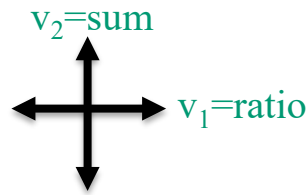
Nonlinear decoupling

- It's often easier to make nonlinear decoupler based on static model or insight

Example: Mixing of hot (u_1) and cold (u_2) water

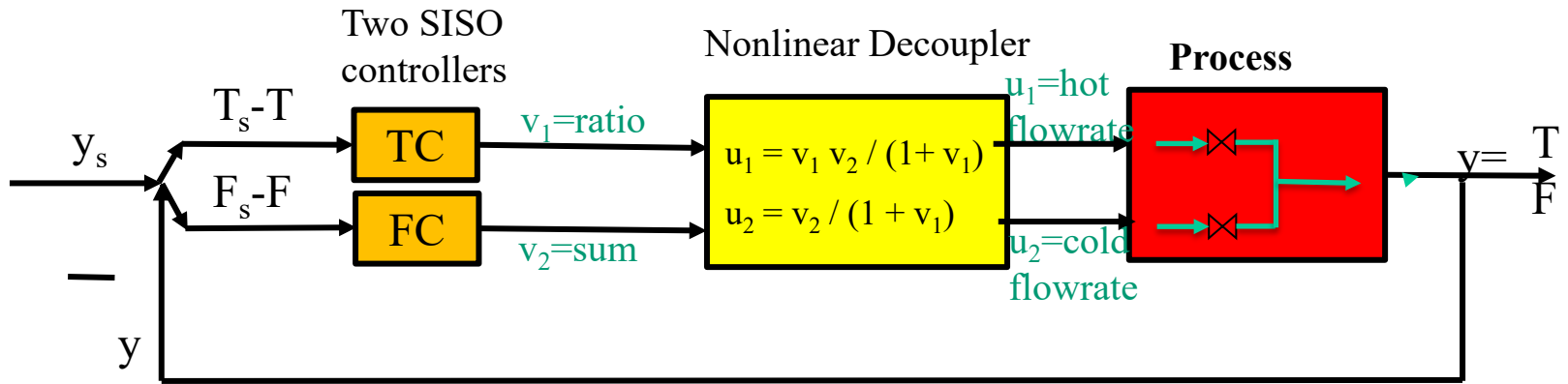


- Want to control
 - $y_1 = \text{Temperature } T$
 - $y_2 = \text{total flow } F$
- Inputs, $u = \text{flowrates}$
- May use two SISO PI-controllers
 - TC
 - FC
- Insight: Get decoupled response with transformed inputs
 - TC sets flow ratio, $v_1 = u_1/u_2$
 - FC sets flow sum, $v_2 = u_1 + u_2$
- Decoupler: Need «static calculation block» to solve for inputs



$$u_1 = v_1 v_2 / (1 + v_1)$$

$$u_2 = v_2 / (1 + v_1)$$



Pairings:

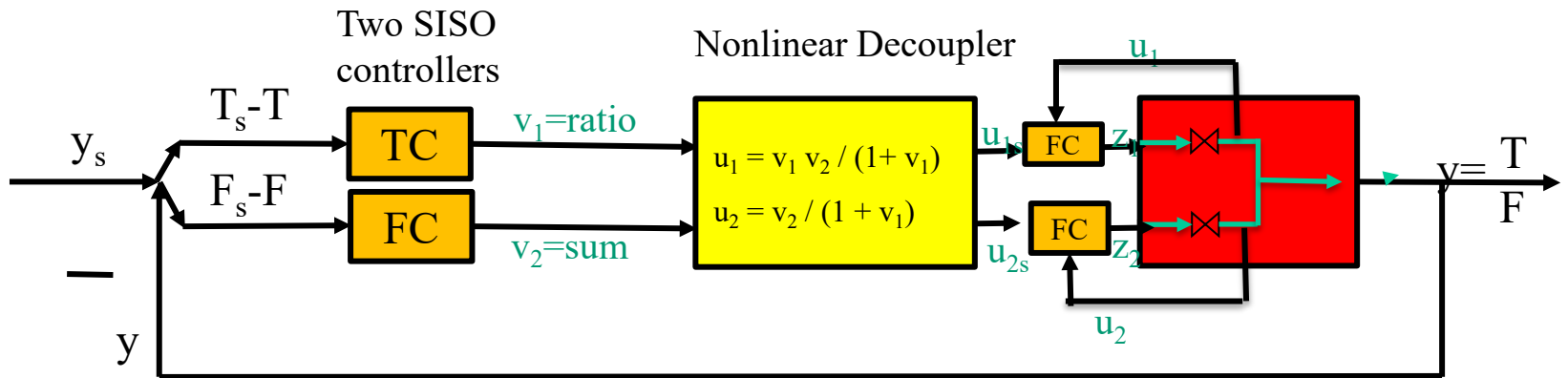
- $T - v_1$
- $F - v_2$

No interactions for setpoint change

Note:

- In practice $u = \text{valve position } (z)$
- So must add two flow controllers
 - These generate inverse by feedback

In practice must add two slave flow controllers



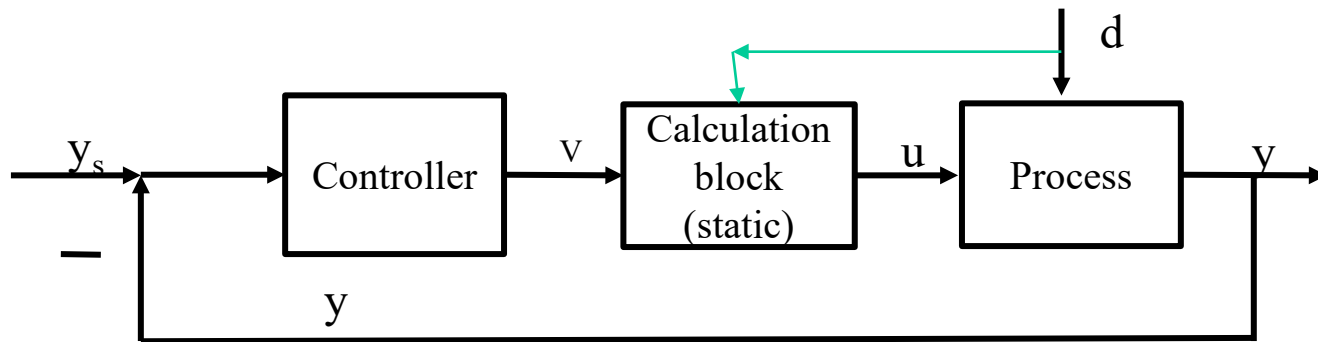
v = transformed inputs

u = flowrates

z = valve positions

General approach: Combined Nonlinear decoupling, feedforward and linearization using Transformed Inputs *

- Linear decoupling and feedforward often work poorly because of nonlinearity
- Example of nonlinear feedforward: Ratio control
- Generalization: Nonlinear calculation block



General Method: Select «transformed inputs» v as right hand side of steady state model equations, $y = \text{RHS}(d, u, \dots)$

Example: Combined nonlinear decoupling and feedforward.

