3D SHAPE MEASURING INSTRUMENTS USING HIGH STIFFNESS VIBRATION TOUCH SENSOR

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Abstract: In this paper, a novel method to determine the shape of a 3D object is proposed. Instead of a flexible conventional touch probe, a high-stiffness vibration touch sensor was developed. The proposed sensor consisted of a piezo-electric device and thin probe, and generated feedback voltage according to the relative distance from the object. The structure of this sensor is simple, so that it can withstand high rates of acceleration during the measurement process. A nonlinear synchronous multi-axis control algorithm is also proposed. The effectiveness of our proposed sensor and control algorithm is demonstrated through experiments and simulations. Copyright ©2005 IFAC

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1. INTRODUCTION

Computer Aided Testing (CAT) is a very important technology for the manufacturing industry, because it acts as the final stage of the manufacturing process beginning with design, followed by production, and concluding with quality control by testing. However, conventional coordinate measuring instruments are prohibitively expensive; moreover, such instruments take a long time to examine a single object. Therefore, complete testing has been impossible for cheap, mass-produced goods such as those manufactured by casting, molding and press. Thus, the ultimate purpose of our research is to develop fast and inexpensive instruments for 3D shape and/or coordinate measurements.

Generally, 3D shape-measuring instruments can be classified into two types: a touch type, such as the touch trigger probe, and a non-touch type, such as that consisting of a laser sensor and a CCD device. The former is extremely precise, but its measurements are time consuming because the probe’s structure is complicated; furthermore, its flexibility prohibits movement of the probe at high rates of acceleration. Meanwhile, the latter can measure an object very quickly, but the color and/or the shape of the object is limited due to diffused reflection and/or the angle of reflection.

In order to solve these problems, we herein propose a novel vibration touch sensor consisting of a piezo-electric device, and plan to develop a 3D shape-measuring instruments to scan the surface of 3D objects(K. Enokishima and Terashima, 2004). Our proposed sensor has a simple structure and a high degree of stiffness so that it resists motion at high rates of acceleration, thereby making very fast measurement possible. Furthermore, it is not affected by the color and/or the gloss of an object since it is a touch-type sensor.

To date, many researchers have reported on piezo-electric devices in the field of control and measurement engineering. However, most of these
studies have focused either on actuators such as those employed for precise positioning and active damping (Sacconi A. and W., 1999; Shuo Hung Chang and Chien, 1999; Kakatcioglu S. and H., 1997) or on sensors such as those that measure pressure and force (T. and J., 1997; Shimizu T. and K., 2002). However, our proposed sensor has characteristics of both an actuator that generates vibration and a sensor that can detect touch; i.e., it is a bilateral vibration touch sensor.

In their research regarding the bilaterality of a piezo-electric device, Baqlio et al. have developed a system for recognizing the nature of an object’s surface (Baqlio S. and N., 2002). They use two piezo-electric devices: one stimulates the surface of the material, and the other detects the response from the object to this stimulation. Their system detects the nature of a material, e.g., whether glass, plastic, or wood. However, we use only a single piezo-electric device whose purpose is to detect the existence of an object.

In this paper, the control method for our measuring instruments is also discussed. The vibrating sensor is moved toward the object while it vibrates, and it identifies the position of the object by detecting the cessation of vibration caused by the sensor touching the object’s surface. Based on this principle, we have developed a commercial product, the PIX-30 (Roland. D.G. Co., 2002). However, it took too much time to measure an object because the sensor checked each insignificant grid partitioned within the wide target space.

In order to solve this problem, a new control law is proposed that is expected to produce faster measurements. With it, the sensor moves so as to maintain a steady vibration, scanning the surface of the object continuously. Furthermore, a synchronous multi-axis control algorithm is proposed in order to scan the surface of an object effectively and precisely in cooperation with each other. The effectiveness of our proposed method is demonstrated through experiments and simulations.

