

Problem 1

(a) We assume constant density.
Overall mass balances and component mass balances (acid)
for the two tanks then given

Tank 1: (1) $\frac{dV_1}{dt} = q_A + q_{w1} - q_1$ [mol/s]

(2) $\frac{d(C_1 V_1)}{dt} = q_A C_A + q_{w1} C_{w1} - q_1 C_1$ [mol acid/s]

Tank 2: (3) $\frac{dV_2}{dt} = q_1 + q_{w2} - q_2$ [mol/s]

(4) $\frac{d(C_2 V_2)}{dt} = q_1 C_1 + q_{w2} C_{w2} - q_2 C_2$ [mol acid/s]

(b) Transfer matrix G:

$$\begin{matrix} V_1 \\ C_1 \end{matrix} \begin{pmatrix} q_A & q_{w1} & q_1 \\ \frac{1}{s} & \frac{1}{s} & -\frac{1}{s} \\ \frac{k_A}{\tau s + 1} & \frac{k_{w1}}{\tau s + 1} & 0 \end{pmatrix}$$

$\tau = \frac{V_1}{q_A + q_{w1}} \approx \frac{V_1}{q_1}$ = residence time (could be derived by uncertainty (2) but not required.)

$k_A > 0$
 $k_{w1} < 0$

(c) (1) Feedback

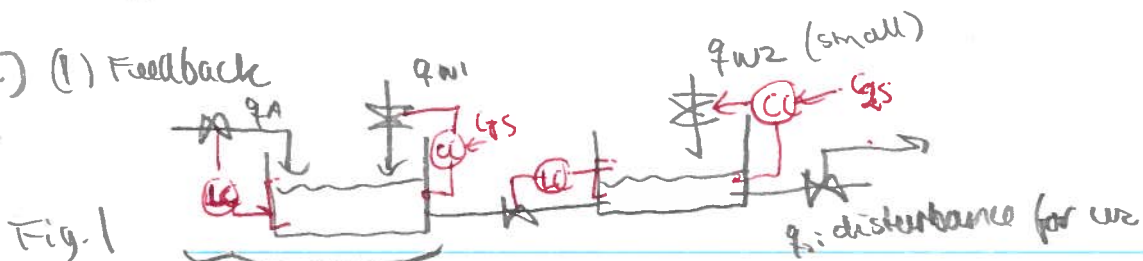


Fig. 1

Could interchange LC and CC, the largest flow should be used for level control to reduce interactions, so here we have assumed $q_A > q_{w1}$, but it could be opposite. Actually, this is probably more likely since it is given that q_{w1} is large!

2) No measurement of G_1

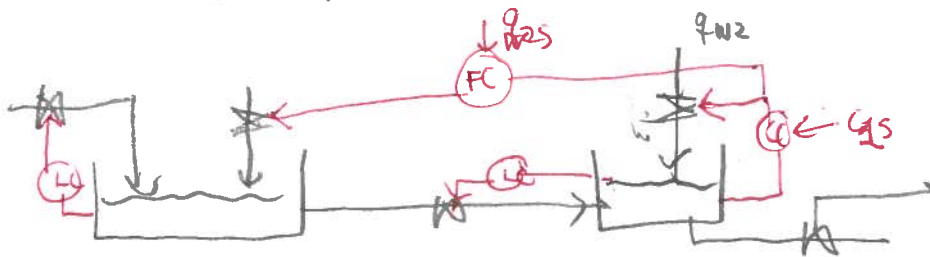


Fig. 2

- Also here the (L1) and (FC) could be interchanged (tank 1).
- The FC may alternatively be called VPC (value position controller) as it resets the position of the valve for q_{w2} to some middle position (say 50% open)
- Comment on structure 1 (feedback): The setpoint for G_1 needs to be decided. It could be set by a FC (or VPC) similar to Fig. 2.

Problem 2.

Overall process:

$$(a) \quad G(s) = \frac{3 e^{-3s}}{(100s+1)(105s+1)} \approx \begin{cases} \frac{3}{105s+1} e^{-1s} & \text{(for PI)} \\ \text{unchanged} & \text{(for PID)} \end{cases}$$

Here I would go for the PID, since in the original model $\tau_2 = 10 > \theta = 3$. But let us design both.