Mixing of hot and cold water

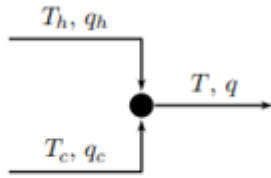
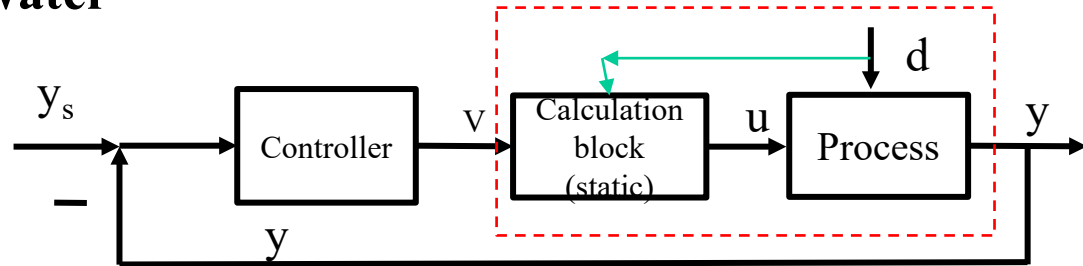


Figure 1: Mixer system



Steady-state model written as $y=f(u,d)$:

$$T = \frac{q_h T_h + q_c T_c}{q_h + q_c}$$

$$q = q_c + q_h$$

Select transformed inputs as right hand side, $v = f$

$$v_1 = \frac{q_h T_h + q_c T_c}{q_h + q_c} \quad (1) \quad \text{Generalized ratio}$$

$$v_2 = q_c + q_h \quad (2)$$

Model from v to y (red box) is then decoupled and with perfect disturbance rejection:

$$T = v_1$$

$$q = v_2$$

- Can then use two single-loop PI controllers for T and q !
 - These controllers are needed to correct for model errors and unmeasured disturbances
- Note that v_1 used to control T is a generalized ratio, but it includes also feedforward from T_c and T_h .

Implementation (calculation block) : Solve (1) and (2) with respect to $u=(q_c \ q_h)$:

Decoupler with feedforward:

$$q_h = \frac{v_2(v_1 - T_c)}{T_h - T_c}$$

$$q_c = v_2 - q_h$$

$$u = \begin{pmatrix} q_h \\ q_c \end{pmatrix}$$

$$d = \begin{pmatrix} T_h \\ T_c \end{pmatrix}$$

$$y = \begin{pmatrix} T \\ q \end{pmatrix}$$

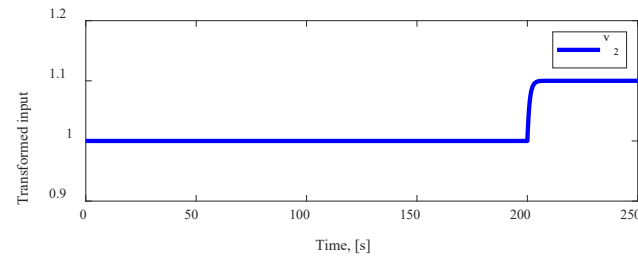
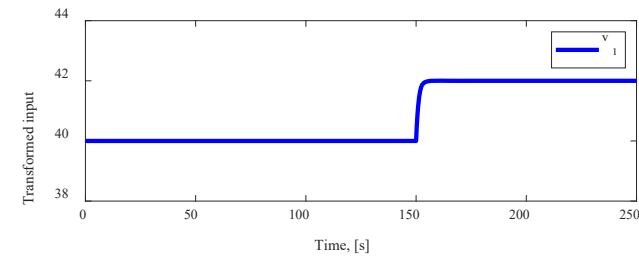
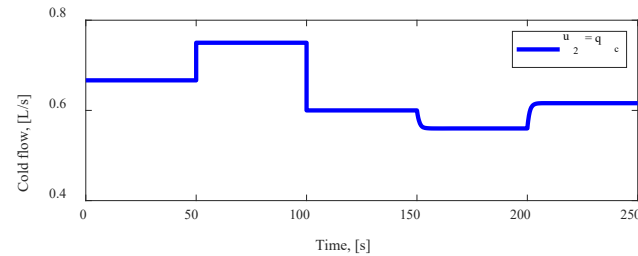
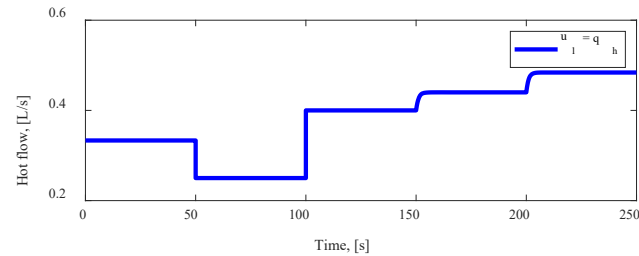
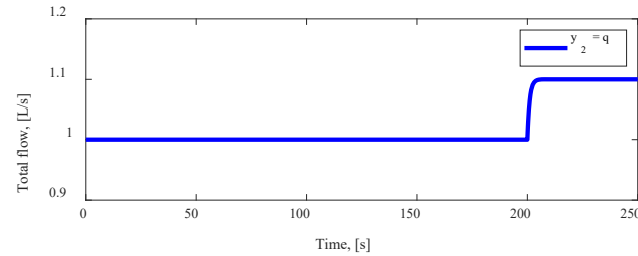
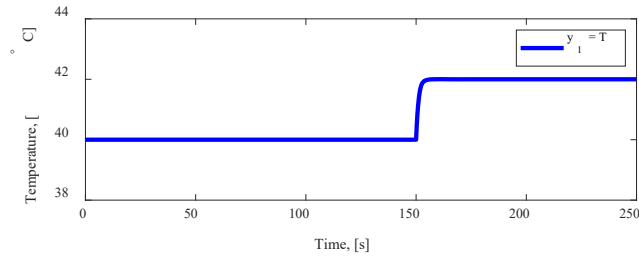
Transformed MVs for decoupling, linearization and disturbance rejection

Mixing of hot and cold water (static process)

New system: $T=v_1$ and $q=v_2$

Outer loop: Two I-controllers with $\tau_C = 1$ s

1. T_h : 60 \rightarrow 70 °C at $t = 50$ s
2. T_c : 30 \rightarrow 20 °C at $t = 100$ s
3. T_h^s : 40 \rightarrow 42 °C at $t = 150$ s
4. q^s : 1 \rightarrow 1.1 L/s at $t = 200$ s



Advanced multivariable control with explicit constraint handling = MPC

Use for: Interacting process and changes in active constraints

- + Easy handling of feedforward control
- + Easy handling of changing constraints
 - no need for logic
 - But does not always work
- Requires multivariable dynamic model
- Tuning may be difficult
- Less transparent
- “Everything goes down at the same time”

MPC = model predictive control

Multivariable control: MPC versus decoupling

- Both MPC and decoupling require a multivariable process model
- MPC is usually preferred instead of decoupling because it can also handle feedforward control, nonsquare processes (cascade, input resetting) etc.
- MPC can also handle constraints
 - Don't need to add anti-windup

Model predictive control (MPC) = “online optimal control”

The quadratic program of equations (1)-(5) is solved each control sample to find the optimal control actions.