2. STRUCTURE AND SPECIFICATIONS OF THE SENSOR

2.1 Structure of the sensor

The structure of the proposed sensor is shown in Fig.1. A thin probe is attached only to the piezo-electric device. Therefore, the sensor is simple, rigid, and extremely robust. The piezo-electric element consists of two parts. One has a drive electrode and works as the vibrator by applying an alternating current, while the other has a feedback electrode and detects the magnitude of the vibration as an alternating voltage. This sensor plays the roles of both vibrator and sensor.

We defined the X-, Y-, and Z-axis coordinates as shown in Fig.1. The piezo-electric device is a 7BB35-3CA0 made by Murata Corp.

![Fig. 1. Structure of the vibration touch sensor](image)

2.2 Frequency characteristic of the sensor and standing wave on the probe

The sensor has a natural frequency. Therefore, we had to apply a resonant drive current in order to oscillate the piezo-electric device most efficiently and in order to acquire a large feedback voltage. The frequency characteristics of the feedback voltage when a 0.147[Vp-p] sinusoidal voltage was applied to the drive electrode are shown in Fig.2.

Without the probe, this piezo-electric device has a 2.58[kHz] natural frequency without a probe (Murata Manufacturing Co., 2003). Note that the resonant frequency moved due to the attachment of the probe. A sinusoidal wave of 2.66[kHz] was selected as the applied voltage. The small peak at 170[Hz] in this figure demonstrates the probe’s first order vibration, in which the resonant frequency is consistent with the result of vibration analysis by ANSYS as shown in Fig.3(a).

![Fig. 2. Frequency characteristics of the feedback voltage](image)

Whereas, the vibration of the multiple-order mode of 2.66[kHz] in Fig.2 is shown in Fig.3(b).

A standing wave with three antinodes and two nodes was observed on the probe according to the position. The phase at the top of the figures and that at the tip of the probe ware consistent with that of the source signal. However, the phase inverse was observed in the third figure. In addition,
2.3 Steady-state characteristic of the sensor

The bilateral sensor outputs the induced voltage according to the vibration of the probe. The magnitude of the vibration changes according to the relative distance from the object. When the probe is pressed to the object strongly, the vibration stops and the feedback voltage becomes minimal as shown in Fig.5(a). Meanwhile, when the probe vibrates apart from the object, the feedback voltage reaches its maximum as shown in Fig.5(b). When the probe has light contact with the object, the feedback voltage begins to decrease slightly as shown in Fig.5(c). Note that the Fig.5 is emphasized. Fig.6 shows the steady state relationship between the relative distance from the sensor to the object with respect to each axis and the feedback voltage when 0.147[Vp-p] sinusoidal voltage is applied to the drive electrode. The origin on the horizontal axis is the first contact position with the object.
distance $[\mu m]$ from the object to the center of the sensor respectively.

### 2.4 Robustness for high rates of acceleration

In order to ensure that the proposed sensor would not encounter trouble during high rates of acceleration motion, we conducted an experiment at approximately $1G(9.8m/s^2)$ acceleration.

The experimental condition were as bellows: Beforehand, we established the origin where the feedback voltage is the reference voltage in Fig.6. We measured the feedback voltage when the sensor moved from the position of $0.1m$ distance from the origin at the maximum speed of $1m/s$ and $1G$ approximately and stopped at the origin. The experimental result is shown in Fig.7. The low level of the positioning signal in the third figure indicates that the sensor exists in the area within $2[\mu m]$ from the origin. This experimental result shows that the proposed sensor worked correctly in spite of high rates of acceleration and detected the position of the object precisely. Thus, our proposed sensor appears to make high-throughput 3D shape-measuring instruments possible.

![Fig. 8. Experimental apparatus of measuring instruments](image)

### 3. NONLINEAR SYNCHRONOUS CONTROL FOR MEASURING INSTRUMENTS

#### 3.1 Experimental apparatus of the measuring instruments

An X-Y-Z table made by the Roland D.G. Corp. and shown in Fig.8 was used as the platform of the measuring instruments. This platform moves our proposed sensor attached at the end-effector toward the X-, Y-, and Z-axes independently. The position of the sensor is controlled at each axis by a stepping motor, and is measured by a laser displacement meter.