PI-controller:

$$K_c = \frac{1}{K} \frac{\tau}{\tau_c + \theta} = \frac{1}{3} \frac{105}{18+18} = 0.972$$

$$\tau_c = \min(\tau_1, 4(\tau_c + \theta)) = \min(105, \frac{4 \cdot 2 \cdot 18}{1}) = 105$$

Cascade-PID:

$$K_c = \frac{1}{3} \frac{100}{3+3} = 5.6$$

$$\tau_c = \min(\tau_1, 4(\tau_c + \theta)) = \min(100, 4 \cdot 2 \cdot 3) = 24$$

$$\tau_0 = \tau_2 = 10$$

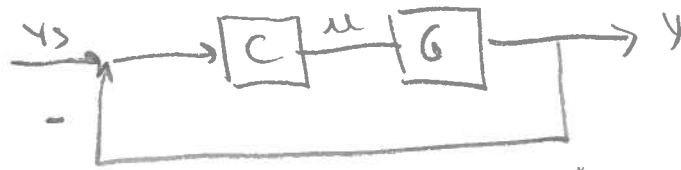
$$(b) \quad G(s) = \frac{3 e^{-2.25s}}{(100s+1)(105s+1)} \approx \frac{3}{105s+1} e^{-2.25s}$$

$$\text{PI: } (\tau_c = \theta) \quad K_c = \frac{1}{K} \frac{\tau}{\tau_c + \theta} = \frac{1}{3} \frac{105}{2 \cdot 2.25} = 0.65$$

$$\tau_c = \tau = 105$$

Here PI is recommended since $\tau_2 = 10 < \theta = 2.25$.

Problem 3.



$$(a) \quad y = T_1 y_s \quad T_1 = \frac{\text{"direct"}}{1 + \text{loop}} = \frac{GC}{1 + GC}$$

$$u = T_2 y_s \quad T_2 = \frac{C}{1 + GC}$$

$$(b) \quad G = \frac{k}{\tau_s s + 1} \quad (\theta = 0)$$

SMC-rule with $\tau_c = \tau_1$

$$k_c = \frac{1}{k} \frac{\tau_1}{\tau_c + \theta} = \frac{1}{k} \frac{\tau_1}{\tau_1} = \frac{1}{k}$$

$$\tau_c = \min(\tau_1, 4(\tau_c + \theta)) = \min(\tau_1, 4\tau_1) = \tau_1$$

$$\text{Get } GC = \frac{k}{\tau_s s + 1} \cdot k_c \frac{\tau_c s + 1}{\tau_c s} = \frac{k}{\tau_s s + 1} \cdot \frac{1}{k} \frac{\tau_1 s + 1}{\tau_1 s} = \frac{1}{\tau_s s} \quad (\text{very simple!!})$$

$$T_1 = \frac{GC}{1 + GC} = \frac{1/\tau_s s}{1 + 1/\tau_s s} = \frac{1}{\tau_s s + 1} \quad (\text{as expected when we use SMC!!})$$

$$T_2 = \frac{C}{1 + GC} = \frac{\frac{1}{k} \frac{(\tau_s s + 1)}{\tau_s}}{1 + 1/\tau_s s} = \frac{\frac{1}{k} (\tau_s s + 1)}{\tau_s s + 1} = \frac{1}{k}$$

(very simple again!!)

Comment:

This makes sense, we with this controller make the closed-loop response have the same dynamics as the open-loop response. So just stepping u gives the desired $T = \frac{1}{\tau_s s + 1}$.

Problem 4.

We want to avoid pairing on negative RGA.

(a) There are two possible pairings on positive values

$$RGA(0) = \begin{bmatrix} \textcircled{0.45} & 0.22 & \textcircled{0.33} \\ -0.82 & \textcircled{2.45} & -0.64 \\ \textcircled{1.36} & -1.67 & \textcircled{1.31} \end{bmatrix} \begin{matrix} y_1 \\ y_2 \\ y_3 \end{matrix} \quad \begin{matrix} \textcircled{Alt-1} \\ \textcircled{Alt-2} \end{matrix}$$

Probably, alternative 2 is the best because of the delay on the 11-element in $G(s)$. Conclusion (Alt-2):

$$\begin{aligned} u_1 &\leftrightarrow y_3 \\ u_2 &\leftrightarrow y_2 \\ u_3 &\leftrightarrow y_1 \end{aligned}$$

(b) The RGA is defined as $\lambda_{ij} = \frac{g_{ij}}{\sum_k g_{kj}}$

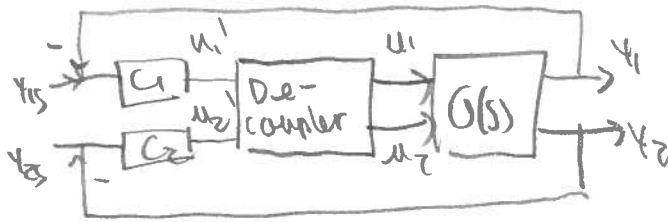
where g_{ij} is the open-loop gain from u_j to y_i (with the other inputs u_k constant) and \hat{g}_{ij} is the closed-loop gain with the other outputs controlled. We here consider steady-state RGA. Ideal case is $\lambda_{ij} = 1$ for pairing.