$$\min_{\Delta u} y_{dev}^T Q_y y_{dev} + u_{dev}^T Q_u u_{dev} + \Delta u^T P \Delta u \quad (1)$$

$$u_{min} < u < u_{max} \quad (2)$$

$$\Delta u_{min} < \Delta u < \Delta u_{max} \quad (3)$$

$$y_{min} < y < y_{max} \quad (4)$$

$$y = M(y, u, d, v) \quad (5)$$

$$y_{dev} = y - y_s$$

$$u_{dev} = u - u_s$$

Discretize in time:

$$y = [y_1 \ y_2 \ \dots \ y_n]$$

$$u = [u_1 \ u_2 \ \dots \ u_k]$$

$$\Delta u = [\Delta u_1 \ \Delta u_2 \ \dots \ \Delta u_k]$$

$$\Delta u_i = u_i - u_{i-1}$$

Note: Implement only current input Δu_1

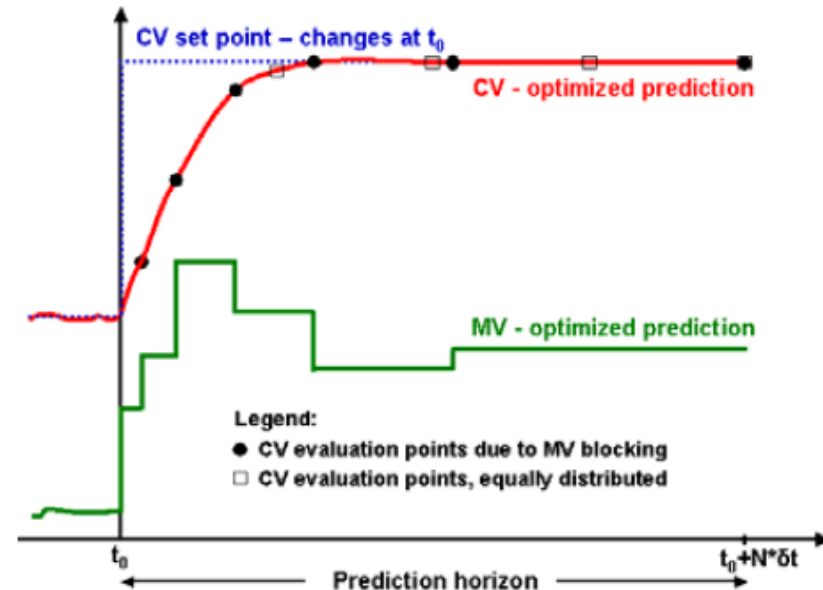


Fig. 1. MV blocking and CV evaluation

The quadratic objective function (1) penalizes CV (y) deviations from set point, MV (u) deviations from ideal values, and MV moves. The constraints are: MV high and low limits (2); MV rate of change limits (3); and CV high and low limits (4). The dynamic model (5) predicts the CV response from past and future CV and MV values as well as past DV (d) values and estimated and optionally predicted unmeasured disturbances v .

Implementation MPC project (Stig Strand, Equinor)

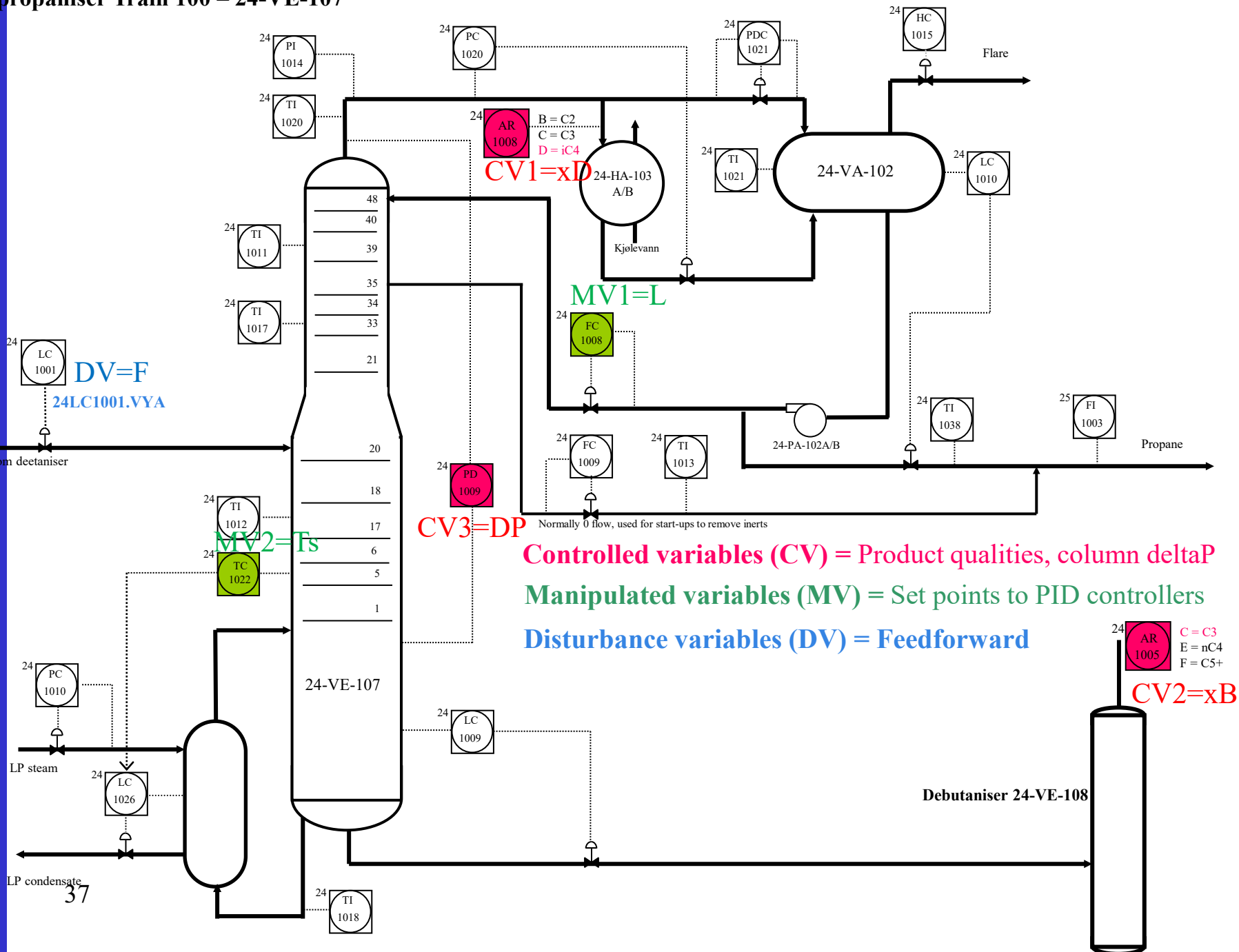
- Initial MV/CV/DV selection
- DCS* preparation (controller tuning, instrumentation, MV handles, communication logics etc)
- Control room operator pre-training and motivation
- Product quality control → Data collection (process/lab) → Inferential model (“soft sensor”)

- MV/DV step testing → dynamic models

- Model judgement/singularity analysis → remove models? change models?
- MPC pre-tuning by simulation → MPC activation – step by step and with care – challenging different constraint combinations – adjust models?
- Control room operator training
- MPC in normal operation, with at least 99% service factor

*DCS = “distributed control system” = Basic PID control layer

Bottoms from dectaniser



Controlled variables (CV) = Product qualities, column deltaP
Manipulated variables (MV) = Set points to PID controllers
Disturbance variables (DV) = Feedforward

AR 1005 C = C3
 E = nC4
 F = C5+
CV2=xB

AR 1008 B = C2
 C = C3
 D = iC4
CV1=xD

PD 1009
CV3=DP

Normally 0 flow, used for start-ups to remove inerts

FC 1008
MV1=L

TC 1022
MV2=Fs

LC 1001 **DV=F**
 24LC1001.VYA

Debutaniser 24-VE-108

Flare

Propane

24-VE-107

24-PA-102 A/B

24-HA-103 A/B
 Kjølevann

24-VA-102

TI 1018

TI 1011

TI 1017

TI 1012

TC 1022

PI 1014

TI 1020

PC 1020

TI 1021

TI 1013

FC 1009

LC 1010

TI 1038

FI 1003

PDC 1021

HC 1015

PC 1010

LC 1026

AR 1005

LP condensate

LP steam

TI 1018

TI 1011

TI 1017

TI 1012

TC 1022

PI 1014

TI 1020

PC 1020

TI 1021

TI 1013

FC 1009

LC 1010

TI 1038

FI 1003

PDC 1021

HC 1015

PC 1010

LC 1026

AR 1005

LP condensate

LP steam

Conventional control (PID)