![Fig. 9. Block diagram of independent control](image)

#### 3.2 Independent control with single axis

As preparation for scanning a 3D shape, each axis was independently controlled as shown in Fig.9.

The reference voltage is those shown in Fig.6. The position of the sensor was controlled by the motion of the X-Y-Z table, regulating the feedback voltage into the reference voltage so that the sensor was stabilized on the surface of the object. We assumed that the motor moved immediately in accordance with the reference velocity, so that the motor was denoted as a mere integration $1/s$. The block of the "Sensor" in Fig.9 indicates the function in Eq.(1). The controller was a PI compensator written in Eq.(2).

$$u = K_p e + K_i \int e dt, \quad (2)$$

where, $K_p$ is the proportional gain, and $K_i$ is the integral gain. $e$ is the error between the reference voltage and the feedback voltage, and the control input $u$ is the reference velocity. The simulation and the experimental result show the transient response when the sensor moves close to the object and touches the object.

Figure 10 shows the simulation and experimental results in the Y axis, and Fig.11 shows them in...
3.3 Nonlinear synchronous control on two axes

A block diagram of the synchronous control of the Y- and Z-axes is shown in Fig. 12. The control strategy was as follows: The Z-axis was controlled as a master axis by a PI compensator because the Z axis is so sensitive. On the other hand, the Y axis was controlled as a slave axis by the cosine of the common integration and another proportional gain. The control law was written in the following equation:

\[
\begin{align*}
    u_z &= K_{pz}e + K_{iz} \int e \, dt \\
    u_y &= V_{max} \cos(K_{py}e + \int e \, dt),
\end{align*}
\]

where \( u_z \) and \( u_y \) are the reference velocities, \( V_{max} \) is the maximum velocity of the Y-axis, \( e \) is the error between the reference voltage and the feedback voltage, \( K_{pz} \) and \( K_{py} \) are the proportional gain of each axis, and \( K_{iz} \) is the integral gain of the Z-axis.

If the feedback voltage is equal to the reference, then the output of the proportional gain is zero and the integral value is maintained as a steady value. Therefore, the Y- and Z-axes continue to move at those velocities. Thus, if the object has a linear inclination, then the feedback voltage is consistent with the reference and integral values is fixed in the steady-state, and the velocities of both the Y- and Z-axes maintain a straight-line motion on the object’s surface.

On the other hand, in the case that the object has a curved surface, the velocity on the Y-axis may become slow according to the increase of the integration. Note that the integral value becomes saturated up to \( \pm \pi/2 \). This means that motion of the Y-axis stops and the only Z-axis moves up or down.

The most difficult condition for surface scanning is a perpendicular shape (It goes without saying that an overhang is beyond the scope of this measurement device). The simulation result for a rectangle 1[mm] \( \times \) 1[mm] is shown in Fig. 13. The term of the inside of the cosine function is stabilized at \( \pm \pi/2 \) at the perpendicular wall. Whereas, the same term is stabilized at 0 at the top of the rectangle. Then, both the error and its integration become zero, and \( u_z \) and \( u_y \) are equal to zero and \( V_{max} \), respectively. This means that the sensor moves toward the horizontal direction at maximum speed on the flat top of the rectangle.

In Fig. 13, the solid line shows the trajectory of the sensor, and the dotted line shows the contour of the object. The solid line is consistent with the dotted line except for inside the circle. The difference of the trajectory in the circle is caused by the overshoot due to the delay of the control.
4. CONCLUDING REMARKS

In this paper, a novel vibration touch sensor and its measurement algorithm were proposed. The proposed sensor consisted of a piezo-electric device and a thin probe, and had high stiffness and robustness against high rates of acceleration, due to the simplicity of the structure. It was determined that the relationship between the feedback voltage and the relative distance from an object is approximately linear. By controlling the sensor position on a single axis, it was demonstrated that the model was accurate and that the proposed system could be stabilized on the surface of the object. Finally, a nonlinear synchronous control was proposed in order to scan the 3D shape of objects, and its effectiveness was demonstrated through experiments and simulations. The proposed system obtained the contour of a rectangle to 3.0[µm] degree of precision in the X-Y plane.

REFERENCES