1. $\lambda_{ij} < 0$: Gain reversal, that is, gain from u_j to y_i will change sign when the other loops are closed. This may obviously lead to instability, at least with integral action in the controller.

2. $0 < \lambda_{ij} < 1$: We then have $|\hat{g}_{ij}| > |g_{ij}|$ which means that the interactions increase the gain. This may give stability problems, especially with λ_{ij} close to 0.

3. $\lambda_{ij} > 1$: The interactions reduce the gain, that is, the loops "fight each other". This may yield poor performance (slow response) when λ_{ij} is large.

c) Decoupling is used to get acceptable performance when using single-loop controllers.



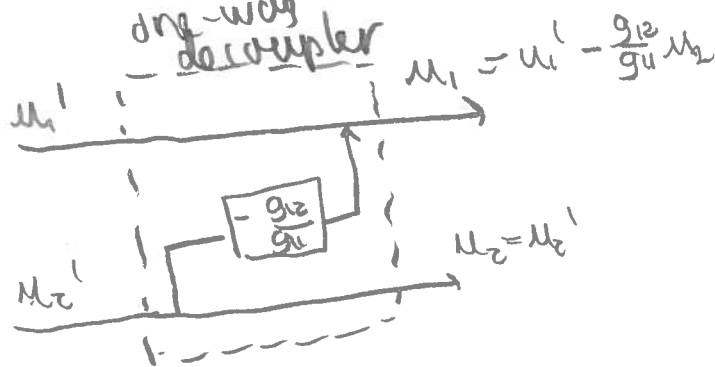
A one-way decoupler may be designed by considering u_2 as a disturbance on y_1 .

$$y_1 = g_{11} u_1 + g_{12} u_2$$

Design a "feedforward" controller based on measuring u_2 . Ideal case is to have $y_1 = 0$ which gives

$$u_1 = -\frac{g_{12}}{g_{11}} u_2$$

so the one-way decoupler is



To be realizable we need $-\frac{g_{12}}{g_{11}}$ to be realizable. This means that the delay in g_{11} should be smaller than in g_{12} and g_{11} should be lower order ($\# \text{ poles} - \# \text{ zeros}$) than g_{12} , that is, g_{11} should have a more direct effect, that is, "fair close"!

Problem 5.

$$(a) \quad G(s) = \frac{1-5s}{8s+1} \quad C(s) = k_c \left(\frac{1-s}{8s} \right) \quad \text{with } \tau_D = 8$$

loop transfer function

$$L(s) = G(s)C(s) = \frac{(1-5s)}{(8s+1)} \cdot \frac{k_c(1-s)}{8s} = k_c \frac{(1-5s)}{8s}$$

Closed-loop poles

$$1 + L(s) = 0 \Rightarrow 1 + \frac{k_c(1-5s)}{8s} = 0 \Rightarrow 8s + k_c - 5k_c s = 0$$

$$\Rightarrow (8 - 5k_c)s + k_c = 0$$

Want both coefficients to have same sign to be stable. This gives

$$k_c > 0$$

$$8 - 5k_c > 0 \Leftrightarrow k_c < \frac{8}{5} = \underline{\underline{1.6}} \quad \leftarrow k_{cmax}$$

(b) SIMC. We first have to approximate as a first-order plus delay process

$$G(s) = \frac{1-5s}{8s+1} \approx \frac{e^{-5s}}{8s+1}$$

SIMC-rules with $\tau_c = \theta = 5$

$$k_c = \frac{1}{K} \frac{\tau}{\tau + \theta} = \frac{1}{1} \cdot \frac{8}{5+5} = 0.8$$

$$\tau_D = \min(\tau, 4(\tau + \theta)) = \min(8, 40) = 8$$

This is the same as the controller from part a) but with $k_c = 0.8$. Since $k_{cmax} = 1.6$, we have that $GM = 2$.