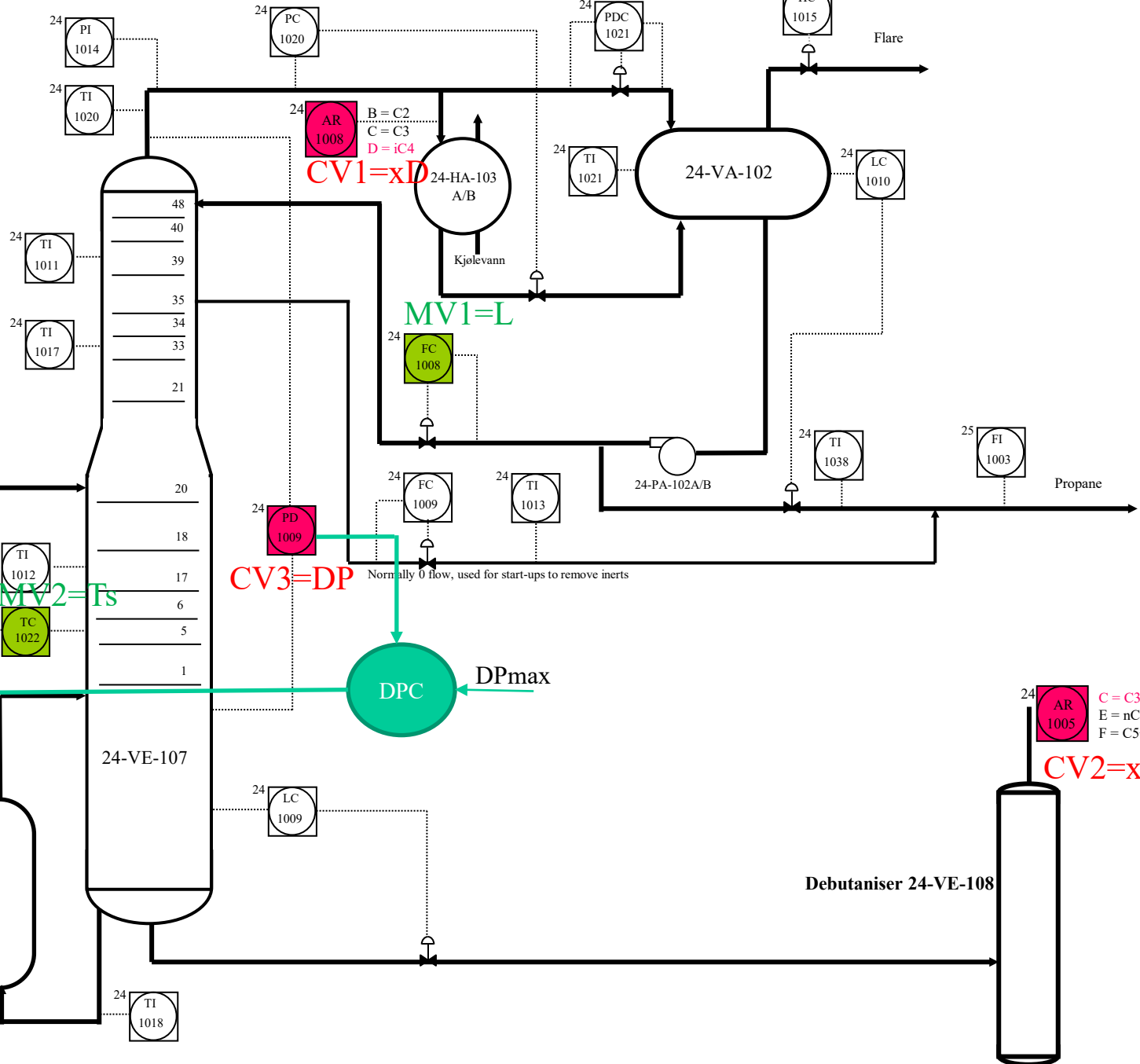
Select pairings MV-CV (obvious here: L-xD, Ts-xB)

Selector: Give up xB when we meet DP constraint

Tune 3 PI controllers

DV=F
24LC1001.VYA

Here selector on MV from TC.
Could have selector on Ts instead



C = C3
E = nC4
F = C5+
CV2=xB

Conventional control (PID)

Select pairings MV-CV (obvious here: L-xD, Ts-xB)

Selector: Give up xB when we meet DP constraint

Tune 3 PI controllers

MPC

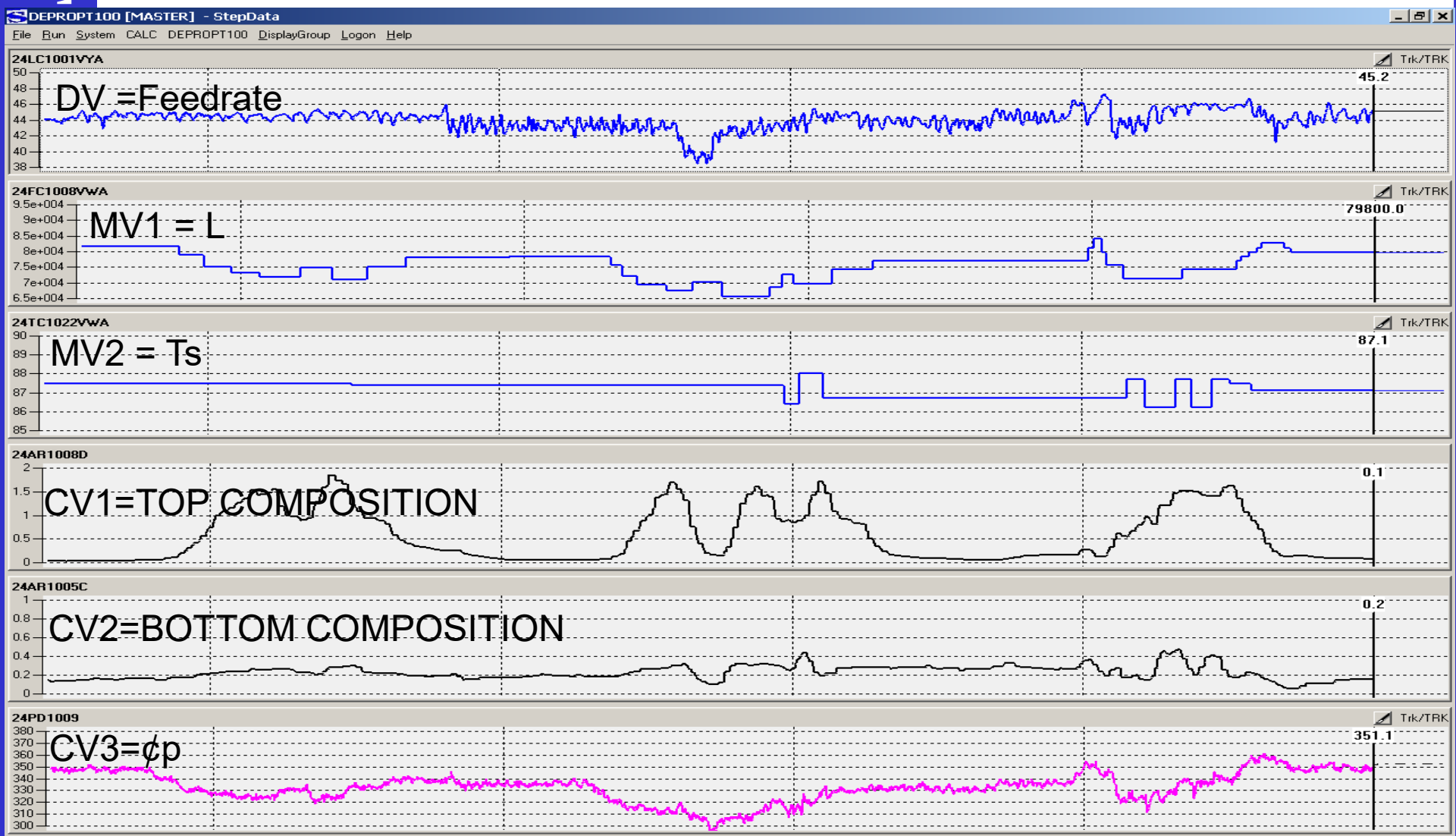
Don't need to make pairing choices

But need model

And need to tune MPC controller

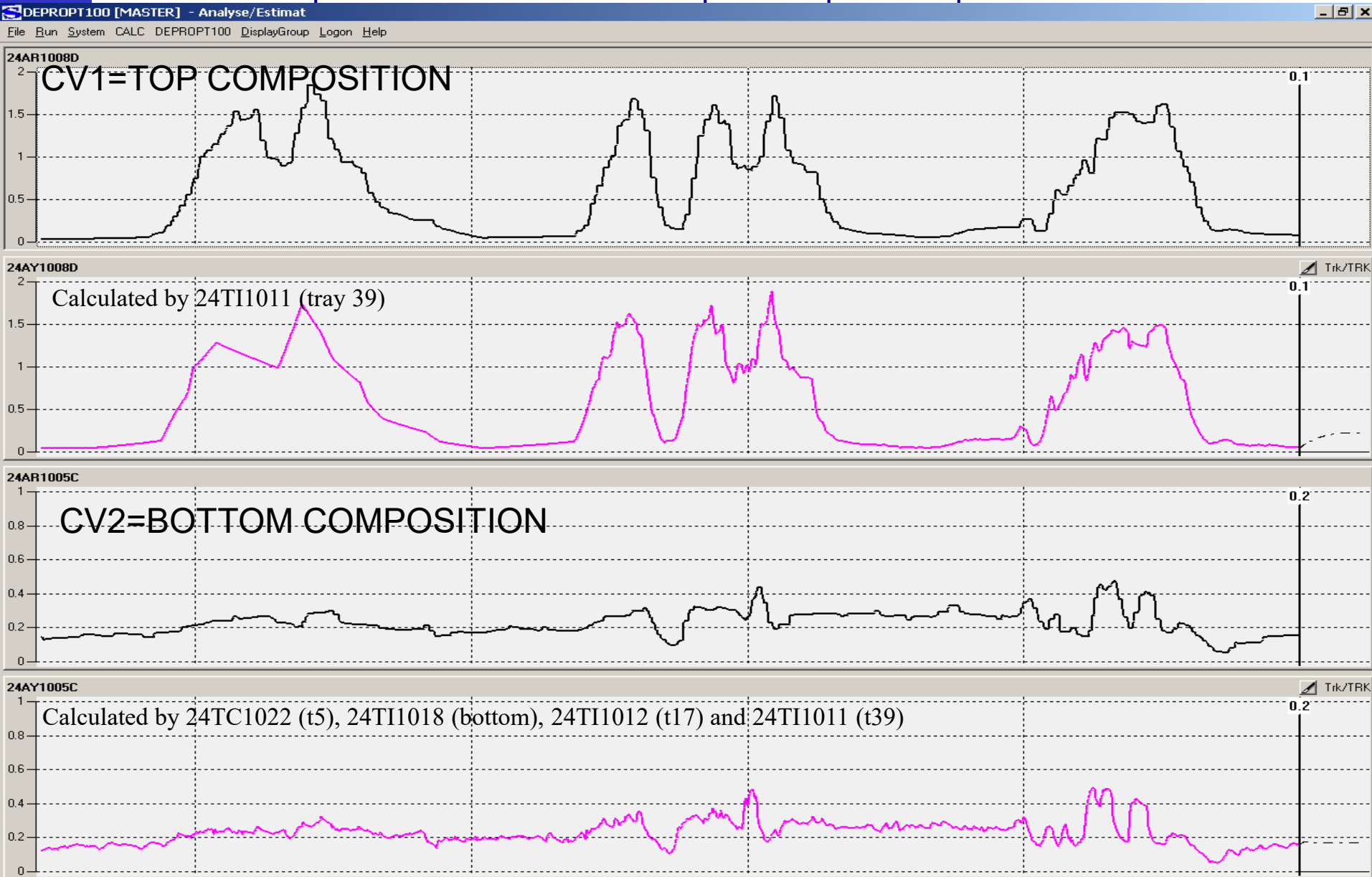
Depropaniser Train100 step testing

3 days – normal operation during night



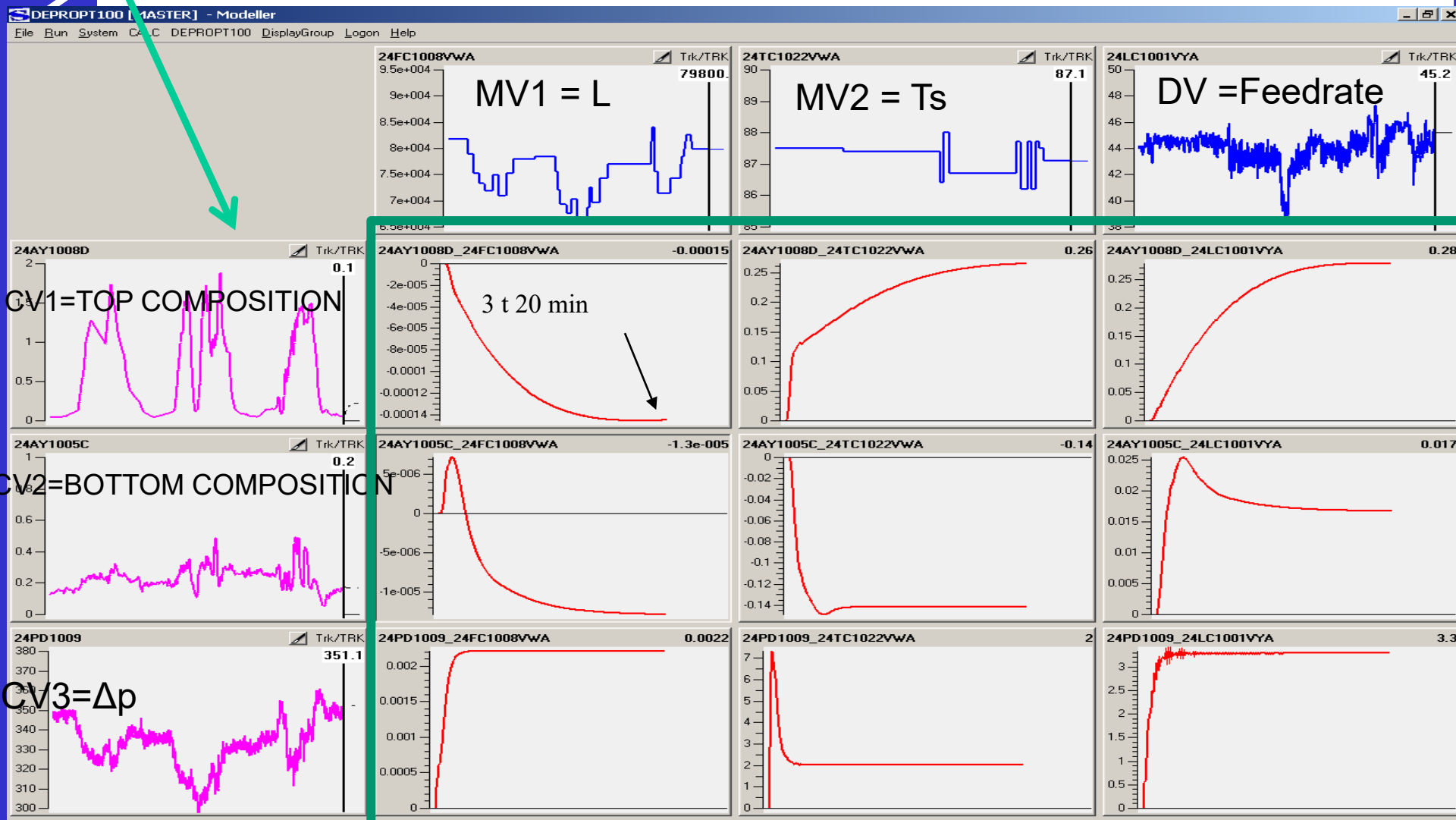
Estimator: inferential models