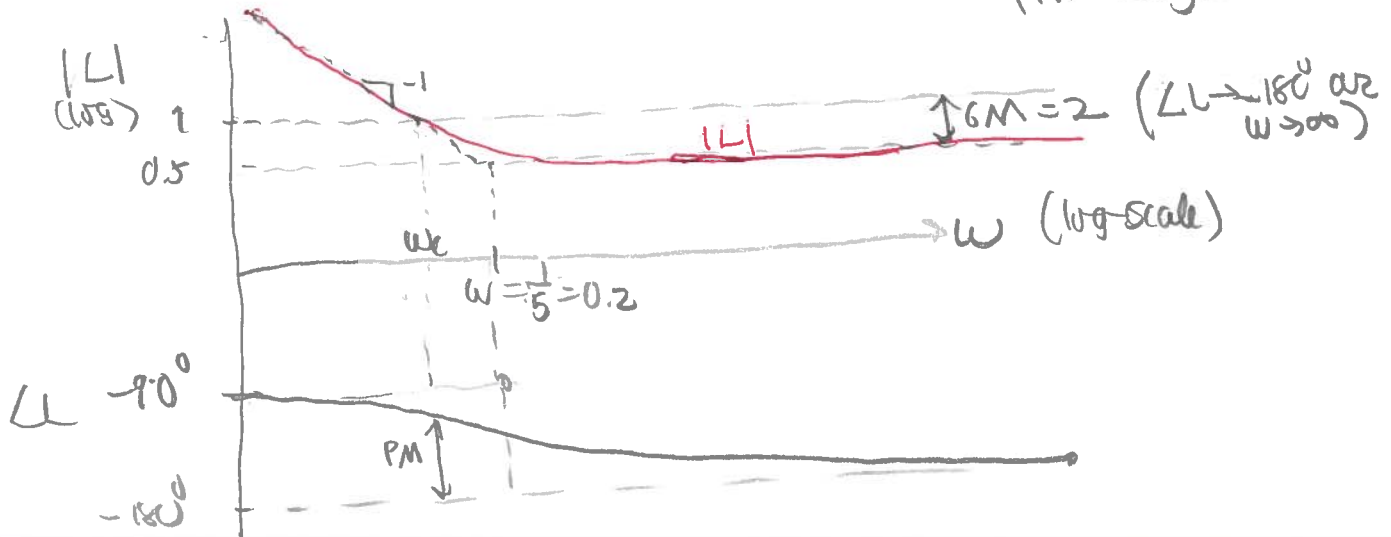
This can also be found from the Bode-plot:

$$L(s) = \frac{1-5s}{8s+1} \cdot 0.8 \frac{8s+1}{8s} = 0.1 \frac{1-5s}{s} = \begin{cases} -\frac{1}{5} & \omega \rightarrow 0 \\ \rightarrow 0.5 & \text{as } s \rightarrow \infty \end{cases}$$

Exact values: $|L(j\omega)| = \frac{0.1 \sqrt{1+(5\omega)^2}}{\omega}$

$$\angle L(j\omega) = \text{arctan}(-5\omega) - \frac{\pi}{2}$$

From integrator



From the ^{asymptotic} plot we see that $\omega_c \approx \frac{0.5}{2} = 0.25$ rad/s, but we make a small table to be more accurate

ω	$ L(j\omega) $
0.1	$0.1 \sqrt{1+(5 \cdot 0.1)^2} / 0.1 = 1.1$
0.11	1.038
0.115	1.003

Conclusion: $\omega_c = 0.115$ rad/s

This gives $\angle L(j\omega_c) = -\text{arctan}(5 \cdot 0.115) - \frac{\pi}{2} =$
 $= -0.522 - 1.571 = -2.091$ rad

So $PM = \pi + \angle L(j\omega_c) = 3.14 - 2.091 = 1.049$ rad
 $(= 60.1^\circ)$

Delay margin = $\Delta\theta = \frac{PM}{\omega_c} = \frac{1.049 \text{ rad}}{0.115 \text{ rad/s}} = \underline{\underline{9.12 \text{ s}}}$

Problem 6.



(a) $F = 30\%$ (it is meant $F = +30\%$) is what is given in the figure, so the red curve corresponds to $d = 1$ (step). Since no steady state is needed for $y (=T)$, the red curve directly gives

G_d :

$$G_d = \frac{3}{\tau_d s + 1}$$

τ_d is where y is 63%, that is when $y = T = 3 \cdot 0.63 = 1.89$ and we find $\tau_d \approx 3$

$Q = 50\%$ (it is meant $Q = +50\%$) corresponds to $u = 1$ so the response in the figure is with $u = 0.2$ ($Q = +10\%$). So the steady-state gain from u to y is 5 times larger than for the blue curve. Otherwise, the dynamics are the same and we derive

$$G = \frac{5 e^{-s}}{3 s + 1}$$

↑ since it reaches 0.63 at $t \approx 4 = \tau + \theta$

(b) - We have $|G| > |G_d|$ at all frequencies so there is no problem with input constraints.

- But what about disturbances? Can we react fast enough? We have a delay $\theta = 1$, so we need $\omega_c < \frac{1}{\theta} = 1$

But for acceptable disturbance rejection we need

$\omega_c > \omega_d \approx \frac{kd}{\tau_d} = \frac{3}{3} = 1$. Hmm... this is a borderline

case, so we may not be OK. From the oversized response, I think we will not be OK with PI since $\gamma > 1$.