- Analyser responses are delayed – temperature measurements respond 20 min earlier
- Combine temperature measurements → predicts product qualities well



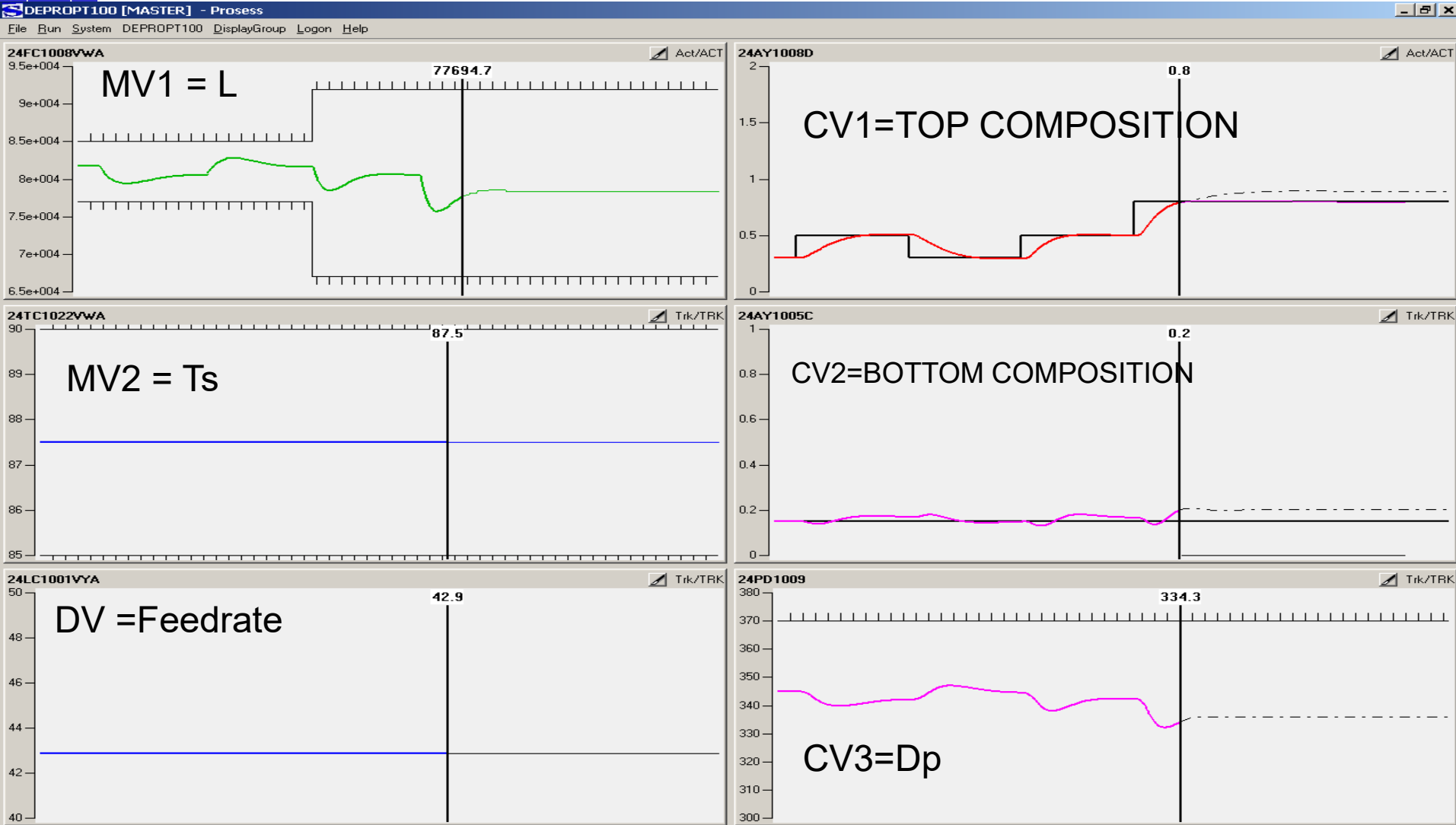
Depropaniser Train100 step testing – Final model

- Step response models:
 - MV1=reflux set point increase of 1 kg/h
 - MV2=temperature set point increase of 1 degree C
 - DV=output increase of 1%.

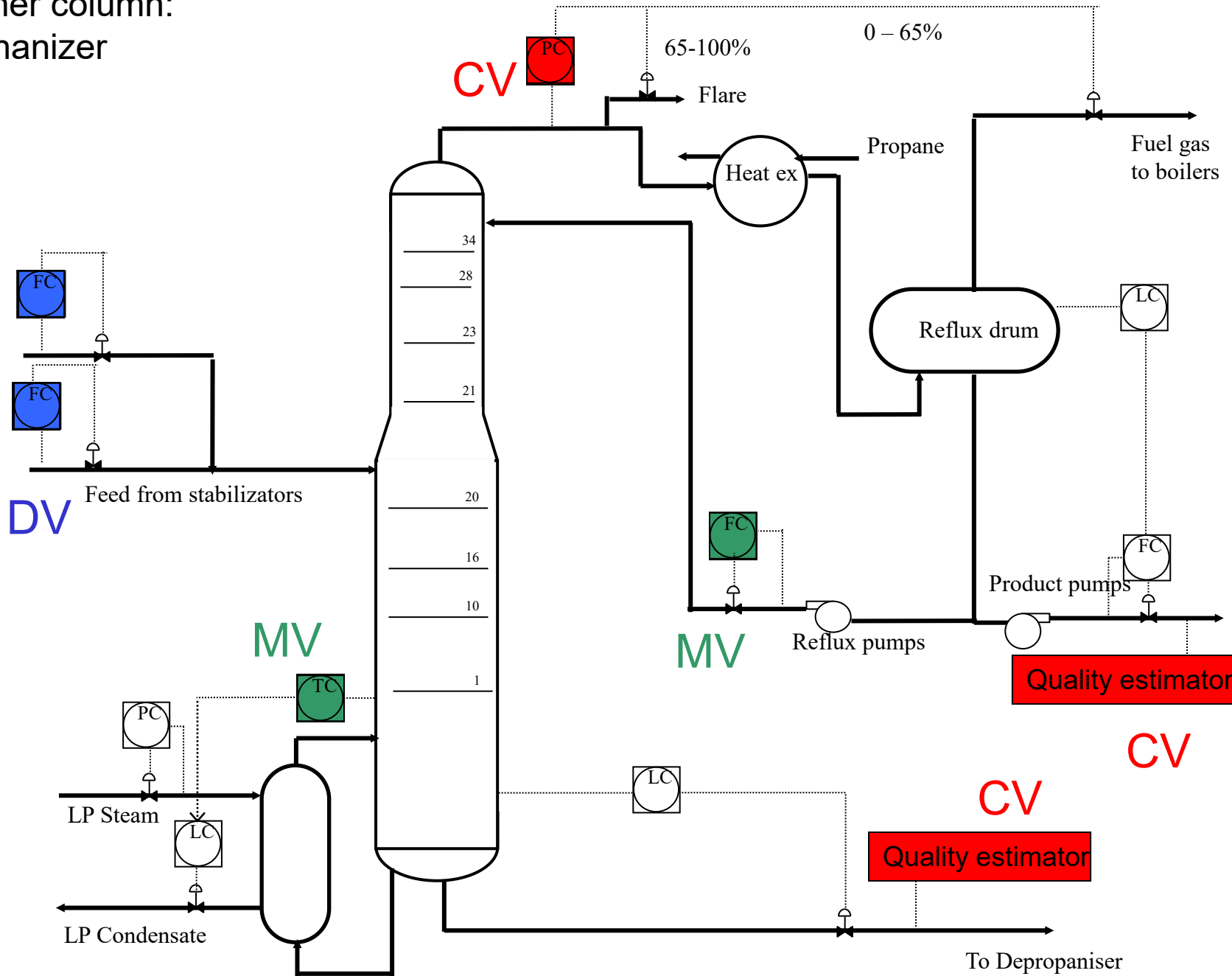


Depropaniser Train100 MPC – controller activation

- Starts with 1 MV and 1 CV – CV set point changes, controller tuning, model verification and corrections
- Shifts to another MV/CV pair, same procedure
- Interactions verified – controls 2x2 system (2 MV + 2 CV)



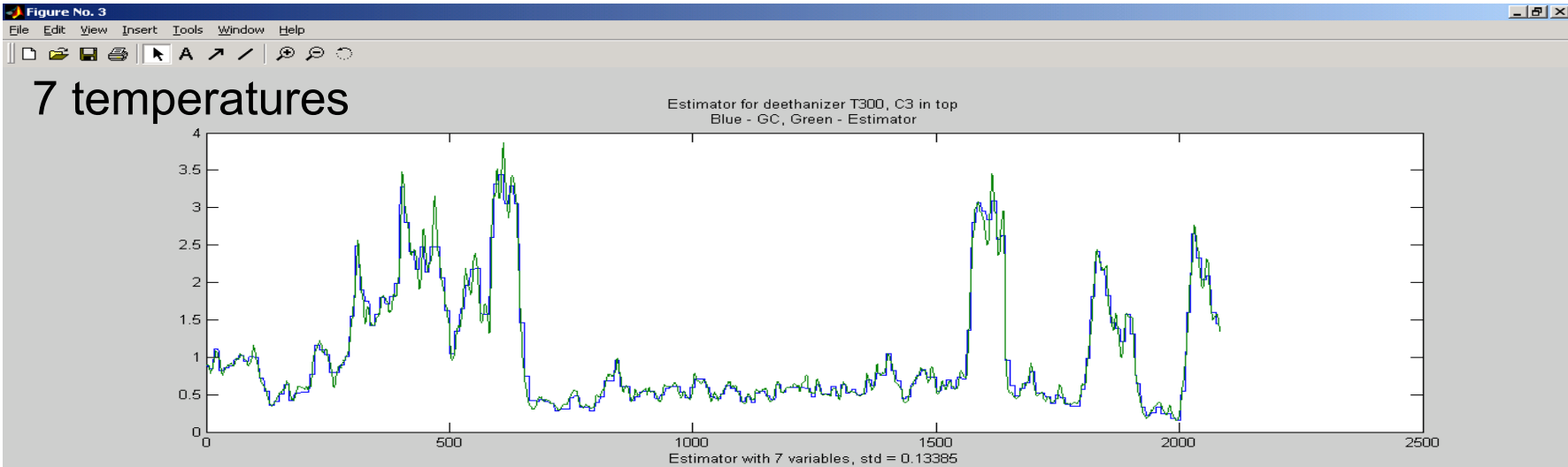
Another column: Deethanizer



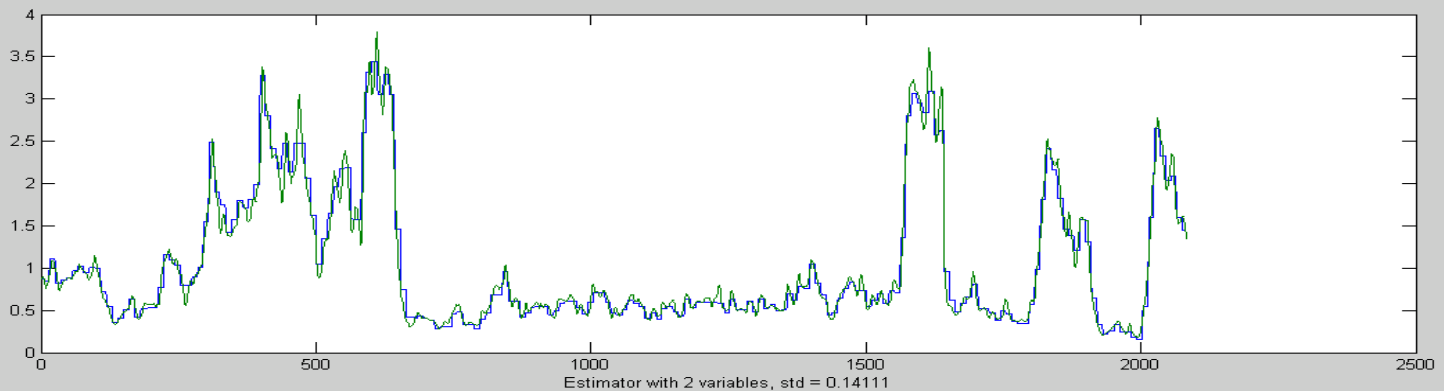
Top: Binary separation in this case

Quality estimator vs. gas chromatograph

(use logarithmic composition to reduce nonlinearity, $CV = -\ln x_{\text{impurity}}$)

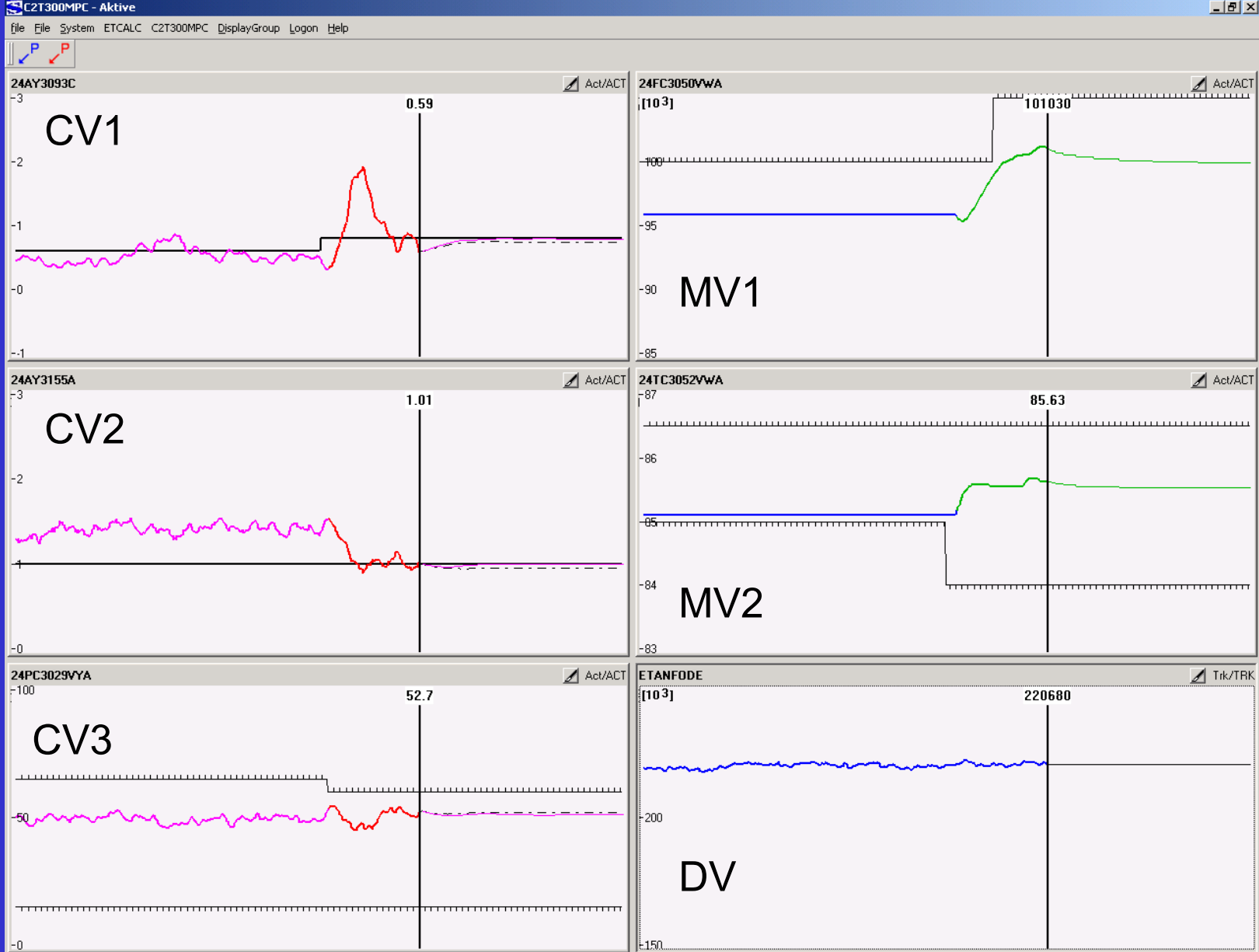


2 temperatures



=little difference if the right temperatures are chosen

The final test: MPC in closed-loop



Conclusion MPC

- Sometimes simpler than previous advanced control
- Well accepted by operators
- Equinor: Use of in-house technology and expertise successful

V. Pole placement (state feedback)

- Place closed-loop poles. Old design method
- Useful for insight, but difficult to use. Not used much in practice, at least not for linear controllers
- Basis:
 - Linear system on state space form

$$\frac{dx}{dt} = A x + B u$$

- and use “State feedback” (assuming we can measure all the states)

$$u = K x$$

Note

1. SIMC is “pole placement” ($p = -1/\tau_{auc}$), but with output feedback (y), and we also place zeros
2. If we cannot measure all the states, then we can estimate x from y using a “Kalman filter”.
3. State feedback uses extra measurements – an alternative is cascade control

8.5.1 Stability and state feedback

The poles of the transfer function, which are the zeros of its denominator polynomial, determine the dynamic characteristics of the system, in particular its stability and its damping characteristics. Transferring this statement to equation (8.60), it follows that the roots of the equation

$$\det(s \cdot I - A) = 0 \quad (8.67)$$

are essential for the behaviour of the system. The determinant in equation (8.67) is a n -th order polynomial in s and corresponds to the characteristic polynomial. The roots of the determinant in equation (8.67) are also designated as the eigenvalues of the matrix A . All of them must exhibit negative real parts, if the system described by the matrix A is supposed to be stable.

which will be combined to yield

$$\dot{\boldsymbol{x}} = (A - B \cdot K) \cdot \boldsymbol{x} \quad (8.71)$$

Equation (8.71) describes a system without any input variables with the system matrix

$$A_K = A - B \cdot K \quad . \quad (8.72)$$

8.5.2 Pole placement

One possibility for the controller design is to select desirable eigenvalues of the matrix A_K and to determine from this and the known matrices A and B the controller or feedback matrix K .

As an example a state feedback is to be determined according to the mentioned procedure of pole placement for a transfer system with a single input and a single output variable. Figure 8-6 shows the functional diagram of the system with feedback.

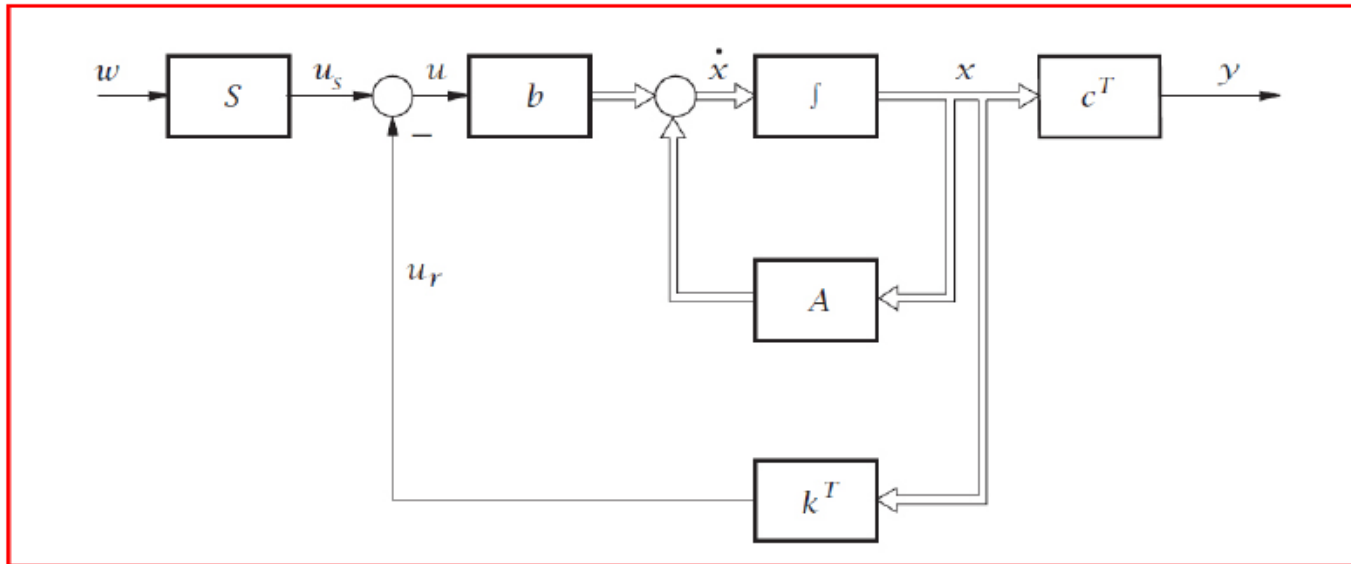


Figure 8-6: SISO-system with state feedback

The transfer system may be stated in controller canonical form according to equation (8.23). The state variables of the controller canonical form can be obtained for this purpose by transformation of the original state variables in the way described in chapter 8.2. According